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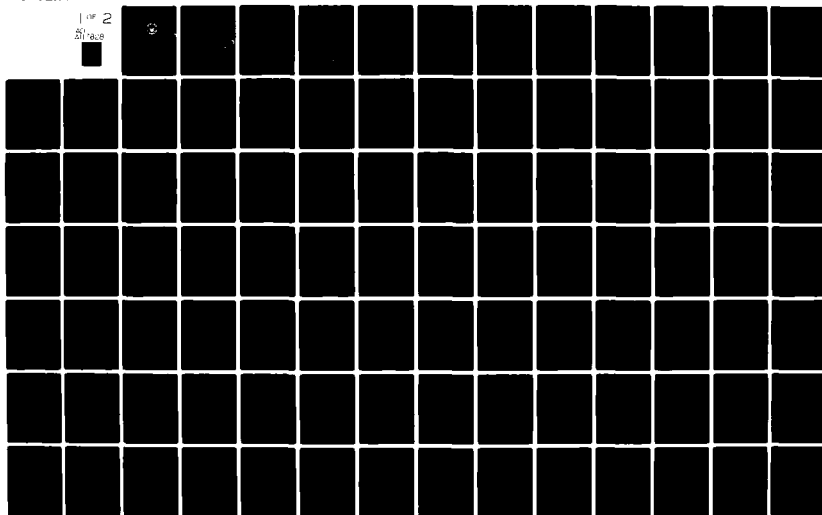
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THESIS

AN INTERACTIVE COMPUTER PROGRAM
FOR THE PRELIMINARY DESIGN AND
ANALYSIS OF MARINE REDUCTION GEARS

by

Joseph Louis Paquette

March, 1982

Thesis Advisor:

G. Cantin

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An Interactive Computer Program
for the Preliminary Design and
Analysis of Marine Reduction Gears

by

Joseph Louis Paquette
Lieutenant, United States Navy
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Submitted in partial fulfillment of the
requirements for the degree of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING

from the

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ABSTRACT

The objective of this project was to develop an interactive computer program providing flexibility in the design and analysis of marine propulsion gears. The program, Reduction Gear Analysis and Design (REGAD), will handle conventional parallel axis and simple epicyclic reduction gears. It is capable of generating preliminary designs of new gear sets or providing analyses of existing or proposed gear sets. Program development, organization, and operation are discussed.

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I. INTRODUCTION

In the conceptual stage of ship design, many parameters and options are considered. This is especially true with respect to the propulsion plant. Changes in hull design, displacement, and numbers of propellers all affect changes in the requirements for the propulsion plant. There are also various options under consideration in the propulsion plant: turbines or internal combustion engines, the number of engines, the auxiliaries required for support, etc. All of these will affect the initial design of the reduction gears. It is, therefore, important to be able to produce preliminary designs of reduction gears for the options under consideration.

Preliminary designs provide useful information on feasible arrangements and size without going into the specific design details dependent upon manufacturing. Since any number of preliminary designs may be required due to perturbations discussed above, it is important to be able to automate the design process. An interactive computer program providing various options would free the engineer from tedious, time consuming, and often error prone number manipulation and allow him to produce multiple designs for consideration. It would also provide a quick means of checking

the effects of various parameters in addition to the ability to analyze proposed designs or configurations.

Reduction Gear Analysis and Design (RZGAD) was developed to fill this need. It is an interactive computer program offering close user control through numerous options. Being interactive, it provides a rapid means of designing or analyzing a gear set, thereby reducing the turn-around time inherent in the use of batch systems. The program was kept modularized and well documented for ease of maintenance and modification. The modularized construction also provides an additional benefit of being able to use this program on smaller computers by using an overlay scheme.

II. PROGRAM CAPABILITIES

A. SCOPE

REGAD was written to provide preliminary designs or analyses of marine propulsion reduction gears. It is incapable of providing detailed designs or performing detailed analyses since specifics of manufacturing are not required for input. The program does not consider shafting, bearings, lubrication, couplings, casings, or other auxiliaries. It will provide sizing information in the form of pitch diameters, effective facewidths, gear ratios, and numbers of teeth per gear. In addition, the program will provide estimates of loadings and stress levels. Estimated weight and dimensions of the gear set are also provided.

All computations are based primarily on the American Gear Manufacturers Association's standards [Ref. 1, 2, 3] using appropriate constants for marine propulsion gears [Ref. 4, 5]. As an option in the program, these constants can be replaced by the user to enable him to investigate other applications such as reduction gears for ships service or emergency generators.

B. LIMITATIONS AND OPTIONS

Program application is limited to marine reduction gears with a maximum of three reduction stages. Conventional parallel axis and simple epicyclic arrangements with helical gears are possible. When dealing with epicyclics, it is assumed that load sharing of the planets is achieved and that the ring gear is suitably flexible. Efficiencies of the gear sets are not provided since power losses are not computed. While estimates of bending and contact stresses are provided, scoring can not be estimated since lubrication is not considered. REGAD does not require the K-factor as input as in previous programs since hardness ranges for the pinions and gears are required. However, the K-factors are computed and displayed for reference purposes. The weight estimates are based on actual designs and do not include turning gears, attached lubrication oil pumps, or other auxiliaries.

The following is a list of major options provided by the program:

- (1) brief, on-line program description
- (2) choice of design or analysis
- (3) listing of preprogrammed constants and an ability to change selected constants of the user's choice
- (4) choice of single, double, or triple reductions
- (5) choice of single or double helical gears
- (6) choice of six hardness ranges for gears and pinions

- (7) conventional parallel axis arrangements (see Figures 1, 2, and 3)
 - (a) one or two power inputs
 - (b) single power path (articulated) or dual power paths (locked train)
- (8) simple epicyclic arrangements (see Figure 4)
 - (a) choice of planetary or star arrangements
 - (b) single power input
 - (c) choice of three, four, or five planet/star gears.

III. PROGRAM ORGANIZATION AND OPERATION

A. REGAD FLOWPATHS

As stated previously, the program was designed in modular form with each module consisting of a number of subprograms. These modules are just conceptual groupings of associated subprograms, and are not related to actual program implementation on any specific computer. Figure 5 shows the basic flow paths of the program. Module One is for program initialization and problem set up. Module Two performs calculations for conventional parallel axis gear sets, while Module Three handles epicyclic gear sets. Module Four is a grouping of all the computational subprograms required by the other modules.

B. MODULE IDENTIFICATION AND DESCRIPTION

This section provides a brief description of each subprogram in each module.

1. Module One : Initialization and Set-up

Module One contains the subprograms necessary for initialization, execution, and initial data entry. It is, basically, the control module for the program. The following is a grouping of the subprograms in Module One.

a. REGAD

REGAD is the main program. It provides the options for either design or analysis and either parallel axis or epicyclic arrangements and controls the flow to the proper module. It then calls the required subprograms for execution of Modules One, and Two or Three.

b. BLOCK DATA

The BLOCK DATA subprogram initializes variables in each of the common blocks.

c. SUBROUTINE DSCRPT

This subroutine is called by REGAD after an affirmative response to a user option to provide a brief description of the REGAD package. It contains an option to stop the program if only a program description is desired.

d. SUBROUTINE INPUT

All options and initial design parameters are entered via this subroutine which is called by REGAD.

e. SUBROUTINE AGMA

The constants for marine propulsion gears required by various AGMA formulations are initialized in the BLOCK DATA subprogram, and can be listed as an option in REGAD. REGAD calls this subroutine after an affirmative user response to display the preprogrammed values. This subroutine then allows the user to selectively change any constant desired.

2. Module Two : Parallel Axis

This module contains all the major subprograms called by REGAD to provide an initial design or to perform an analysis of conventional parallel axis reduction gears. The following is a grouping of the subprograms in Module Two.

a. SUBROUTINE PRLDES

This subroutine will produce a design of a parallel axis gear set. All pinion and gear diameters, effective facewidths, and gear ratios are computed using a basic random search optimization technique to find a feasible design by attempting to minimize a function of gear pair volume. It should be noted that, while attempting to minimize gear volume, the design is not necessarily optimized for minimum weight. The optimization technique is used here only to produce a feasible design in terms of dimension and power constraints by minimizing a function of gear pair volume. To produce a truly optimized design for minimum weight, a full optimization must include many more design variables such as helix and pressure angles, pitches, and hardnesses in addition to the dimensions. Additional constraints such as stress and unit load levels would need to be incorporated. All of this would require a more sophisticated and efficient optimization technique than is used here.

b. SUBROUTINE PRLANL

To analyze a proposed or existing design, REGAD will call this subroutine. It will request, as user-supplied input, the basic information calculated in PRLDES, i.e., pitch diameters and effective facewidths. Using this information, PRLANL will compute other parameters such as gear ratios, power and speed splits, and numbers of teeth per gear.

c. SUBROUTINE PRLRES

Immediately following a call to PRLDES or PRLANL, REGAD will call PRLRES to compute all remaining information such as expected loadings and stress levels. The user should be aware that the stress levels are computed according to AGMA formulations [Ref. 2 and 3] and take into account load distribution and overloads. This will produce levels that may seem high but are actually closer to actual levels to be expected in service.

d. SUBROUTINE PRLSIZ

REGAD calls this subroutine after PRLRES to compute estimates of gear set weight and gearbox dimensions. These estimates are determined by empirical relationships obtained from a rather limited data base of actual designs.

e. SUBROUTINE PRLOUT

This is the last subroutine called by REGAD in the parallel axis path. It provides a detailed output of the results obtained from the design or analysis including design parameters entered by the user, the dimensions of each component, expected loadings and stress levels, and configuration information.

3. Module Three : Epicyclic

Module Three contains all the major subprograms called by REGAD to design or analyze simple epicyclic reduction gears. Subroutines EPCDES, EPCANL, EPCRES, EPCSIZ, and EPCOUT are all analogous to those in Module Two. They perform the same functions, but for simple epicyclic gears. Therefore, individual descriptions will not be repeated here.

4. Module Four : Computational Subprogram Library

This module is an organizational grouping of all the subprograms called by those in Modules One, Two, and Three.

a. Subroutine Subprograms

The following are the subroutines used:

- (1) GFI - subroutine to compute the AGMA durability geometry factor, I
- (2) GFJ - subroutine to compute the AGMA strength geometry factor, J.

b. Real Function Subprograms

The following are the real function subprograms used:

- (1) ARCCOS - computes the arc cosine of two arguments
- (2) ARCSIN - computes the arc sine of two arguments
- (3) AGMAE1 - uses LaGrangian interpolation of Table E-1 [Ref. 1] to compute the constants required for the stress concentration factor formulation
- (4) CKDATA - called by SUBROUTINE AGMA to allow the user to change the preprogrammed constants
- (5) POWERB - computes allowable service power based on AGMA strength rating [Ref. 3]
- (6) POWERH - computes allowable service power based on AGMA durability rating [Ref. 2]
- (7) RTFNDR - a modified version of FUNCTION ZEROIN [Ref. 6] used to find a zero of a function in a specified interval
- (8) FALFA - the function required by SUBROUTINE GPJ and the zero of which is computed in FUNCTION RTFNDR
- (9) SHRLD - computes the load sharing ratio, m_n

- (10) THICK - computes tooth thickness at any diameter given a known thickness at a different diameter.

C. DATA TRANSFER

All data transfer between subprograms in Modules One, Two, and Three is via combinations of seven common blocks. Data transfer to and from subprograms in Module Four is via argument lists and common blocks as required. The following is a list of the common blocks used:

- (1) /AGMAB/ : constants for AGMA strength formulations
- (2) /AGMAH/ : constants for AGMA durability formulations
- (3) /DESDAT/ : design parameters and options
- (4) /DESPRL/ : parallel axis design information
- (5) /RESPRL/ : parallel axis computational results
- (6) /DESEPC/ : epicyclic design information
- (7) /RESEPC/ : epicyclic computational results.

The variables in each common block along with their definitions can be found in Appendix B.

D. PROGRAM OPERATION

REGAD is an interactive program designed to allow the user to solve his problem at a terminal. Being interactive, the program has many options that control program execution, in addition to requests for data necessary for the execution

of the program. Each request for information will contain the necessary guidelines needed by the the user to respond. This may take the form of a mini-table containing information on each option choice, the range of values when a specific quantity is requested, or units, where applicable, of the requested data.

All option parameters are integer values and should not be entered with a decimal. Option codes entered by the user are checked for validity to ensure they fall within the allowed range. If two options are offered, enter a 1 or a 2. Any value entered less than one will automatically default to one, and any value greater than two will automatically default to two. In those cases where there are more than two options, the response is checked to see if it falls within the allowed range. If it does not, a message alerts the user to this fact and allows him to re-enter the correct code. Some questions require affirmative or negative responses. To reply, use a Y for yes or an N for no. Use of other values may give undesirable results.

Every attempt has been made to anticipate possible error conditions. If one of these is encountered, a message is generated to inform the user. If the error encountered is a terminal error, the message will also indicate that the program run was aborted under program control.

A detailed development of this package is provided in Appendix A where specifics can be found. Appendix B provides a cross-reference of the variables used in Appendix A with those used in the program. It also contains detailed information on the common blocks. Sample runs of the program can be found in Appendix C, and a complete listing of the program is in Appendix D.

IV. CONCLUSIONS AND RECOMMENDATIONS

Computer aided design (CAD) is an important and useful tool for engineers. As computer technology continues to expand, CAD will become increasingly available for the practicing engineer, allowing him to use his initiative in design instead of being a slave to the numbers involved. REGAD is such a tool for use in the preliminary design of marine reduction gears during the conceptual stages of propulsion plant design.

REGAD could become even more useful if additional options are provided. A module to perform sensitivity analyses of a given design would greatly enhance the use of this program. This option would allow the user to start with any design and vary a selected variable over a specified range to determine its impact on the design. It could also be used to "fine tune" a design by modifying selected parameters to produce the results desired without having to rerun the program for each modification. Graphics would add another dimension by providing graphical displays of the gear arrangements and of certain data such as the results of a sensitivity analysis. A module to handle various composite designs of parallel axis and epicyclic gears would be an important addition. Also, it is recommended that a larger

data base be collected to provide more accurate empirical constants for the weight and gearbox size estimates.

APPENDIX A

PROGRAM DEVELOPMENT

With the exception of several general conversion relationships, all computations are accomplished in Modules Two and Three with calls to subprograms in Module Four. The analytical relationships used in the program will be examined, however, most of the relations used can be easily found in the literature and in various texts, so background developments will not be given.

I. GENERAL RELATIONSHIPS

The following relationships are used in Module One and in various other subprograms. The transverse diametral pitch of any gear is the ratio of its number of teeth to its pitch diameter;

$$P_d = \frac{N}{d} \quad (1)$$

The normal and transverse diametral pitches are related by;

$$P_d = P_{nd} \cos \psi \quad (2)$$

and the pressure angles by;

$$\tan \phi_n = \tan \phi \cos \psi \quad (3)$$

Axial pitch is defined as;

$$p_x = \frac{\pi}{P_{nd} \sin \psi} = \frac{\pi}{P_d \tan \psi} \quad (4)$$

II. CONVENTIONAL PARALLEL AXIS FORMULATIONS

Subroutines PRLANL and PRLDES each provide the pitch diameters of the pinions and gears, the effective face-widths, the stage reduction ratios, the numbers of teeth per gear, speed and power splits, and the geometry factors to subroutines PRLRES and PRLSIZ to compute all further information. The speed splits are the actual speeds of the individual gears and a power split is the actual power transferred by a gear. The strength and durability geometry factors are computed in separate subroutines in Module Four and will be discussed later.

A. COMMON RELATIONSHIPS

Power splits are determined from the configuration. For a single power path configuration, the power is transferred equally from the pinion to the gear, where in a dual power path configuration, the pinion transfers one half its power

to each of two gears. These splits are computed exactly since losses are neglected.

Speed splits and stage reduction ratios are based on;

$$m_g = \frac{D}{d} = \frac{n_g}{n_p} \quad (5)$$

Numbers of teeth on each gear are computed from the equation below and are rounded to the nearest integer.

$$N = d \times P_d \quad (6)$$

B. DETERMINATION OF DIAMETERS AND FACEWIDTHS

All diameters and facewidths are entered by the user in subroutine PRLANL. Stage gear ratios, power and speed splits and numbers of teeth per gear are computed as discussed in the previous section.

In subroutine PRLDES, the diameters, facewidths, and stage gear ratios are determined by using a basic local random search optimization technique to produce a feasible design. This algorithm requires an initial design to start.

The initial design is based on Dudley's [Ref. 7] formulation for preliminary estimates of gear size;

$$C^2F = \frac{31500}{K} \frac{Pwr}{n_p} \frac{(m_g + 1)^3}{m_g} \quad (7)$$

$$C = \frac{d}{2} (m_g + 1) \quad (8)$$

By substituting equation 8 into equation 7, a formula for estimating pinion diameter is obtained;

$$d^3 = \frac{126000}{n_p K} \frac{Pwr}{(F/d)} \frac{(m_g + 1)}{m_g} \quad (9)$$

where $F/d = 1.0$ for single helical gears and $F/d = 2.25$ for double helical gears. The term K is the K-factor which is an indication of durability. An expression for estimating K is provided by Thoma [Ref. 4];

$$K \leq \left(\frac{s_{ac} \times 10^{-4}}{C_R} \right)^2 \times \left(\frac{2.80}{C_o C_m} \right) \quad (10)$$

where the constants used are the AGMA durability constants. The K-factor in equation 10 is for the second reduction. For the first reduction, multiply K from equation 10 by 1.20. The initial estimates for the stage gear ratios are:

- (1) single reduction $m_g = M_o$
- (2) double reduction $m_{g_2} = \sqrt{M_o} + 3$ dual power path
 $m_{g_2} = \sqrt{M_o} - 1$ single power path
 $m_{g_1} = \sqrt{M_o} / m_{g_2}$
- (3) triple reduction $m_{g_2} = \sqrt[3]{M_o}$
 $m_{g_3} = \sqrt[3]{M_o} + 3$
 $m_{g_1} = \sqrt[3]{M_o} / m_{g_2} m_{g_3}$

The initial facewidths used are:

- (1) single helical gears $F = d$
- (2) double helical gears $F = 2.25 d$.

With this initial design as a starting point for the random search algorithm, successive designs are determined by randomly adding small amounts of between +1.0 and -1.0 to the diameters, facewidths, and stage gear ratios. These small amounts are scaled to take into account the difference in range of values for each variable. This process will attempt to find a feasible design in which all specified constraints are satisfied. If the initial design violates one or more constraints, the design that violates them the least in succeeding iterations will be kept until a design satisfying all constraints is found. Once a feasible design is found, an attempt to improve this design is made by trying to reduce the size of the gears by minimizing a function of gear pair volume;

$$\text{Volume} = \sum \sum C^2 F = \sum \sum \left[\frac{1}{4} (m_g + 1)^2 d^2 F \right] \quad (11)$$

The interior summation is over the number of reduction stages, and the exterior summation is over the number of power inputs. The constraints imposed which determine the limits on each of the designs are:

- (1) actual transmitted power is less than or equal to the allowable service power in accordance with references 2 and 3
- (2) maximum gear diameter of 200 inches due to manufacturing limitations
- (3) minimum facewidth greater than four axial pitches to ensure proper helical action
- (4) maximum facewidth less than the pinion pitch diameter for a single helical gear or 2.25 times the pinion pitch diameter for a double helical gear
- (5) pinions and gears in succeeding reduction stages are to be larger than those in the previous stage due to the greater amounts of torque carried
- (6) in dual power path arrangements, the gear ratio for each reduction stage is greater than the preceeding stage due to the torque carried.

The design obtained can than be adjusted by the user as desired by changing parameters with the analysis option.

III. EPICYCLIC FORMULATIONS

As in Module Two, the pitch diameters, effective face-widths, stage reduction ratios, numbers of teeth, speed and power splits, and the geometry factors are all entered or computed in the analysis or design subroutines (EPCANL or EPCDES) for use in the final computations subroutines (EPCRES and EPCSI2). Here, the speed splits are the rotational speeds of the sun and planet gears and of either the ring gear or the carrier, depending on the configuration. Planetary arrangements have fixed ring gears while star arrangements have fixed carriers. Also, the direction of rotation must be considered. Star arrangements reverse the direction of rotation of the input and the planetary arrangements will maintain direction of rotation. Assuming equal load sharing of the planets and neglecting losses, power splits are straightforward. The input and output powers are equal while each planet carries an equal share of the total power. Load sharing is an important consideration in the design of epicyclic gears, and must be assured in marine reduction gears due to the high power levels experienced. Equal load sharing of the planets can be reasonably achieved in several different ways. One method requires the sun gear to float, supported only by the planet gears, with

a relatively flexible ring gear to allow for inaccuracies in the teeth. There are also mechanical devices available to assist in achieving an equal division of the load. Experience has shown, for marine applications, that three to five planets with stage ratios in the range of two to eight work best.

A. COMMON RELATIONSHIPS

Unlike conventional parallel axis arrangements, there are specific numerical rules governing the proper assembly and operation of an epicyclic gear set. These involve the selection of the numbers of teeth and planets along with computing the various speed ratios. Mesh frequencies are also configuration dependent as seen in a following section.

There are basically three relationships that must be satisfied to ensure proper assembly and operation. The first is a relationship defining the speed ratio of the epicyclic stage since it is not merely the ratio of numbers of teeth or diameters as in a conventional parallel axis gear set. The second relationship requires the ring gear diameter to be equal to the sum of the sun gear diameter and twice a planet gear's diameter. This ensures the planets' ability to fit between the sun and ring gear. For the final relationship, it can be shown geometrically that the sum of the numbers of teeth on the sun gear and ring gear must be

an integral multiple of the number of planets in the gear set to ensure proper alignment and meshing of all teeth. It should be noted that these relationships are based on equally spaced planets around the sun gear. The above relationships are conveniently expressed in terms of numbers of teeth on each gear as seen in references 7 and 8. The speed ratio for a planetary arrangement is;

$$m_G = \frac{n_o}{n_i} = \frac{N_R}{N_S} + 1 \quad (12)$$

and for a star arrangement;

$$m_G = \frac{-n_o}{n_i} = \frac{-N_R}{N_S} \quad (13)$$

where the negative sign indicates the star arrangement's reversal of rotational direction of the input. The rotational speed of the planet gears is required for the design of their bearings and can be determined by;

$$n_{PLN} = \frac{N_R}{N_{PLN}} n_o \quad (14)$$

where n_o in each equation above is the speed of the carrier for a planetary arrangement or is the ring gear's speed for a star arrangement. The assembly and meshing relations in terms of tooth numbers are;

$$N_R = N_S + 2 N_{PLN} \quad (15)$$

and;

$$N_R + N_S = k NP \quad (16)$$

where k is an integer and NP is the number of planets.

B. DETERMINATION OF DIAMETERS AND FACEWIDTHS

The random search technique discussed for conventional parallel axis gears is used to provide the epicyclic diameters, facewidths, and stage gear ratios. Equation 9 is used to provide an initial estimate of sun gear diameters where m_g is replaced by the ratio of the planet's pitch diameter to the sun's pitch diameter. This value is usually in the range of 1.5 to 3; therefore, a random number in this range is used to start the problem. Once the sun gear diameter is estimated, the other diameters can be found using the relationships in equations 12 to 16. The initial estimates for the stage gear ratio are the roots of the overall ratio corresponding to the number of reduction stages. For example, m_{g_1} and m_{g_2} for a double reduction gear set would be the square root of the overall ratio. Initial facewidths are chosen as before. The initial estimates of the diameters, facewidths, and the gear ratios provide a starting point for

the random search algorithm discussed previously. Again, the method will attempt to improve feasible designs by minimizing a function of gear volume;

$$\text{Volume} = \sum (NP \cdot d_{PN}^2 + d_S^2 + d_R^2) \cdot F \quad (17)$$

where the summation is over the number of reduction stages. The constraints imposed are similar to those for the parallel axis gears:

- (1) actual transmitted power is less than or equal to the allowable service power in accordance with references 2 and 3
- (2) maximum ring gear diameter of 150 inches due to manufacturing limitations
- (3) minimum facewidth greater than four axial pitches to ensure proper helical action
- (4) maximum facewidth less than the sun's pitch diameter for a single helical gear or 2.25 times the sun's pitch diameter for a double helical gear
- (5) planet gears are to be larger than sun gears due to the greater amounts of torque carried
- (6) stage gear ratios are to be between 2 and 8 for each reduction stage.

As before, once a design is obtained, the user can utilize the analysis option to obtain the desired results.

IV. COMPUTATIONAL RESULTS AND DESIGN INFORMATION

Once the geometry is determined in the analysis or design subroutines, the computational results subroutine (PRLRES or EPCRES) and the size estimates subroutine (PRLSIZ or EPCSIZ) are called to provide design information concerning tooth loads, stresses, and other configuration, geometry, and size information. This section describes the formulations used.

The facewidth to diameter ratio is computed using the effective facewidth and the pitch diameter of the pinion for parallel axis gears or the sun gear for epicyclics. Center distance is taken as the average of the pinion and gear pitch diameters. A center distance is computed for epicyclics by finding the average of the sun and a planet gears' pitch diameters.

Pitchline velocity, V , is determined by;

$$V = \frac{\pi d n_p}{12} \quad (18)$$

where V is in feet per minute, d is in inches, and n_p is in revolutions per minute. The tangential component of tooth load, W_t , is computed from;

$$W_t = \frac{126000 \text{ Pwr}}{n_p d} \quad (19)$$

where W_t is in pounds-force, Pwr is in horsepower, and d and n_p are as before. Tooth loading per inch of facewidth is computed from;

$$\text{Tooth Load per Inch} = W_t / F \quad (20)$$

and the unit load, a normalized value of the load per inch above, is;

$$\text{Unit Load} = \frac{W_t P_{nd}}{F} \quad (21)$$

where the unit load is expressed in pounds-force per square inch.

Mesh frequencies provide information on how often a tooth is loaded. Mesh frequencies for parallel axis gears are determined by;

$$f = \frac{N_p n_p}{60} \quad (22)$$

with f expressed in Hertz. For epicyclic gears, the following are used:

$$\begin{aligned}
(a) \quad f_s &= \frac{NP}{N_R + N_S} \frac{N_R}{N_S} n_s & (d) \quad f_s &= NP \frac{N_S}{N_R} n_s \\
(b) \quad f_p &= \frac{N_R}{N_{PLN}} \frac{N_S}{N_R + N_S} n_s & (e) \quad f_p &= 2 \frac{N_S}{N_{PLN}} n_s \\
(c) \quad f_R &= \frac{NP}{N_R + N_S} \frac{N_S}{N_R} n_s & (f) \quad f_R &= NP \frac{N_S}{N_R} n_s
\end{aligned} \tag{23}$$

where (a) through (c) are for planetary arrangements and (d) through (f) are for star arrangements.

The K-factor is computed for reference purposes by;

$$K = \frac{W_t}{F d} \frac{(m_G + 1)}{m_G} \tag{24}$$

The contact stresses are computed according to reference 2 by;

$$s_c = C_p \sqrt{\frac{W_t C_o}{C_v} \frac{C_s}{d F} \frac{C_m C_t}{I}} \tag{25}$$

Bending stresses are computed according to reference 3 by;

$$s_t = \frac{W_t K_o}{K_v} \frac{P_d}{F} \frac{K_s K_m}{J} \tag{26}$$

Individual torques, T, are found by;

$$T = \frac{W_t \cdot d}{2000} \quad (27)$$

while the total output torque is computed by;

$$T = \frac{63 \text{ SHP}}{n_p} \quad (28)$$

where T has the units of thousands of inch-pounds-force in both cases. Shaft horsepower, SHP, is the total power transferred to the output shaft.

Weight and size estimates are based on empirical relations obtained from a limited number of actual designs. The relations used are;

$$\begin{aligned} \text{Weight} &= C1 \cdot [\sum (d^2 F)]^{c2} \\ \text{Length} &= C3 \cdot \sum F \\ \text{Width} &= C4 \cdot D \\ \text{Height} &= C5 \cdot D \end{aligned} \quad (29)$$

where the constants used are found in Table 1. All dimensions are in inches and the weight is in pounds-force rounded to three significant figures.

Table 1: Empirical Constants for Weight and
Size Formulations

<u>Constant</u>	<u>Parallel Axis</u>	<u>Epicyclic</u>
C1	1196.0	0.905
C2	0.34	0.89
C3	2.26	2.85
C4	1.20	1.30
	1.37	--
C5	1.28	1.20
D	Bull Gear Diameter	Ring Gear Diameter

first C4: for single power inputs
second C4: for double power inputs

V. COMPUTATIONAL SUBPROGRAMS LIBRARY FORMULATIONS

The formulations provided below are for the major computational subprograms in Module Four. Those that are self-explanatory or are not computational in nature are only described in general.

A. ARCCOS AND ARCSIN

These function subprograms find the arc cosine and arc sine, respectively, for any two arguments. They were added for convenience since not all compilers have them as internal functions.

B. AGMAE1

This function subprogram returns the value of the constants H, L, and M required for the determination of the stress concentration factor, K_t , according to reference 1, for use in computing the strength geometry factor, J. Table E-1 in reference 1 provides the tabulated data necessary to perform a LaGrangian interpolation for each constant for a specified normal pressure angle in degrees. The interpolation formula used is:

$$F(\phi_n) = \frac{(\phi_n - 20)(\phi_n - 14.5)}{57.75} F_1 + \frac{(\phi_n - 14.5)(\phi_n - 25)}{-27.50} F_2 + \frac{(\phi_n - 14.5)(\phi_n - 20)}{52.50} F_3 \quad (30)$$

where F represents the appropriate values of H, L, or M.

C. CKDATA

FUNCTION CKDATA is called by subroutine AGMA to allow the user to selectively change the preprogrammed constants by checking if the value entered is zero. If it is zero, the current value of the specified constant is not changed. This provides for flexibility in changing constants with multiple values, and it guards against inadvertently entering a value of zero.

D. POWERB AND POWERH

These function subprograms are used to compute the maximum allowed service power, in horsepower, that can be transmitted by a gear according to references 2 and 3. The formulation based on the strength rating is;

$$P = \frac{n \ d \ K_v}{126000 \ SF \ K_o} \frac{F}{K_m} \frac{J}{K_s \ P_d} \frac{S_{ac} \ K_L}{K_R \ K_T} \quad (31)$$

and the durability rating formulation is;

$$P = \frac{n \ d}{126000 \ SF} \frac{I \ C_v}{C_s \ C_f \ C_o \ C_m} \left[\frac{S_{at} \ d}{C_p} \frac{C_L \ C_H}{C_R \ C_T} \right]^2 \quad (32)$$

where J and I are the respective geometry factors, F is the effective facewidth, n is the speed of d in revolutions per minute, and d is the pinion pitch diameter for parallel axis or is the sun pitch diameter for epicyclics. All other values are the preprogrammed constants.

E. RTFNDR AND FALFA

The function subprogram RTFNDR, a slightly modified version of FUNCTION ZEROIN [Ref. 6], is used to find the value of the root of the equation programmed in function FALFA. This root is required by the subroutine GFJ for the computation of the strength geometry factor, J.

F. SHRLD

This function subprogram computes the load sharing ratio used in computing the geometry factors. The load sharing ratio, m_N , is determined by;

$$m_N = \frac{p_N}{.95 Z} = \frac{\pi \cos \phi_n}{.95 Z p_{nd}} \quad (33)$$

where Z is the length of action defined as;

$$Z = \frac{1}{2} \left(\sqrt{D_o^2 - D_b^2} + \sqrt{d_o^2 - d_b^2} - \sqrt{D^2 - D_b^2} - \sqrt{d^2 - d_b^2} \right) \quad (34)$$

The subscripts on the pitch diameters are:

(1) o : outside diameter; $d_o = d + (2/P_d)$

(2) b : base diameter; $d_b = d \cos \phi_i$

For epicyclics, replace the outside diameters in equation 34 with inside diameters : $d_i = d - (2/P_d)$.

G. THICK

FUNCTION THICK returns the value of the normal arc thickness of a tooth at a specified diameter given a thickness at another diameter. For external gears;

$$t_2 = d_2 \left((t_1/d_1) + \text{inv } \phi_1 - \text{inv } \phi_2 \right) \quad (35)$$

and for internal gears;

$$t_2 = d_2 \left((t_1/d_1) - \text{inv } \phi_1 + \text{inv } \phi_2 \right) \quad (36)$$

where the subscript 2 represents the desired point and subscript 1 represents the known point. The involute function is defined as;

$$\text{inv } x = \tan x - x \quad (37)$$

The arguments of the involute functions in equations 35 and 36 are the transverse pressure angles at the points under consideration. The pressure angle at the desired point is defined as;

$$\cos \phi_2 = \frac{d_1 \cos \phi_1}{d_2} \quad (38)$$

The known point is usually taken at the pitch circle where $d_1 = d$, $\phi_1 = \phi_n$, and t_1 is defined as

$$t = \frac{P_n}{2} = \frac{\pi}{2 P_d} \cos \psi \quad (39)$$

H. GFI

This subroutine is used to compute the AGMA durability geometry factor, I , in accordance with reference 2. The geometry factor is defined as;

$$I = \frac{\cos \phi_1 \sin \phi_1}{2 m_N} \frac{m_G}{(m_G \pm 1)} \quad (40)$$

where m_N is computed by function SHRLD described above. The plus sign applies to external gears and the minus sign applies to internal gears.

I. GFJ

SUBROUTINE GFJ is used to compute the AGMA strength geometry factor, J , in accordance with reference 1 with one major difference: the values used are from analytical developments and are not scaled to a normal diametral pitch of one as are the values used in a graphical layout discussed in reference 1. The strength geometry factor is defined as;

$$J = \frac{Y_C \cos^2 \psi}{K_t m_N} \quad (41)$$

The load sharing ratio, m_N , is computed in FUNCTION SHRLD. The stress concentration factor, K_t , is determined from;

$$K_t = H + \left(\frac{t}{r_t} \right)^L \cdot \left(\frac{t}{h} \right)^M \quad (42)$$

where H , L , and M are determined in FUNCTION AGMAE1. The value of the root fillet radius, r_t , is;

$$r_t = r_r + \frac{(b - r_r)^2}{(d/2 \cos^2 \psi) + (b - r_r)} \quad (43)$$

with the dedendum, $b = 1.25/P_d$, and the root tip radius,

$\tau_r \cong 0.28/P_{nd}$. The values of t and h are determined from the analytical geometry of the tooth form layout described below.

The tooth form factor, Y , is defined as;

$$Y_c = P_{nd} \left[\frac{\cos \phi_{Ln}}{\cos \phi_n} \left(\frac{1.5}{x C_n} - \frac{\tan \phi_{Ln}}{t} \right) \right]^{-1} \quad (44)$$

where t and x are also from the tooth form layout mentioned previously. The helical factor, C_n , is defined as;

$$C_n = \left[1 - \frac{\nu}{100} \left(1 - \frac{\nu}{100} \right) \right]^{-1} \quad (45)$$

where $\tan \nu = \tan \psi \sin \phi_n$ for $\psi \leq 50^\circ$. The normal load pressure angle at the tip of the tooth, ϕ_{Ln} , can be seen in figures 6 and 7 and is given by;

$$\phi_{Ln} = \cos^{-1} \left(\frac{d_b}{d_o} \right) \pm \frac{t_o}{d_o} \quad (46)$$

where the subscript o pertains to the point on the outside diameter and subscript b pertains to the base circle. The plus sign applies to internal gears and the minus, to external. The thickness, t_o , at the outside diameter is determined by function THICK. For internal gears, replace the outside values with the inside values as before.

The graphical tooth form layout is a method by which the variables h , t , and x can be determined from actual

measurements of a tooth form drawn and scaled for a normal diametral pitch of one for the case where tooth loading is at the tip. Loading at the tip of the tooth is the general practice for considering loads on helical gears. Refer to Figure 7 for the meanings of h , x , and T where $t = 2T$. Before determining h , x , and T analytically, several reference parameters must be determined as suggested by McIntire and Lyon [Ref. 9]. The first is the radius from the center of the gear to the tip of the inscribed Lewis stress parabola which is point E in Figure 7. This point is the intersection of the line of action of the tip load, tangent to the base circle, with the tooth centerline. The radius to this point is;

$$r_v = \frac{d_v}{2} = \frac{d_b}{2\cos \phi_{Ln}} \quad (47)$$

An additional reference point is required to fix the geometry. The center of of the root fillet is taken as this point which can be obtained by a very close approximation. To locate this point, the gear center is taken as the origin of a cartesian coordinate system with the tooth centerline as the vertical axis. Two possible cases exist for the location of this point with respect to the base circle. Figure 8 shows the case where the point is inside the base circle and Figure 9 shows the case where it is outside. The

coordinates of this point, (XC,YC), can be found from Figures 8 and 9. For both cases it can be seen in Figures 8 and 9 that;

$$HYP = d_n + r, \quad (48)$$

where $d_n = d - 2b = d - (2.5/P_s)$. From Figure 6, the angle, ϵ , is;

$$\epsilon = \text{inv } \phi + \sin^{-1} \frac{t_c}{d} \quad (49)$$

where t_c is the chordal tooth thickness given by;

$$t_c = t - \frac{t^3 \cos^2 \psi}{6 d^3} \quad (50)$$

and t is the normal arc tooth thickness defined earlier. For the case in Figure 8;

$$XX = (HYP) \sin \epsilon$$

$$XC = XX + r, \quad (a)$$

$$YC = \sqrt{HYP^2 - XC^2} \quad (b) \quad (51)$$

and for the case in Figure 9;

$$\phi_1 = \cos^{-1} \frac{(d_b/2)}{HYP}$$

$$OPP_1 = (HYP) \sin \phi_1$$

$$OPP_2 \cong OPP_1 - r_1$$

$$HYP_1 = \sqrt{OPP_2^2 + (d_b/2)^2}$$

$$\phi_2 = \cos^{-1} \frac{(d_b/2)}{HYP_1}$$

$$\lambda = \phi_1 \pm \text{inv } \phi_2 - \phi_2$$

"-" for external gears

"+" for internal gears (see Figure 11)

$$\delta = \lambda + \epsilon$$

$$XC = (HYP) \sin \delta \quad (a)$$

$$YC = (HYP) \cos \delta \quad (b)$$

(52)

With the reference values of r_1 , XC , and YC determined, the values of h , $t=2T$, and x can be analytically determined. From Figure 7;

$$XT = r_1 \cos \alpha \quad (53)$$

$$YH = r_1 \sin \alpha \quad (54)$$

$$h = r_1 - YC + YH = r_1 - YC + r_1 \sin \alpha \quad (55)$$

$$T = (t/2) = XC - XT = XC - r_1 \cos \alpha \quad (56)$$

$$YK = \frac{T}{\tan \alpha} \quad (57)$$

where α must be determined such that;

$$YK = 2h \quad \text{or} \quad YK - 2h = 0 \quad (58)$$

Substituting equations 53 through 57 into 58 yields;

$$F(\alpha) = XC - r, \cos \alpha - 2 \tan \alpha (r, -YC + r, \sin \alpha) = 0 \quad (59)$$

Equation 59 is the function in FALFA called by RTFNDR to solve for α . Once α is determined, h can be determined from equation 55 and T and t from equation 56. To obtain x , observe the following;

$$\gamma = \tan^{-1} (h/T)$$

$$\gamma_1 = (\pi/2) - \gamma$$

and

$$x = T \tan \gamma_1 \quad (60)$$

While not precise, the identical methodology is used for internal gears. Figures 10 and 11 apply. The expressions for internal gears are given without further development;

$$\begin{aligned}
h &= -r_v + YC + r, \sin \alpha \\
T &= XC - r, \cos \alpha \\
t &= 2T \\
\gamma &= \tan^{-1} (h/T) \\
\gamma_1 &= (\pi/2) - \gamma \\
x &= T \tan \gamma_1
\end{aligned}
\tag{61}$$

The values for h , t , and x are now used to determine the stress concentration factor, equation 42, and the tooth form factor, equation 44, required to compute the strength geometry factor, J , in equation 41.

APPENDIX B

LIST OF PARAMETERS

While it is not practical to list all variables used in the formulations or the program, it is useful to provide a list of the major variables with a cross-reference between the analytical names and the FORTRAN names. A detailed listing of each common block is also useful when studying the program.

I. PARAMETER CROSS-REFERENCE

This section provides a listing of parameters with both their analytical and FORTRAN names.

<u>Math</u> <u>Symbol</u>	<u>FORTRAN</u> <u>Name</u>	<u>Variable</u> <u>Definition</u>
K_l	AKL	life factor
K_m	AKM	load distribution factor
K_o	AKO	overload factor
K_R	AKR	reliability factor
K_s	AKS	size factor
K_T	AKT	temperature factor
K_v	AKV	dynamic factor
SP	SPB	service factor
C	CDE	center distance (theoretical) (in)

	CDP	(E=epicyclic, P=parallel axis)
C_f	CF	surface finish factor
C_H	CH	hardness factor
C_L	CL	life factor
C_m	CM	load distribution factor
C_o	CO	overload factor
C_p	CP	elastic properties factor
C_R	CR	reliability factor
C_s	CS	size factor
C_T	CT	temperature factor
C_v	CV	dynamic factor
SP	SPH	service factor
ψ	DHELIX	helix angle (deg)
	HELIX	helix angle (rad)
ϕ_t	DPHI	transverse pressure angle (deg)
	PHI	transverse pressure angle (rad)
ϕ_n	DPHIN	normal pressure angle (deg)
	PHIN	normal pressure angle (rad)
D	DG	diameter of gear (in)
d	DP	diameter of pinion (in)
d_{PLN}	DPLN	diameter of planet gears (in)
d_R	DR	diameter of ring gear (in)
		root diameter of a gear (in)
d_s	DS	diameter of sun gear (in)
F	FACEE	facewidth (in)
	FACEP	(E=epicyclic, P=parallel axis)

F/d	FBYDE	f/d ratio (facewidth/diameter)
	FBYDP	(E=epicyclic, P=parallel axis)
I	GEOMI	durability geometry factor (pinion)
	GI	durability geometry factor (sun)
J	GEOMJG	strength geometry factor (gear)
	GEOMJP	strength geometry factor (pinion)
	GJS	strength geometry factor (sun)
	GJPL	strength geometry factor (planet)
K	KFCTRE	computed k-factor
	KFCTRP	(E=epicyclic, P=parallel axis)
f	MFE	mesh frequency (Hz)
	MFP	(E=epicyclic, P=parallel axis)
M _o	MGOE	overall reduction ratio
	MGOP	(E=epicyclic, P=parallel axis)
m _g	MGE	stage reduction ratio
	MGP	(E=epicyclic, P=parallel axis)
N _G	NG	number of teeth, gear
N _p	NP	number of teeth, pinion
NP	NPLNT	number of planet gears in epicyclic set
N _{PLN}	NPLN	number of teeth, planet
N _R	NR	number of teeth, ring
N _S	NS	number of teeth, sun
P _d	PD	transverse diametral pitch
P _{nd}	PND	normal diametral pitch
V	PLVE	pitch line velocity (fpm)
	PLVP	(E=epicyclic, P=parallel axis)

PWR	PWRE	power split per gear pair (hp)
	PWRP	(E=epicyclic, P=parallel axis)
n_{in}	RPMIN	source speed input (rpm)
n_{out}	RPMOUT	output shaft/propeller speed (rpm)
n_i	RPMI	stage input speed, epicyclic (rpm)
n_o	RPMO	stage output speed, epicyclic (rpm)
n_{PLM}	RPMPL	planet speed, epicyclic (rpm)
n_p, n_g	RPMP	stage pinion and gear speed, parallel axis (rpm)
S_{ac}	SAC	allowable contact stress number
S_{at}	SAT	allowable bending stress number
SHP	SHP	shaft horsepower, output (hp)
s_i	SIGBE	bending stress (psi)
	SIGBP	(E=epicyclic, P=parallel axis)
s_c	SIGHE	contact stress (psi)
	SIGHP	(E=epicyclic, P=parallel axis)
T	TORQE	torque (k in-lb)
	TORQP	(E=epicyclic, P=parallel axis)
W_t	WTE	tangential tooth load (lb)
	WTP	(E=epicyclic, P=parallel axis)

II. COMMON BLOCK DETAILS

The following provides information concerning the variables in each common block. The numbers in parentheses are the size of the array where applicable.

COMMON BLOCK AGMAB (FOR STRENGTH RATING)

SFB : R⁴ (2,2); service factor
AKV : R⁴; dynamic factor
AKS : R⁴; size factor
AKM : R⁴; load distribution factor
AKO : R⁴ (2); overload factor
SAT : R⁴ (6); allowable bending stress number
AKL : R⁴ (2); life factor
AKR : R⁴ (6); reliability factor
AKT : R⁴; temperature factor

COMMON BLOCK AGMAH (FOR DURABILITY RATING)

SFH : R⁴ (2,2); service factor
CV : R⁴ (3); dynamic factor
CS : R⁴; size factor
CM : R⁴ (2); load distribution factor
CF : R⁴; surface finish factor
CO : R⁴ (2); overload factor
SAC : R⁴ (6); allowable contact stress number
CP : R⁴; elastic properties factor
CL : R⁴ (2); life factor
CH : R⁴; hardness factor
CT : R⁴; temperature factor
CR : R⁴ (6); reliability factor

COMMON BLOCK DESDAT (DESIGN PARAMETERS, INPUT)

PWRIN : R*4 (2); source power input (hp)
RPMIN : R*4 (2); source speed input (rpm)
RPMOUT: R*4; output shaft/propeller speed (rpm)
DHELIX: R*4 (3); helix angle (deg)
HELIX : R*4 (3); helix angle (rad)
PD : R*4 (3); transverse diametral pitch
PND : R*4 (3); normal diametral pitch
DPHI : R*4 (3); transverse pressure angle (deg)
PHI : R*4 (3); transverse pressure angle (rad)
DPHIN : R*4 (3); normal pressure angle (deg)
PHIN : R*4 (3); normal pressure angle (rad)
NDIFP : I*4; number of different power sources
IARR : I*4; arrangement code (1=parallel axis,
2=epicyclic)
IEPIC : I*4 (3); epicyclic code (1=planetary, 2=star)
IHARD : I*4 (3,2); hardness range code (1-6, see SUBR.
AGMA)
IOPRO : I*4; operational profile code (1=naval pro-
file full power 5% max; 2=other, max
power continuous)
NPWRIN: I*4; number of power sources (inputs)
IPWRSR: I*4 (2); power source code (1=turbine or motor,
2=multicylinder internal combustion
engine)
NRED : I*4; number of reduction stages

NPATH : I*4; number of power paths (1=single,2=dual)
 NPLNT : I*4 (3); number of planet gears in epicyclic set
 NHELX : I*4; number of helicies (1=single, 2=double)

COMMON BLOCK DESEPC (EPICYCLIC DESIGN PARAMETERS)

MGOE : R*4; overall reduction ratio
 MGE : R*4 (3); stage reduction ratio
 RPMI : R*4 (3); stage input speed (rpm)
 RPMPL : R*4 (3); planet speed (rpm)
 RPMO : R*4 (3); stage output speed (rpm)
 PWRE : R*4 (3); stage power split per planet (hp)
 DS : R*4 (3); diameter of sun gear (in)
 DPLN : R*4 (3); diameter of planet gears (in)
 DR : R*4 (3); diameter of ring gear (in)
 FACEE : R*4 (3); facewidth (in)
 GI : R*4 (3); durability geometry factor (sun/planet)
 GJS : R*4 (3); strength geometry factor (sun)
 GJPL : R*4 (3); strength geometry factor (planet)
 NS : I*4 (3); number of teeth, sun
 NPLN : I*4 (3); number of teeth, planet
 NR : I*4 (3); number of teeth, ring

COMMON BLOCK DESPRI (PARALLEL AXIS DESIGN PARAMETERS)

PWRPAC: R*4 (2,3); stage power split factor
 MGOP : R*4 (2); overall reduction ratio
 MGP : R*4 (3,2); stage reduction ratio

RPM_P : R*4 (6,2); stage pinion and gear speed (rpm)
 PWR_P : R*4 (6,2); stage power split per gear (hp)
 DP : R*4 (3,2); diameter of pinion (in)
 DG : R*4 (3,2); diameter of gear (in)
 FACE_P : R*4 (3,2); facewidth (in)
 GEOM_I : R*4 (3,2); durability geometry factor
 GEOM_{JG} : R*4 (3,2); strength geometry factor (gear)
 GEOM_{JP} : R*4 (3,2); strength geometry factor (pinion)
 NP : I*4 (3,2); number of teeth, pinion
 NG : I*4 (3,2); number of teeth, gear

COMMON BLOCK RESEPC (EPICYCLIC PARAMETERS, RESULTS)

PLVE : R*4 (3); pitch line velocity (fpm)
 FBYDE : R*4 (3); f/d ratio (facewidth/sun diameter)
 CDE : R*4 (3); center distance (theoretical) (in)
 WTE : R*4 (3); tangential tooth load (lb)
 TLPIE : R*4 (3); tooth load per in (lb/in)
 UNTLDE : R*4 (3); unit load (psi)
 MFE : R*4 (3,3); mesh frequency (Hz)
 KFCTRE : R*4 (3); computed k-factor
 SIGHE : R*4 (3); contact stress (psi)
 SIGBE : R*4 (3); bending stress (psi)
 TORQUE : R*4 (3,3); torque (k in-lb)
 RPME : R*4 (3,3); gear speeds (rpm)
 PDIAME : R*4 (3,3); pitch diameters (in)
 WGHTTE : R*4; gear set weight estimate (lb)

SPCWTE: R*4; specific weight (lb/hp)
 MTHZ : I*4 (3,3); tooth numbers
 ISIZEE: I*4 (3); length, width, height estimates (in)

COMMON BLOCK RESPRI (PARALLEL AXIS PARAMETERS, RESULTS)

PLVP : R*4 (3,2); pitch line velocity (fpm)
 FBYDP : R*4 (3,2); f/d ratio (facewidth/pinion diameter)
 CDP : R*4 (3,2); center distance (theoretical) (in)
 WTP : R*4 (6,2); tangential tooth load (lb)
 TLPIP : R*4 (6,2); tooth load per inch (lb/in)
 UNTLDP: R*4 (6,2); unit load (psi)
 MFP : R*4 (3,2); mesh frequency (Hz)
 KFCTRP: R*4 (6,2); computed k-factor
 SIGHP : R*4 (3,2); contact stress (psi)
 SIGBP : R*4 (6,2); bending stress (psi)
 TORQP : R*4 (6,2); torque (k in-lb)
 PDIAMP: R*4 (6,2); pitch diameters (in)
 SCDMIN: R*4; minimum source center distance (in)
 SCDMAX: R*4; maximum source center distance (in)
 SHP : R*4; shaft horsepower, output (hp)
 WGHTP : R*4; gear set weight estimate (lb)
 SPCWTP: R*4; specific weight (lb/hp)
 TRQOUT: R*4; torque, output (k in-lb)
 MTHP : I*4 (6,2); tooth numbers
 ISIZEP: I*4 (3); length, width, height estimates (in)

APPENDIX C

REGAD SAMPLE RUNS

This appendix contains samples of actual terminal sessions using REGAD. For the sake of brevity, only two complete sessions are included. However, a number of analysis and design runs were made using a full range of options and configurations, and they compared favorably to actual designs. The comparisons are not shown here due to the proprietary nature of the designs used for verification. The first example is an analysis run for a locked train, double reduction gear set with two different inputs. Following it, is the results section from a design run using the identical parameters as the analysis run. The second example is a double reduction epicyclic gear set with the complete analysis session followed by the results section of a design run as before. The analysis and design sessions are identical with one exception. A seed for a random number generator is requested in the design option instead of diameters and facewidths as in the analysis option. For those cases where an infeasible design is generated, a message will alert the user and the program will continue. To obtain a feasible design, or just a different one, rerun the program and

provide a different seed for the random number generator. This method was used on several occasions to obtain the desired results. Once a feasible design is obtained, the user can then use the analysis option to obtain a design that more closely suits his needs.

REGAD REDUCTION GEAR ANALYSIS AND DESIGN

DO YOU DESIRE A PROGRAM DESCRIPTION? (Y OR N) :

卷之五

THIS PROGRAM IS CAPABLE OF PERFORMING PRELIMINARY DESIGN OR ANALYSIS OF MULTIREDUCTION, PARALLEL AXIS AND EPICYCLIC REDUCTION GEARS. THE CAPABILITIES AND FEATURES OF THE PROGRAM ARE AS FOLLOWS:

- 1) MAXIMUM OF THREE REDUCTION STAGES ALLOWED
- 2) CHOICE OF SINGLE OR DOUBLE HELICALS
- 3) WEIGHT AND SIZE ESTIMATES PROVIDED
- 4) FOR PARALLEL AXIS GEARS:
 - ONE OR TWO POWER SOURCES ALLOWED
 - SINGLE OR DUAL POWER PATHS ALLOWED
- 5) FOR EPICYCLIC GEARS:
 - ONLY ONE POWER SOURCE ALLOWED
 - LIMITED TO 3, 4, OR 5 PLANET GEARS
 - ONLY SIMPLE EPICYCLICS PER REDUCTION STAGE
 - PLANETARY OR STAR ARRANGEMENTS POSSIBLE

THE STANDARDS OF THE AMERICAN GEAR MANUFACTURING ASSOCIATION WERE USED AS A BASIS FOR THIS PROGRAM. THE CONSTANTS

USED IN THE AGMA FORMULATIONS ARE BASED ON THOSE PUBLISHED BY P. A. THOMA, OF DELAVAL TURBINE, FOR MARINE PROPULSION GEARS. AN OPTION IS PROVIDED DURING EXECUTION OF THE PROGRAM TO OBTAIN A LISTING OF THESE CONSTANTS, AND TO CHANGE ANY OF THEM FOR OTHER POSSIBLE APPLICATIONS.

IT SHOULD BE NOTED THAT THE STRESSES LISTED IN THE OUTPUT ARE THOSE COMPUTED FROM THE AGMA FORMULATIONS AND ARE NOT FROM A DETAILED STRESS ANALYSIS.

FOR MORE SPECIFIC INFORMATION, SEE THE USERS MANUAL OR OBTAIN A LISTING OF THE PROGRAM.

DO YOU WISH THE PROGRAM TO CONTINUE INTO THE ANALYSIS AND DESIGN SEGMENTS? (Y OR N):

Y

YOU WILL NOW BE ASKED TO PROVIDE THE PARAMETERS REQUIRED FOR THE ANALYSIS OR DESIGN IN THIS RUN.

** ENTER PROGRAM OPTION CODE (1=DESIGN, 2=ANALYSIS):

?
2

** ENTER ARRANGEMENT CODE (1=PARALLEL AXIS, 2=EPICYCLIC):

?
1

CHOOSE OPERATIONAL PROFILE CODE BELOW:
OPERATIONAL MODE SERVICE PROFILE CODE

FULL POWER 5 PERCENT MAX 1
 MAXIMUM LOAD CONTINUOUS 2

** ENTER OPERATIONAL PROFILE CODE:

?
 1

** ENTER NUMBER OF REDUCTIONS (1, 2, OR 3):

?
 2

CHOOSE DESIRED HELIX TYPE BELOW:

TYPE	ANGLE	CODE
SINGLE	15-25	1
DOUBLE	25-50	2

** ENTER HELIX CODE:

?
 2

** ENTER NUMBER OF POWER PATHS (1=SINGLE, 2=DUAL):

?
 2

** ENTER NUMBER OF POWER SOURCES (1 OR 2):

?
 2

WILL ANY POWER SOURCE BE A MULTICYLINDER I. C. ENGINE? (Y OR N):

N

** ENTER POWER AND SPEED OF HIGH POWER SOURCE (HP,RPM):

?
 21250,6990

ENTER POWER AND SPEED OF LOW POWER SOURCE (HP, RPM) :

?

21250, 5980

ENTER OUTPUT SHAFT/PROPELLER SPEED (RPM) :

?

300

WHICH DIAMETRAL PITCH WILL YOU SPECIFY?

(1=TRANSVERSE, 2=NORMAL) :

?

1

WHICH PRESSURE ANGLE WILL YOU SPECIFY?

(1=TRANSVERSE, 2=NORMAL) :

?

2

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 1 :

ENTER HELIX ANGLE (DEGREES) :

?

35

ENTER TRANSVERSE DIAMETRAL PITCH:

?

4.5

ENTER NORMAL PRESSURE ANGLE (DEGREES) :

?

20

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN CODE

160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4
360 - 400	5
400 - 640	6

ENTER HARDNESS CODES FOR PINION AND GEAR (HCPIN, HCGEAR):

?
2,1

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 2 :

ENTER HELIX ANGLE (DEGREES):

?
35

ENTER TRANSVERSE DIAMETRAL PITCH:

?
3.5

ENTER NORMAL PRESSURE ANGLE (DEGREES):

?
20

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN CODE

160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4
360 - 400	5

400 - 640 6

ENTER HARDNESS CODES FOR PINION AND GEAR (HCPIN, HCGEAR):
?
2,1

TO MAKE CORRECTIONS TO DATA JUST ENTERED, THE PROGRAM MUST BE
ABORTED AND RE-STARTED. DO YOU WISH TO ABORT THIS RUN? (Y OR N):
n

DO YOU DESIRE A LISTING OF THE PRE-PROGRAMMED CONSTANTS USED
IN THE AGMA FORMULATIONS? (Y OR N):
y

THE FOLLOWING IS A LISTING OF THE PRE-PROGRAMMED CONSTANTS
USED IN THE AGMA FORMULATIONS WITH APPROPRIATE NOTES ON
THEIR APPLICATION. NOTE: THOSE STARTING WITH A 'C' ARE
DURABILITY CONSTANTS AND THOSE WITH A 'K' ARE STRENGTH
CONSTANTS. SERVICE FACTOR APPLIES TO BOTH FORMULATIONS.

ID	CONST	VALUE(S)	NOTES
1	SF(1,1)	1.00	SERVICE FACTOR; A1, B1
	SF(1,2)	1.50	A1, B2
	SF(2,1)	1.50	A2, B1
	SF(2,2)	1.75	A2, B2
2	CV(1)	1.00	DYNAMIC FACTOR; C1
	CV(2)	0.83	C2
	CV(3)	0.69	C3
3	CS	1.00	SIZE FACTOR
4	CM(1)	1.25	LOAD DISTRIBUTION FACTOR; A1
	CM(2)	1.35	A2

5	CP	1.00	SURFACE CONDITION FACTOR
6	CO(1)	1.15	OVERLOAD FACTOR; A1
	CO(2)	1.14	A2
7	CP	2300.0	ELASTIC PROPERTIES FACTOR
8	CL(1)	0.80	LIFE FACTOR; A1
	CL(2)	0.68	A2
9	CH	1.00	HARDNESS RATIO FACTOR
10	CT	1.00	TEMPERATURE FACTOR
11	CR(1)	1.16	RELIABILITY FACTOR; D1
	CR(2)	1.19	D2
	CR(3)	1.22	D3
	CR(4)	1.27	D4
	CR(5)	1.31	D5
	CR(6)	1.35	D6
12	SAC(1)	95000.	ALLOWABLE CONTACT STRESS; D1
	SAC(2)	108000.	D2
	SAC(3)	125000.	D3
	SAC(4)	146000.	D4
	SAC(5)	165000.	D5
	SAC(6)	182000.	D6
13	KV	0.70	DYNAMIC FACTOR
14	KS	1.00	SIZE FACTOR
15	KM	1.10	LOAD DISTRIBUTION FACTOR
16	KO(1)	1.21	OVERLOAD FACTOR; E1

	KO(2)	1.28	E2
17	KL(1)	0.80	LIFE FACTOR; A1
	KL(2)	0.68	A2
18	KT	1.00	TEMPERATURE FACTOR
19	KR(1)	1.16	RELIABILITY FACTOR; D1
	KR(2)	1.18	D2
	KR(3)	1.23	D3
	KR(4)	1.29	D4
	KR(5)	1.31	D5
	KR(6)	1.33	D6
20	SAT(1)	32900.	ALLOWABLE MATERIAL STRESS; D1
	SAT(2)	38100.	D2
	SAT(3)	44500.	D3
	SAT(4)	51750.	D4
	SAT(5)	54250.	D5
	SAT(6)	61000.	D6

DEFINITIONS OF CODED NOTES FROM ABOVE:

A1 NAVAL PROFILE - FULL POWER, 5 PERCENT MAX

A2 OTHER - MAXIMUM LOAD, CONTINUOUS

B1 POWER SOURCE - TURBINE OR MOTOR

B2 POWER SOURCE - MULTICYLINDER I. C. ENGINE

C1 FIRST REDUCTION STAGE

C2 SECOND REDUCTION STAGE

C3 THIRD REDUCTION STAGE

D1 HARDNESS RANGE: 160 - 200 BHN

D2 HARDNESS RANGE: 200 - 240 BHN

D3 HARDNESS RANGE: 240 - 300 BHN

D4 HARDNESS RANGE: 300 - 360 BHN

D5 HARDNESS RANGE: 360 - 400 BHN
D6 HARDNESS RANGE: 400 - 640 BHN

E1 SINGLE POWER PATH
E2 DOUBLE POWER PATH

DO YOU DESIRE TO CHANGE ANY OF THE ABOVE VALUES? (Y OR N):

Y

TO CHANGE A CONSTANT ABOVE, ENTER THE ID NUMBER WHEN PROMPTED.
USE ID NUMBER 99 WHEN NO FURTHER CHANGES ARE TO BE MADE.
NOTE: WHEN ASKED FOR THE NEW VALUE OF THE CONSTANT, ENTERING
A ZERO WILL CAUSE THE ORIGINAL VALUE TO REMAIN UNCHANGED.
THIS IS USEFUL WHEN A CONSTANT HAS MULTIPLE VALUES, BUT NOT
ALL OF THEM ARE TO BE CHANGED.

** ENTER THE CONSTANT ID NUMBER (1-20, 99 TO STOP):

?
16

** ENTER KO(1):

?
1.14

** ENTER KO(2):

?
0

** ENTER THE CONSTANT ID NUMBER (1-20, 99 TO STOP):

?
99

THE FOLLOWING IS A LISTING OF THE PRE-PROGRAMMED CONSTANTS USED IN THE AGMA FORMULATIONS WITH APPROPRIATE NOTES ON THEIR APPLICATION. NOTE: THOSE STARTING WITH A 'C' ARE DURABILITY CONSTANTS AND THOSE WITH A 'K' ARE STRENGTH CONSTANTS. SERVICE FACTOR APPLIES TO BOTH FORMULATIONS.

ID	CONST	VALUE(S)	NOTES
1	SP (1,1)	1.00	SERVICE FACTOR; A1,B1
	SP (1,2)	1.50	A1,B2
	SP (2,1)	1.50	A2,B1
	SP (2,2)	1.75	A2,B2
2	CV (1)	1.00	DYNAMIC FACTOR; C1
	CV (2)	0.83	C2
	CV (3)	0.69	C3
3	CS	1.00	SIZE FACTOR
4	CM (1)	1.25	LOAD DISTRIBUTION FACTOR; A1
	CM (2)	1.35	A2
5	CP	1.00	SURFACE CONDITION FACTOR
6	CO (1)	1.15	OVERLOAD FACTOR; A1
	CO (2)	1.14	A2
7	CP	2300.0	ELASTIC PROPERTIES FACTOR
8	CL (1)	0.80	LIFE FACTOR; A1
	CL (2)	0.68	A2
9	CH	1.00	HARDNESS RATIO FACTOR
10	CT	1.00	TEMPERATURE FACTOR
11	CR (1)	1.16	RELIABILITY FACTOR; D1

12	CR(2) CR(3) CR(4) CR(5) CR(6)	1.19 1.22 1.27 1.31 1.35		D2 D3 D4 D5 D6
	SAC(1) SAC(2) SAC(3) SAC(4) SAC(5) SAC(6)	95000. 108000. 125000. 146000. 165000. 182000.	ALLOWABLE CONTACT STRESS;	D1 D2 D3 D4 D5 D6
13	KV	0.70	DYNAMIC FACTOR	
14	KS	1.00	SIZE FACTOR	
15	KM	1.10	LOAD DISTRIBUTION FACTOR	
16	KO(1) KO(2)	1.14 1.28	OVERLOAD FACTOR; E1 E2	
17	KL(1) KL(2)	0.80 0.68	LIFE FACTOR; A1 A2	
18	KT	1.00	TEMPERATURE FACTOR	
19	KR(1) KR(2) KR(3) KR(4) KR(5) KR(6)	1.16 1.18 1.23 1.29 1.31 1.33	RELIABILITY FACTOR; D1 D2 D3 D4 D5 D6	
20	SAT(1) SAT(2)	32900. 38100.	ALLOWABLE MATERIAL STRESS;	D1 D2

D3
D4
D5
D6

SAT(3) 44500.
SAT(4) 51750.
SAT(5) 54250.
SAT(6) 61000.

DEFINITIONS OF CODED NOTES FROM ABOVE:

A1 NAVAL PROFILE - FULL POWER, 5 PERCENT MAX

A2 OTHER - MAXIMUM LOAD, CONTINUOUS

B1 POWER SOURCE - TURBINE OR MOTOR

B2 POWER SOURCE - MULTICYLINDER I. C. ENGINE

C1 FIRST REDUCTION STAGE

C2 SECOND REDUCTION STAGE

C3 THIRD REDUCTION STAGE

D1 HARDNESS RANGE: 160 - 200 BHN

D2 HARDNESS RANGE: 200 - 240 BHN

D3 HARDNESS RANGE: 240 - 300 BHN

D4 HARDNESS RANGE: 300 - 360 BHN

D5 HARDNESS RANGE: 360 - 400 BHN

D6 HARDNESS RANGE: 400 - 640 BHN

E1 SINGLE POWER PATH

E2 DOUBLE POWER PATH

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 1
IN POWER TRAIN 1.

ENTER DIAMETERS OF PINION AND GEAR, INCHES (DP,DG):

?

9.31,29.35

** ENTER FACEWIDTH OF GEAR PAIR, INCHES:
?
16.62

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 2
IN POWER TRAIN 1.

** ENTER DIAMETERS OF PINION AND GEAR, INCHES (DP,DG):
?
14.24, 105.25

** ENTER FACEWIDTH OF GEAR PAIR, INCHES:
?
25.97

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 1
IN POWER TRAIN 2.

** ENTER DIAMETERS OF PINION AND GEAR, INCHES (DP,DG):
?
10.01, 26.98

** ENTER FACEWIDTH OF GEAR PAIR, INCHES:
?
18.02

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 2
IN POWER TRAIN 2.

** ENTER ONLY DIAMETER OF PINION, INCHES (DP):
?
14.24

POWER SOURCE 1: TURBINE OR MOTOR
 INPUT POWER (HP): 21250. INPUT SPEED (RPM): 6990.
 ARRANGEMENT: PARALLEL AXIS, 2 INPUT(S), 2 POWER PATH(S), 2 REDUCTION(S)
 OUTPUT POWER (HP): 42500.0 OUTPUT SPEED (RPM): 300.
 RATIO: 23.301 OUTPUT TORQUE (K IN-LB): 8925.0
 SOURCE CENTER DISTANCE (IN): MIN= 38.8 MAX= 49.4

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:
 WEIGHT (LB): 93200. SPECIFIC WEIGHT (LB/HP): 2.19
 LENGTH (IN): 99 WIDTH (IN): 135 HEIGHT (IN): 144

	REDUCTION 1		REDUCTION 2	
	PINION	GEAR	PINION	GEAR
POWER SPLIT	HP	21250.	10625.	10625.
SPEED	RPM	6990.	2217.	21250.
NUMBER OF TEETH		42	132	50
NORMAL DIAMETRAL PITCH		5.493		4.273
TRANS. DIAMETRAL PITCH		4.500		3.500
NORMAL PRESSURE ANGLE		20.0		20.0
TRANS. PRESSURE ANGLE		24.0		24.0
HELIX ANGLE		35.0		35.0
GEAR RATIO		3.153		7.391
PITCH DIAMETER	IN	9.31	29.35	14.24
EFFECTIVE FACEWIDTH	IN	16.62		25.97
F/DP		1.79		1.82
CENTER DISTANCE	IN	19.33		59.74
PITCHLINE VELOCITY	FPM	17037.		8266.
TANGENTIAL LOAD	LB	41160.	20580.	42417.
TOOTH LOAD/IN	LB/IN	2477.	1238.	1633.
UNIT LOAD	PSI	13605.	6802.	6979.

MESH FREQUENCY HZ | 4893. | 1848. |
 K FACTOR (COMPUTED) | 350. | 175. | 130. | 260. |
 CONTACT STRESS PSI | 89094. | 59079. |
 BENDING STRESS PSI | 37381. | 17155. | 18416. | 33846. |
 TORQUE K IN-LB | 191.6 | 302.0 | 302.0 | 4464.4 |
 HARDNESS RANGE BHN | 200-240 | 160-200 | 200-240 | 160-200 |

*** 此處所列之數據均係根據本廠之標準及經驗所得，其準確度與實際情況之符合程度，尚須視具體條件而定。 ***

POWER SOURCE 2: TURBINE OR MOTOR
 INPUT POWER (HP): 21250. INPUT SPEED (RPM): 5980.

ARRANGEMENT: PARALLEL AXIS, 2 INPUT(S), 2 POWER PATH(S), 2 REDUCTION(S)
 OUTPUT POWER (HP): 42500.0 OUTPUT SPEED (RPM): 300.
 RATIO: 19.921 OUTPUT TORQUE (K IN-LB): 8925.0

SOURCE CENTER DISTANCE (IN): MIN= 38.8 MAX= 49.4

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 93200. SPECIFIC WEIGHT (LB/HP): 2.19
 LENGTH (IN): 99 WIDTH (IN): 135 HEIGHT (IN): 144

		REDUCTION 1		REDUCTION 2	
		PINION	GEAR	PINION	GEAR
POWER SPLIT	HP	21250.	10625.	10625.	21250.
SPEED	RPM	5980.	2219.	2219.	300.
NUMBER OF TEETH		45	121	50	368
NORMAL DIAMETRAL PITCH			5.493		4.273
TRANS. DIAMETRAL PITCH			4.500		3.500
NORMAL PRESSURE ANGLE			20.0		20.0
TRANS. PRESSURE ANGLE			24.0		24.0
HELIX ANGLE			35.0		35.0
GEAR RATIO			2.695		7.391
PITCH DIAMETER	IN	10.01	26.98	14.24	105.25

EFFECTIVE FACEWIDTH IN	18.02	25.97
F/DP	1.80	1.82
CENTER DISTANCE IN	18.49	59.74
PITCHLINE VELOCITY FPM	15671.	8271.
TANGENTIAL LOAD LB	44747. 22374.	42390. 84781.
TOOTH LOAD/IN LB/IN	2483. 1242.	1632. 3265.
UNIT LOAD PSI	113641. 6821.	6974. 13949.
MESH FREQUENCY HZ	4485.	1849.
K FACTOR (COMPUTED)	340. 170.	130. 260.
CONTACT STRESS PSI	87718.	59060.
BENDING STRESS PSI	37061. 17230.	18404. 33824.
TORQUE K IN-LB	224.0 301.8	301.8 4461.6
HARDNESS RANGE BHN	200-240 160-200	200-240 160-200

此表係根據美國機械工程師學會齒輪設計標準及美國齒輪製造商協會標準編制而成

Results from Design Session

** ENTER SEED FOR RANDOM NUMBER GENERATOR (X.XX):

?

0.76

POWER SOURCE 1: TURBINE OR MOTOR

INPUT POWER (HP): 21250. INPUT SPEED (RPM): 6990.

ARRANGEMENT: PARALLEL AXIS, 2 INPUT(S), 2 POWER PATH(S), 2 REDUCTION(S)

OUTPUT POWER (HP): 42500.0 OUTPUT SPEED (RPM): 300.

RATIO: 23.300 OUTPUT TORQUE (K IN-LB): 8925.0

SOURCE CENTER DISTANCE (IN): MIN= 57.7 MAX= 57.7

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 125000. SPECIFIC WEIGHT (LB/HP): 2.94

LENGTH (IN): 143 WIDTH (IN): 165 HEIGHT (IN): 177

80

	REDUCTION 1		REDUCTION 2	
	PINION	GEAR	PINION	GEAR
POWER SPLIT	HP	21250.1	10625.1	10625.1
SPEED	RPM	6990.1	2411.1	2411.1
NUMBER OF TEETH		72	209	56
NORMAL DIAMETRAL PITCH		5.493		4.273
TRANS. DIAMETRAL PITCH		4.500		3.500
NORMAL PRESSURE ANGLE		20.0		20.0
TRANS. PRESSURE ANGLE		24.0		24.0
HELIX ANGLE		35.0		35.0
GEAR RATIO		2.899		8.037
PITCH DIAMETER	IN	15.99	46.35	16.06
EFFECTIVE FACEWIDTH	IN	30.91		32.19

TRANS. DIAMETRAL PITCH	4.500	3.500
NORMAL PRESSURE ANGLE	20.0	20.0
TRANS. PRESSURE ANGLE	24.0	24.0
HELIX ANGLE	35.0	35.0
GEAR RATIO	3.231	6.169
PITCH DIAMETER IN	16.97	20.92
EFFECTIVE FACEWIDTH IN	27.14	27.14
F/DP	1.60	1.30
CENTER DISTANCE IN	35.89	75.00
PITCHLINE VELOCITY FPM	26562.	10137.
TANGENTIAL LOAD LB	26400.	34587.
TOOTH LOAD/IN LB/IN	973.	1274.
UNIT LOAD PSI	5343.	5444.
MESH FREQUENCY HZ	7575.	2252.
K FACTOR (COMPUTED)	75.	71.
CONTACT STRESS PSI	40667.	43269.
BENDING STRESS PSI	13453.	13701.
TORQUE K IN-LB	224.0	361.8
HARDNESS RANGE BHN	200-240	200-240

此表係根據 ANSI B2.1-1968 齒輪設計標準之規定而編製，其計算結果僅供參考，實際設計時應參照最新之設計標準及製造規範。

REGAD
REDUCTION GEAR ANALYSIS AND DESIGN

n

83

22

22

CHOOSE OPERATIONAL PROFILE CODE BELOW:		
OPERATIONAL MODE	SERVICE PROFILE	CODE
FULL POWER	5 PERCENT MAX	1
MAXIMUM LOAD	CONTINUOUS	2

** ENTER OPERATIONAL PROFILE CODE:

? 1

** ENTER NUMBER OF REDUCTIONS (1, 2, OR 3):

? 2

CHOOSE DESIRED HELIX TYPE BELOW:

TYPE	ANGLE	CODE
SINGLE	15-25	1
DOUBLE	25-50	2

** ENTER HELIX CODE:

? 2

WILL ANY POWER SOURCE BE A MULTICYLINDER I. C. ENGINE? (Y OR N):

n

** ENTER POWER AND SPEED OF THE POWER SOURCE (HP, RPM):

? 8250, 3600

** ENTER OUTPUT SHAFT/PROPELLER SPEED (RPM):

? 155

WHICH DIAMETRAL PITCH WILL YOU SPECIFY?
(1=TRANSVERSE, 2=NORMAL):

? 2

WHICH PRESSURE ANGLE WILL YOU SPECIFY?
(1=TRANSVERSE, 2=NORMAL):

? 2

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 1 :

** ENTER HELIX ANGLE (DEGREES):

? 25

** ENTER NORMAL DIAMETRAL PITCH:

? 8

** ENTER NORMAL PRESSURE ANGLE (DEGREES):

? 20

** ENTER EPICYCLIC CODE (1=PLANETARY, 2=STAR):

? 1

** ENTER NUMBER OF PLANET GEARS (3 TO 5):

? 4

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN CODE

160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4
360 - 400	5

400 - 640 6

** ENTER HARDNESS CODES FOR SUN/PLANETS AND RING (HCSUN,HCRING) :
?
4,2

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 2 :

** ENTER HELIX ANGLE (DEGREES) :
?
25

** ENTER NORMAL DIAMETRAL PITCH:
?
6

** ENTER NORMAL PRESSURE ANGLE (DEGREES) :
?
20

** ENTER EPICYCLIC CODE (1=PLANETARY, 2=STAR) :
?
1

** ENTER NUMBER OF PLANET GEARS (3 TO 5) :
?
5

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN	CODE
160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4

360 - 400 5
400 - 640 6

** ENTER HARDNESS CODES FOR SUN/PLANETS AND RING (HCSUN,HCRING):

?
4,2

TO MAKE CORRECTIONS TO DATA JUST ENTERED, THE PROGRAM MUST BE
ABORTED AND RE-STARTED. DO YOU WISH TO ABORT THIS RUN? (Y OR N):

n

DO YOU DESIRE A LISTING OF THE PRE-PROGRAMMED CONSTANTS USED
IN THE AGMA FORMULATIONS? (Y OR N):

n

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 1.

** ENTER DIAMETERS, IN INCHES, OF SUN, PLANET, AND RING GEARS
(DS, DPLN, DR):

?
12.55, 18.76, 50.34

** ENTER FACEWIDTH OF GEARS, IN INCHES:

?
19.13

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 2.

** ENTER DIAMETERS, IN INCHES, OF SUN, PLANET, AND RING GEARS
(DS, DPLN, DR):

?
22.99, 30.16, 83.67

ENTER FACEWIDTH OF GEARS, IN INCHES:
?
27.57

POWER SOURCE: TURBINE OR MOTOR
INPUT POWER (HP): 8250. INPUT SPEED (RPM): 3600.
ARRANGEMENT: EPICYCLIC, 2 REDUCTION(S)
OUTPUT POWER (HP): 8250. OUTPUT SPEED (RPM): 155.
RATIO: 23.226 OUTPUT TORQUE (K IN-LB): 3356.5

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:
WEIGHT (LB): 65400. SPECIFIC WEIGHT (LB/HP): 7.93
LENGTH (IN): 133 WIDTH (IN): 109 HEIGHT (IN): 100

GEAR ARRANGEMENT	REDUCTION 1		
	SUN	PLANETS	RING-CAGE
	PLANETARY		
NUMBER OF PLANETS		4	
POWER SPLIT	8250.	2063.	8250.
SPEED	3600.	1928.	718.
NUMBER OF TEETH	91	136	365
NORMAL DIAMETRAL PITCH		8.000	
TRANS. DIAMETRAL PITCH		7.250	
NORMAL PRESSURE ANGLE		20.0	
TRANS. PRESSURE ANGLE		21.9	

HELIX ANGLE		25.0	
GEAR RATIO		5.011	
PITCH DIAMETER IN		18.76	
EFFECTIVE FACEWIDTH IN		19.13	50.34
F/DP		1.52	
CENTER DISTANCE IN		15.65	
PITCHLINE VELOCITY FPM		11828.	
TANGENTIAL LOAD LB		23017.	
TOOTH LOAD/IN LB/IN		1203.	
UNIT LOAD PSI		9626.	
MESH FREQUENCY HZ		1928.	2874.
K FACTOR (COMPUTED)		160.	
CONTACT STRESS PSI		62354.	
BENDING STRESS PSI		26284.	
TORQUE K IN-LB		144.4	215.9 723.5
HARDNESS RANGE BHN		300 - 360	300 - 360 200 - 240

此表係根據美國機械工程師學會所編之《機械設計》一書中之表 13-1 及表 13-2 整理而成，其單位與原表相同，但表中之數值係按四舍五入法修約至三位有效數字，故與原表略有出入，特此聲明。

REDUCTION 2			
SUN		PLANETS	RING-CAGE
-----		-----	-----
PLANETARY			
5			
8250.		1650.	8250.
718.		430.	155.
125		164	455
		6.000	
		5.438	
		20.0	
		21.9	
		25.0	
		4.639	
22.99		30.16	83.67
		27.57	

GEAR ARRANGEMENT

NUMBER OF PLANETS

POWER SPLIT HP

SPEED RPM

NUMBER OF TEETH

NORMAL DIAMETRAL PITCH

TRANS. DIAMETRAL PITCH

NORMAL PRESSURE ANGLE

TRANS. PRESSURE ANGLE

HELIX ANGLE

GEAR RATIO

PITCH DIAMETER IN

EFFECTIVE FACEWIDTH IN

Results from Design Session

ENTER SEED FOR RANDOM NUMBER GENERATOR (X.YX):

?

0.076

POWER SOURCE: TURBINE OR MOTOR

INPUT POWER (HP): 8250.

INPUT SPEED (RPM): 3600.

ARRANGEMENT: EPICYCLIC, 2 REDUCTION(S)

OUTPUT POWER (HP): 8250. OUTPUT SPEED (RPM): 155.

RATIO: 23.093 OUTPUT TORQUE (K IN-LB): 3334.1

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 66300. SPECIFIC WEIGHT (LB/HP): 8.04

LENGTH (IN): 154 WIDTH (IN): 92 HEIGHT (IN): 85

GEAR ARRANGEMENT	REDUCTION 1		
	SUN	PLANETS	RING-CAGE
NUMBER OF PLANETS		PLANETARY	
POWER SPLIT		4	
SPEED	8250.	2063.	8250.
NUMBER OF TEETH	3600.	1786.	684.
NORMAL DIAMETRAL PITCH	98	160	418
TRANS. DIAMETRAL PITCH		8.000	
NORMAL PRESSURE ANGLE		7.250	
		20.0	

APPENDIX D

PROGRAM LISTING

Module One

```

C *****
C      REGAD
C      REDUCTION GEAR ANALYSIS AND DESIGN
C *****
C      CODED BY:  LT J.L. PAQUETTE, USN      JAN 1982
C      NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
C
C      THIS IS THE MAIN PROGRAM FOR THE REGAD PACKAGE OF SUBROUTINES
C      FOR THE PRELIMINARY DESIGN OR ANALYSIS OF MULTIREDUCTION, PARALLEL
C      AXIS AND EPICYCLIC GEARING FOR MARINE APPLICATIONS.  A BRIEF DE-
C      SCRIPTON AND LISTING OF CAPABILITIES CAN BE OBTAINED AS AN OPTION
C      DURING THE EXECUTION OF THE PROGRAM.  THIS PACKAGE HAS BEEN DE-
C      SIGNED IN MODULAR FORM FOR EASE OF MAINTENANCE AND MODIFICATION.
C      WITH THE EXCEPTION OF FREE FORMATTED INPUT, EVERY ATTEMPT WAS MADE
C      TO ENSURE PORTABILITY BY USING ANSI FORTRAN (FORTRAN IV).
C
C      REAL MGOP,MGP,MGOE,MGE,MFP,MFE,KFCTRP,KFCTRE
C
C      SEVEN COMMON BLOCKS ARE USED FOR DATA TRANSFER WITHIN THE PRO-
C      GRAM.  TWO CONTAIN THE PRE-PROGRAMMED AGMA CONSTANTS.  A LISTING
C      OF THESE CONSTANTS AND THEIR VALUES CAN BE OBTAINED AND SELECTIVE-
C      LY CHANGED AS AN OPTION DURING THE EXECUTION OF THE PROGRAM.
C
C *****
C      1MOD0010
C      1MOD0020
C      1MOD0030
C      1MOD0040
C      1MOD0050
C      1MOD0060
C      1MOD0070
C      1MOD0080
C      1MOD0090
C      1MOD0100
C      1MOD0110
C      1MOD0120
C      1MOD0130
C      1MOD0140
C      1MOD0150
C      1MOD0160
C      1MOD0170
C      1MOD0180
C      1MOD0190
C      1MOD0200
C      1MOD0210
C      1MOD0220
C      1MOD0230

```



```

COMMON /AGHAB/ SFB(2,2), AKV, AKS, AKM, AKO(2), SAT(6), AKL(2), AKR(6), AK 1MOD0240
1T 1MOD0250
1MOD0260
1MOD0270
1MOD0280
1MOD0290
1MOD0300
1MOD0310
1MOD0320
1MOD0330
1MOD0340
1MOD0350
1MOD0360
1MOD0370
COMMON /AGMAH/ SFH(2,2), CV(3), CS, CH(2), CF, CO(2), SAC(6), CP, CL(2), CH 1MOD0380
1, CT, CR(6) 1MOD0390
1MOD0400
1MOD0410
1MOD0420
1MOD0430
1MOD0440
1MOD0450
1MOD0460
1MOD0470
1MOD0480
1MOD0490
1MOD0500
1MOD0510
1MOD0520
1MOD0530
1MOD0540
COMMON /DESDAT/ PWRIN(2), RPMIN(2), RPMOUT, DHELIX(3), HELIX(3), PD(3), 1MOD0550
1PND(3), DPHI(3), PHI(3), DPHIN(3), PHIN(3), NDIFP, IARR, IEPIC(3), IHARD(3) 1MOD0560
2, 2), IOPRO, NPWRIN, IPWRSR(2), NRRED, NPATH, NPLNT(3), NHELX 1MOD0570
1MOD0580
1MOD0590
COMMON BLOCK DESDAT (DESIGN PARAMETERS, INPUT)

```

AD-A117 828

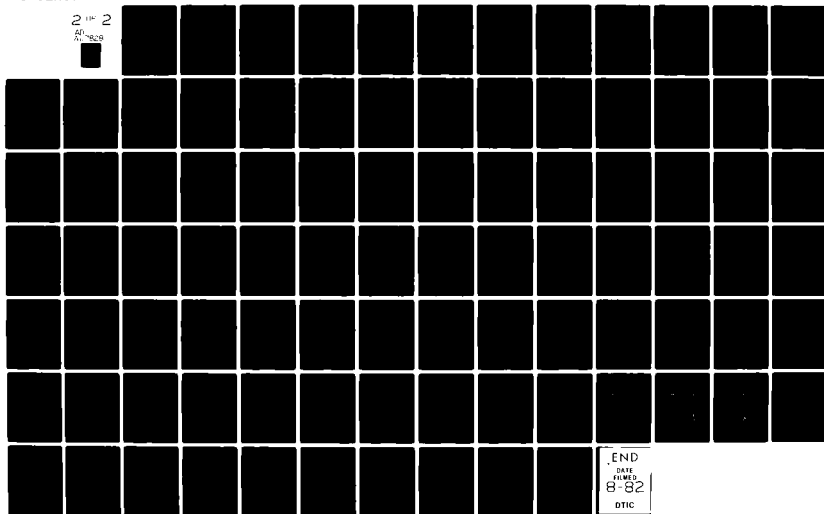
NAVAL POSTGRADUATE SCHOOL MONTEREY CA
AN INTERACTIVE COMPUTER PROGRAM FOR THE PRELIMINARY DESIGN AND --ETC(U)
MAR 82 J L PAQUETTE

F/6 9/2

UNCLASSIFIED

NL

2 OF 2
AD-828



END
DATE
FILMED
8-82
DTIC


```

C      GEOMJG: R*4 ARRAY (3,2): STRENGTH GEOMETRY FACTOR (GEAR) 1MOD0960
C      GEOMJP: R*4 ARRAY (3,2): STRENGTH GEOMETRY FACTOR (PINION) 1MOD0970
C      MGOP : R*4 ARRAY (2): OVERALL REDUCTION RATIO 1MOD0980
C      MGP : R*4 ARRAY (3,2): STAGE REDUCTION RATIO 1MOD0990
C      NG : I*4 ARRAY (3,2): NUMBER OF TEETH, GEAR 1MOD1000
C      NP : I*4 ARRAY (3,2): NUMBER OF TEETH, PINION 1MOD1010
C      PWRPAC: R*4 ARRAY (2,3): STAGE POWER SPLIT FACTOR 1MOD1020
C      PWRP : R*4 ARRAY (6,2): STAGE POWER SPLIT PER GEAR (HP) 1MOD1030
C      RPMP : R*4 ARRAY (6,2): STAGE PINION AND GEAR SPEED (RPM) 1MOD1040
C      1MOD1050
C      COMMON /RESPRL/ PLVP (3,2), FBYDP (3,2), CDP (3,2), WTP (6,2), TLPIP (6,2), 1MOD1060
1UNTLPD (6,2), MFP (3,2), KFCTRP (6,2), SIGHP (3,2), SIGBP (6,2), TORQP (6,2), 1MOD1070
2PDIAMP (6,2), SCDMIN, SCDMAX, SHP, WGHTP, SPCWTP, MTHP (6,2), ISIZEP (3) 1MOD1080
1MOD1090
C      *** COMMON BLOCK RESPRL (PARALLEL AXIS PARAMETERS, RESULTS) 1MOD1100
C      CDP : R*4 ARRAY (3,2): CENTER DISTANCE (THEORETICAL) (IN) 1MOD1110
C      FBYDP : R*4 ARRAY (3,2): F/D RATIO (FACEWIDTH/PINION DIAMETER) 1MOD1120
C      ISIZEP: I*4 ARRAY (3): LENGTH, WIDTH, HEIGHT ESTIMATES (IN) 1MOD1130
C      KFCTRP: R*4 ARRAY (6,2): COMPUTED K-FACTOR 1MOD1140
C      MFP : R*4 ARRAY (3,2): MESH FREQUENCY (HZ) 1MOD1150
C      MTHP : I*4 ARRAY (6,2): TOOTH NUMBERS 1MOD1160
C      PDIAMP: R*4 ARRAY (6,2): PITCH DIAMETERS (IN) 1MOD1170
C      PLVP : R*4 ARRAY (3,2): PITCH LINE VELOCITY (FPM) 1MOD1180
C      SCDMAX: R*4; MAXIMUM SOURCE CENTER DISTANCE (IN) 1MOD1190
C      SCDMIN: R*4; MINIMUM SOURCE CENTER DISTANCE (IN) 1MOD1200
C      SHP : R*4; SHAFT HORSEPOWER, OUTPUT (HP) 1MOD1210
C      SIGBP : R*4 ARRAY (6,2): BENDING STRESS (PSI) 1MOD1220
C      SIGHP : R*4 ARRAY (3,2): CONTACT STRESS (PSI) 1MOD1230
C      SPCWTP: R*4; SPECIFIC WEIGHT (LB/HP) 1MOD1240
C      TLPIP : R*4 ARRAY (6,2): TOOTH LOAD PER INCH (LB/IN) 1MOD1250
C      TORQP : R*4 ARRAY (6,2): TORQUE (K IN-LB) 1MOD1260
C      UNTLPD: R*4 ARRAY (6,2): UNIT LOAD (PSI) 1MOD1270
C      WGHTP : R*4; GEAR SET WEIGHT ESTIMATE (LB) 1MOD1280
C      WTP : R*4 ARRAY (6,2): TANGENTIAL TOOTH LOAD (LB) 1MOD1290
C      1MOD1300
C      COMMON /DESEPC/ MGOE, MGE (3), RPMI (3), RPMPL (3), RPMO (3), PWRE (3), DS (3) 1MOD1310

```

```

C
C
C *** COMMON BLOCK DESEPC (EPICYCLIC DESIGN PARAMETERS)
C DPLN : R*4 ARRAY (3); DIAMETER OF PLANET GEARS (IN) 1MOD1320
C DR : R*4 ARRAY (3); DIAMETER OF RING GEAR (IN) 1MOD1330
C DS : R*4 ARRAY (3); DIAMETER OF SUN GEAR (IN) 1MOD1340
C FACEE : R*4 ARRAY (3); FACEWIDTH (IN) 1MOD1350
C GI : R*4 ARRAY (3); DURABILITY GEOMETRY FACTOR (SUN/PLANETS) 1MOD1360
C GJS : R*4 ARRAY (3); STRENGTH GEOMETRY FACTOR (SUN) 1MOD1370
C GJPL : R*4 ARRAY (3); STRENGTH GEOMETRY FACTOR (PLANET) 1MOD1380
C MGE : R*4 ARRAY (3); STAGE REDUCTION RATIO 1MOD1390
C MGOE : R*4; OVERALL REDUCTION RATIO 1MOD1400
C NPLN : I*4 ARRAY (3); NUMBER OF TEETH, PLANET 1MOD1410
C NR : I*4 ARRAY (3); NUMBER OF TEETH, RING 1MOD1420
C NS : I*4 ARRAY (3); NUMBER OF TEETH, SUN 1MOD1430
C PWRE : R*4 ARRAY (3); STAGE POWER SPLIT PER GEAR PAIR (HP) 1MOD1440
C RPMI : R*4 ARRAY (3); STAGE INPUT SPEED (RPM) 1MOD1450
C RPMO : R*4 ARRAY (3); STAGE OUTPUT SPEED (RPM) 1MOD1460
C RPMPL : R*4 ARRAY (3); PLANET SPEED (RPM) 1MOD1470
C
C COMMON /RESEPC/ PLVE(3),FBYDE(3),CDE(3),WTE(3),TLPIE(3),UNTLDE(3),
1MFE(3,3),KFCTRE(3),SIGBE(3),TORQE(3,3),RPME(3,3),PDIAHE(3,
2,3),WGHTe,SPCWTE,MTHE(3,3),ISIZEE(3)
C
C *** COMMON BLOCK RESEPC (EPICYCLIC PARAMETERS, RESULTS)
C CDE : R*4 ARRAY (3); CENTER DISTANCE (THEORETICAL) (IN) 1MOD1560
C FBYDE : R*4 ARRAY (3); F/D RATIO (FACEWIDTH/SUN DIAMETER) 1MOD1570
C ISIZEE : I*4 ARRAY (3); LENGTH, WIDTH, HEIGHT ESTIMATES (IN) 1MOD1580
C KFCTRE : R*4 ARRAY (3); COMPUTED K-FACTOR 1MOD1590
C MFE : R*4 ARRAY (3,3); MESH FREQUENCY (HZ) 1MOD1600
C MTHE : I*4 ARRAY (3,3); TOOTH NUMBERS 1MOD1610
C PDIAHE : R*4 ARRAY (3,3); PITCH DIAMETERS (IN) 1MOD1620
C PLVE : R*4 ARRAY (3); PITCH LINE VELOCITY (FPM) 1MOD1630
C RPME : R*4 ARRAY (3,3); GEAR SPEEDS (RPM) 1MOD1640
C SIGBE : R*4 ARRAY (3); BENDING STRESS (PSI) 1MOD1650
C SIGHE : R*4 ARRAY (3); CONTACT STRESS (PSI) 1MOD1660
C

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C	SPCWTE: R*4;		SPECIFIC WEIGHT (LB/HP)	1MOD1680
C	TLPIE : R*4 ARRAY (3);		TOOTH LOAD PER IN (LB/IN)	1MOD1690
C	TORQE : R*4 ARRAY (3,3);		TORGUE (K IN-LB)	1MOD1700
C	UNTLDE: R*4 ARRAY (3);		UNIT LOAD (PSI)	1MOD1710
C	WGHTS : R*4;		GEAR SET WEIGHT ESTIMATE (LB)	1MOD1720
C	WTE : R*4 ARRAY (3);		TANGENTIAL TOOTH LOAD (LB)	1MOD1730
C				1MOD1740
C				1MOD1750
C	EXECUTE REGAD			1MOD1760
C				1MOD1770
	DATA YES/1HY/			1MOD1780
	WRITE (6,90)			1MOD1790
	WRITE (6,100)			1MOD1800
	READ (5,130) REP			1MOD1810
	IF (REP.EQ.YES) CALL DSCRPT			1MOD1820
	WRITE (6,120)			1MOD1830
				1MOD1840
C	DESIGN / ANALYSIS OPTION SELECTION			1MOD1850
C				1MOD1860
	WRITE (6,70)			1MOD1870
	READ (5,*) ICODE			1MOD1880
	IF (ICODE.LT.1) ICODE=1			1MOD1890
	IF (ICODE.GT.2) ICODE=2			1MOD1900
C	CONFIGURATION SELECTION			1MOD1910
C				1MOD1920
C				1MOD1930
	WRITE (6,80)			1MOD1940
	READ (5,*) IARR			1MOD1950
	IF (IARR.LT.1) IARR=1			1MOD1960
	IF (IARR.GT.2) IARR=2			1MOD1970
	CALL INPUT			1MOD1980
	WRITE (6,110)			1MOD1990
	READ (5,130) REP			1MOD2000
	IF (REP.EQ.YES) CALL AGMA			1MOD2010
	L=ICODE+(IARR-1)*2			1MOD2020
	GO TO (10,20,40,50), L			1MOD2030

C	PARALLEL AXIS REDUCTION GEARS	1MOD2040
C		1MOD2050
C		1MOD2060
C		1MOD2070
C	DESIGN	1MOD2080
C		1MOD2090
10	CALL PRLDES	1MOD2100
	GO TO 30	1MOD2110
C		1MOD2120
C	ANALYSIS	1MOD2130
C		1MOD2140
20	CALL PRLANL	1MOD2150
C		1MOD2160
C	COMPLETE PARALLEL AXIS COMPUTATIONS	1MOD2170
C		1MOD2180
30	CALL PRLRES	1MOD2190
	CALL PRLSIZ	1MOD2200
	CALL PRLOUT	1MOD2210
	STOP	1MOD2220
C		1MOD2230
C	EPICYCLIC REDUCTION GEARS	1MOD2240
C		1MOD2250
C		1MOD2260
C	DESIGN	1MOD2270
C		1MOD2280
40	CALL EPCDES	1MOD2290
	GO TO 60	1MOD2300
C		1MOD2310
C	ANALYSIS	1MOD2320
C		1MOD2330
50	CALL EPCANL	1MOD2340
C		1MOD2350
C	COMPLETE EPICYCLIC COMPUTATIONS	1MOD2360
C		1MOD2370
60	CALL EPCRES	1MOD2380
	CALL EPCSIZ	1MOD2390


```

1UNTLDP(6,2),MPP(3,2),KFCTRP(6,2),SIGHP(3,2),SIGBP(6,2),TORQP(6,2),1MOD2760
2PDAMP(6,2),SCDHIN,SCDMAX,SHP,WGHTP,SPCWTP,TRQOUT,MTHP(6,2),ISIZEP1MOD2770
3(3)1MOD2780
COMMON /DESEPC/ MGOE,MGE(3),RPNI(3),RPNPL(3),RPHO(3),PWRP(3),DS(3)1MOD2790
1,DPLN(3),DR(3),FACEE(3),GI(3),GJS(3),GJPL(3),NS(3),NPLN(3),NR(3)1MOD2800
COMMON /RESEPC/ PLVE(3),FVDE(3),CDE(3),WTE(3),TLPIE(3),UNTLDE(3),1MOD2810
1MPE(3,3),KFCTRE(3),SIGHE(3),SIGBE(3),TORQE(3,3),RPME(3,3),PDIAME(3)1MOD2820
2,3),WGHTP,SPCWTE,MTHP(3,3),ISIZEE(3)1MOD2830
DATA SPH/1.,1.5,1.5,1.75,CV/1.,.83,.69,CS/1.,CM/1.25,1.35,CF/11MOD2840
1.,CO/1.15,1.14,SAC/95000.,108000.,125000.,146000.,165000.,1820001MOD2850
2.,CR/2300.,CL/.80,.68,CH/1.,CT/1.,CR/1.160,1.185,1.224,1.273,1MOD2860
31.312,1.350/1MOD2870
DATA SPB/1.,1.5,1.5,1.75,AKV/.7,AKS/1.,AKM/1.1,AKO/1.21,1.28,1MOD2880
1SAT/32900.,38100.,44500.,51750.,54250.,61000.,AKL/.80,.68,AKR/1.1MOD2890
2160,1.180,1.230,1.288,1.313,1.330,AKT/1./1MOD2900
DATA IPWRSR/1,1,PWRPAC/1.,1.,1.,.5,.5,.25,NGOP/2*1.,MGOE/1.,PW1MOD2910
1RIN/2*0.,RPMIN/2*0.0/1MOD2920
END1MOD2930
C*****1MOD2940
C*****1MOD2950
C*****1MOD2960
SUBROUTINE DSCRIPT1MOD2970
1MOD2980
1MOD2990
1MOD3000
1MOD3010
1MOD3020
1MOD3030
1MOD3040
1MOD3050
1MOD3060
1MOD3070
1MOD3080
1MOD3090
1MOD3100
1MOD3110

```

CODED BY: LT J.L. PAQUETTE, USN JAN 1982
 NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940

SUBPROGRAM TO PROVIDE A GENERAL DESCRIPTION OF THE REGAD PACKAGE
 AND AN OPTION TO STOP THE RUN IF ONLY THE DESCRIPTION IS DESIRED.

DATA YES/1HY/
 WRITE (6,10)
 WRITE (6,20)
 WRITE (6,30)
 WRITE (6,40)
 WRITE (6,50)


```

1AL OR,/,4X,32HOBTAI A LISTING OF THE PROGRAM.,/,1X,65(1H*))
60  FORMAT (/,/,4X,53HDO YOU WISH THE PROGRAM TO CONTINUE INTO THE ANAL1MOD3480
1YSIS,/,4X,30HAND DESIGN SEGMENTS? (Y OR N):)
70  FORMAT (/,/,5X,35H***** PROGRAM STOPPED BY USER *****1MOD3490
80  FORMAT (1A1)1MOD3500
END1MOD3510
1MOD3520
1MOD3530
1MOD3540
1MOD3550
1MOD3560
1MOD3570
SUBROUTINE INPUT1MOD3580
1MOD3590
1MOD3600
1MOD3610
1MOD3620
1MOD3630
1MOD3640
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELI(3),PD(3),1MOD3650
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3)1MOD3660
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX1MOD3670
1MOD3680
INITIALIZE1MOD3690
1MOD3700
DATA YES/1HY/1MOD3710
DEGRAD=4.*ATAN(1.)/180.1MOD3720
1MOD3730
COMMON DATA1MOD3740
1MOD3750
WRITE (6,230)1MOD3760
READ (5,*) IOPRO1MOD3770
IF (IOPRO.LT.1) IOPRO=11MOD3780
IF (IOPRO.GT.2) IOPRO=21MOD3790
WRITE (6,240)1MOD3800
READ (5,*) NRED1MOD3810
IF ((NRED.GE.1).AND.(NRED.LE.3)) GO TO 201MOD3820
WRITE (6,250) NRED1MOD3830

```

20	GO TO 10	1MOD3840
	WRITE (6,260)	1MOD3850
	READ (5,*) NHELX	1MOD3860
	IF (NHELX.LT.1) NHELX=1	1MOD3870
	IF (NHELX.GT.2) NHELX=2	1MOD3880
	GO TO (30,40), IARR	1MOD3890
C		1MOD3900
	PARALLEL AXIS DATA	1MOD3910
C		1MOD3920
30	WRITE (6,270)	1MOD3930
	READ (5,*) NPATH	1MOD3940
	IF (NPATH.LT.1) NPATH=1	1MOD3950
	IF (NPATH.GT.2) NPATH=2	1MOD3960
	GO TO 50	1MOD3970
C		1MOD3980
	EPICYCLIC DATA	1MOD3990
C		1MOD4000
40	NPWRIN=1	1MOD4010
	NPATH=2	1MOD4020
	GO TO 60	1MOD4030
C		1MOD4040
	COMMON DESIGN DATA	1MOD4050
C		1MOD4060
	POWER SOURCE DATA	1MOD4070
C		1MOD4080
50	WRITE (6,280)	1MOD4090
	READ (5,*) NPWRIN	1MOD4100
	IF (NPWRIN.LT.1) NPWRIN=1	1MOD4110
	IF (NPWRIN.GT.2) NPWRIN=2	1MOD4120
60	WRITE (6,290)	1MOD4130
	READ (5,550) REP1	1MOD4140
	GO TO (70,80), NPWRIN	1MOD4150
C		1MOD4160
	SINGLE POWER SOURCE	1MOD4170
C		1MOD4180
C		1MOD4190

```

70      IF (REP1.EQ.YES) IPWRSR(1)=2
        WRITE (6,300)
        READ (5,*) PWRIN(1),RPMIN(1)
        GO TO 110
C
C      DUAL POWER SOURCES
C
80      IF (REP1.NE.YES) GO TO 100
        WRITE (6,310)
        READ (5,*) IENG
        IF ((IENG.GE.1).AND.(IENG.LE.3)) GO TO 90
        WRITE (6,320) IENG
        GO TO 80
90      IF ((IENG.EQ.1).OR.(IENG.EQ.3)) IPWRSR(1)=2
        IF ((IENG.EQ.2).OR.(IENG.EQ.3)) IPWRSR(2)=2
100     WRITE (6,330)
        READ (5,*) PWRIN(1),RPMIN(1)
        WRITE (6,340)
        READ (5,*) PWRIN(2),RPMIN(2)
C
C      OUTPUT SPEED OF REDUCTION SET
C
C      NDIPP=NPWRIN
110     IF ((NPWRIN.EQ.2).AND.(PWRIN(1).EQ.PWRIN(2)).AND.(RPMIN(1).EQ.RPMIN(2)))
        1N(2)) NDIPP=1
        WRITE (6,350)
        READ (5,*) RPMOUT
C
C      DESIGN PARAMETER DATA
C
C
        WRITE (6,360)
        READ (5,*) IPD
        IF (IPD.LT.1) IPD=1
        IF (IPD.GT.2) IPD=2
        WRITE (6,370)
        READ (5,*) IPHI

```

```

1MOD4200
1MOD4210
1MOD4220
1MOD4230
1MOD4240
1MOD4250
1MOD4260
1MOD4270
1MOD4280
1MOD4290
1MOD4300
1MOD4310
1MOD4320
1MOD4330
1MOD4340
1MOD4350
1MOD4360
1MOD4370
1MOD4380
1MOD4390
1MOD4400
1MOD4410
1MOD4420
1MOD4430
1MOD4440
1MOD4450
1MOD4460
1MOD4470
1MOD4480
1MOD4490
1MOD4500
1MOD4510
1MOD4520
1MOD4530
1MOD4540
1MOD4550

```

```

120 IF (IPHI.LT.1) IPHI=1
    IF (IPHI.GT.2) IPHI=2
    DO 220 I=1,NRED
    WRITE (6,380) I
    WRITE (6,390)
    READ (5,*) DHELIX(I)
    IF ((NHELX.EQ.1).AND.(DHELIX(I).GE.15.0).AND.(DHELIX(I).LE.25.0))
    1GO TO 130
    IF ((NHELX.EQ.2).AND.(DHELIX(I).GE.25.0).AND.(DHELIX(I).LE.50.0))
    1GO TO 130
    WRITE (6,400) DHELIX(I),NHELX
    GO TO 120
130 HELIX(I)=DHELIX(I)*DEGRAD
    GO TO (140,150), IPD
140 WRITE (6,410)
    READ (5,*) PD(I)
    PND(I)=PD(I)/COS(HELIX(I))
    GO TO 160
150 WRITE (6,420)
    READ (5,*) PND(I)
    PD(I)=PND(I)*COS(HELIX(I))
    GO TO (170,180), IPHI
160 WRITE (6,430)
    READ (5,*) DPHI(I)
    PHI(I)=DPHI(I)*DEGRAD
    ARG=TAN(PHI(I))*COS(HELIX(I))
    PHIN(I)=ATAN(ARG)
    DPHIN(I)=PHIN(I)/DEGRAD
    GO TO 190
170 WRITE (6,440)
    READ (5,*) DPHIN(I)
    PHIN(I)=DPHIN(I)*DEGRAD
    ARG=TAN(PHIN(I))/COS(HELIX(I))
    PHI(I)=ATAN(ARG)
    DPHI(I)=PHI(I)/DEGRAD
180 IF (IAER.EQ.1) GO TO 210
190
1MOD4560
1MOD4570
1MOD4580
1MOD4590
1MOD4600
1MOD4610
1MOD4620
1MOD4630
1MOD4640
1MOD4650
1MOD4660
1MOD4670
1MOD4680
1MOD4690
1MOD4700
1MOD4710
1MOD4720
1MOD4730
1MOD4740
1MOD4750
1MOD4760
1MOD4770
1MOD4780
1MOD4790
1MOD4800
1MOD4810
1MOD4820
1MOD4830
1MOD4840
1MOD4850
1MOD4860
1MOD4870
1MOD4880
1MOD4890
1MOD4900
1MOD4910

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250 FORMAT (4X,I2,42H IS NOT A LEGITIMATE NUMBER OF REDUCTIONS.) 1MOD5280
260 FORMAT (//,4X,32HCHOOSE DESIRED HELIX TYPE BELOW://,8X,6H TYPE ,4X1MOD5290
1,5HANGLE,4X,4HCODE,//,8X,6HSINGLE,4X,5H15-25,6X,1H1,//,8X,6HDOUBLE,41MOD5300
2X,5H25-50,6X,1H2,//,1X,20H** ENTER HELIX CODE:) 1MOD5310
270 FORMAT (//,1X,50H** ENTER NUMBER OF POWER PATHS (1=SINGLE, 2=DUAL):1MOD5320
1) 1MOD5330
280 FORMAT (//,1X,42H** ENTER NUMBER OF POWER SOURCES (1 OR 2):) 1MOD5340
290 FORMAT (//,4X,64HWILL ANY POWER SOURCE BE A MULTICYLINDER I. C. ENGIN1MOD5350
1INE? (Y OR N):) 1MOD5360
300 FORMAT (//,1X,54H** ENTER POWER AND SPEED OF THE POWER SOURCE (HP,R1MOD5370
1PM):) 1MOD5380
310 FORMAT (//,4X,52HCHOOSE WHICH SOURCE(S) WILL BE AN I.C. ENGINE BEL1MOD5390
1OW://,20X,12HPower SOURCE,4X,4HCODE,//,21X,10HHIGH POWER,7X,1H1,//,21MOD5400
21X,10HLOW POWER,7X,1H2,//,24X,4HBOTh,10X,1H3,//,1X,26H** ENTER I.C1MOD5410
3. ENGINE CODE:) 1MOD5420
320 FORMAT (4X,I2,38H IS NOT A LEGITIMATE I.C. ENGINE CODE.) 1MOD5430
330 FORMAT (//,1X,55H** ENTER POWER AND SPEED OF HIGH POWER SOURCE (HP,1MOD5440
1RPM):) 1MOD5450
340 FORMAT (//,1X,54H** ENTER POWER AND SPEED OF LOW POWER SOURCE (HP,R1MOD5460
1PM):) 1MOD5470
350 FORMAT (//,1X,44H** ENTER OUTPUT SHAFT/PROPELLER SPEED (RPM):) 1MOD5480
360 FORMAT (//,4X,39HWHICH DIAMETRAL PITCH WILL YOU SPECIFY?//,4X,25H(1MOD5490
11=TRANSVERSE, 2=NORMAL):) 1MOD5500
370 FORMAT (//,4X,38HWHICH PRESSURE ANGLE WILL YOU SPECIFY?//,4X,25H(11MOD5510
1=TRANSVERSE, 2=NORMAL):) 1MOD5520
380 FORMAT (//,4X,48HTHE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE,1MOD5530
1I2,2H :) 1MOD5540
390 FORMAT (//,1X,31H** ENTER HELIX ANGLE (DEGREES):) 1MOD5550
400 FORMAT (//,4X,24HTHE HELIX ANGLE ENTERED,,F5.1,25H, DOES NOT AGREE1MOD5560
1 WITH THE,,4X,11HHELIX TYPE=,I2,36H CHOSEN. TYPE=1, SINGLE: 15-21MOD5570
25DEG.,//,4X,26HTYPE=2, DOUBLE: 25-50 DEG.) 1MOD5580
410 FORMAT (//,1X,36H** ENTER TRANSVERSE DIAMETRAL PITCH:) 1MOD5590
420 FORMAT (//,1X,32H** ENTER NORMAL DIAMETRAL PITCH:) 1MOD5600
430 FORMAT (//,1X,45H** ENTER TRANSVERSE PRESSURE ANGLE (DEGREES):) 1MOD5610
440 FORMAT (//,1X,41H** ENTER NORMAL PRESSURE ANGLE (DEGREES):) 1MOD5620
450 FORMAT (//,1X,46H** ENTER EPICYCLIC CODE (1=PLANETARY, 2=STAR):) 1MOD5630

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460 FORMAT (/,1X,41H** ENTER NUMBER OF PLANET GEARS (3 TO 5):) 1MOD5640
470 FORMAT (/,4X,I2,39H IS NOT A LEGITIMATE NUMBER OF PLANETS.) 1MOD5650
480 FORMAT (/,4X,33HCHOOSE GEAR HARDNESS RANGE BELOW:/,9X,3HBHM,8X,41MOD5660
1HCODE,/,6X,9H160 - 200,7X,1H1,/,6X,9H200 - 240,7X,1H2,/,6X,9H240 - 1MOD5670
2 300,7X,1H3,/,6X,9H300 - 360,7X,1H4,/,6X,9H360 - 400,7X,1H5,/,6X,91MOD5680
3H400 - 640,7X,1H6) 1MOD5690
490 FORMAT (/,1X,59H** ENTER HARDNESS CODES FOR PINION AND GEAR (HCPIN1MOD5700
1,HCGEAR):) 1MOD5710
500 FORMAT (/,1X,64H** ENTER HARDNESS CODES FOR SUN/PLANETS AND RING (1MOD5720
1HCSUN,HCRING):) 1MOD5730
510 FORMAT (/,4X,25HTHE PINION HARDNESS CODE,,I2,31H AND/OR THE GEAR 1MOD5740
1HARDNESS CODE,,/,4X,I2,26H ARE NOT LEGITIMATE CODES.) 1MOD5750
520 FORMAT (/,4X,29HTHE SUN/PLANET HARDNESS CODE,,I2,25H AND/OR THE R1MOD5760
1ING HARDNESS,,/,4X,5HCODE,,I2,26H ARE NOT LEGITIMATE CODES.) 1MOD5770
530 FORMAT (/,4X,61HTO MAKE CORRECTIONS TO DATA JUST ENTERED, THE PRO1MOD5780
1GRAM MUST BE,,/,4X,65HABORTED AND RE-STARTED. DO YOU WISH TO ABORT1MOD5790
2 THIS RUN? (Y OR N):) 1MOD5800
540 FORMAT (/,5X,44H***** RUN ABORTED BY USER --- RE-START ***** 1MOD5810
550 FORMAT (1A1) 1MOD5820
END 1MOD5830
C ***** 1MOD5840
C * * * * * 1MOD5850
C ***** 1MOD5860
SUBROUTINE AGMA 1MOD5870
C 1MOD5880
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982 1MOD5890
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940 1MOD5900
C 1MOD5910
C SUBPROGRAM TO LIST AND OPTIONALLY CHANGE THE PRE-PROGRAMMED 1MOD5920
C AGMA CONSTANTS 1MOD5930
C 1MOD5940
C EXTERNAL SUBPROGRAM(S) REQUIRED: FUNCTION CKDATA 1MOD5950
C 1MOD5960
C COMMON /AGHAB/ SFB(2,2),AKV,AKS,AKH,AKO(2),SAT(6),AKL(2),AKR(6),AK1MOD5970
1T 1MOD5980
C COMMON /AGNAH/ SFH(2,2),CV(3),CS,CH(2),CF,CO(2),SAC(6),CP,CL(2),CH1MOD5990

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C	1,CT,CR(6)	1MOD6000
C		1MOD6010
C	INITIALIZATION	1MOD6020
C		1MOD6030
	DATA YES/1HY//,EP/30.E06//,EG/30.E06//,VP/.3//,VG/.3/	1MOD6040
	ID=0	1MOD6050
C		1MOD6060
C	PROVIDE LISTING OF CONSTANTS AND THEIR VALUES	1MOD6070
C		1MOD6080
C		1MOD6090
10	WRITE (6,340)	1MOD6100
	WRITE (6,350) SFB(1,1),SFB(1,2),SFB(2,1),SFB(2,2)	1MOD6110
	WRITE (6,360) (CV(I),I=1,3),CS,CH(1),CH(2),CF	1MOD6120
	WRITE (6,370) CO(1),CO(2),CP,CL(1),CL(2),CH,CT	1MOD6130
	WRITE (6,380) (CR(I),I=1,6)	1MOD6140
	WRITE (6,390) (SAC(I),I=1,6)	1MOD6150
	WRITE (6,400) AKV,AKS,AKM,AKO(1),AKO(2),AKL(1),AKL(2)	1MOD6160
	WRITE (6,410) AKT,(AKR(I),I=1,6)	1MOD6170
	WRITE (6,420) (SAT(I),I=1,6)	1MOD6180
	WRITE (6,430)	1MOD6190
	WRITE (6,440)	1MOD6200
	IF (ID.EQ.99) RETURN	1MOD6210
	WRITE (6,450)	1MOD6220
	READ (5,700) REP	1MOD6230
	IF (REP.NE.YES) RETURN	1MOD6240
C		1MOD6250
C	CHANGE SELECTED CONSTANTS	1MOD6260
C		1MOD6270
	WRITE (6,460)	1MOD6280
	WRITE (6,470)	1MOD6290
20	READ (5,*) ID	1MOD6300
	IF (ID.EQ.99) GO TO 10	1MOD6310
	IF ((ID.LT.1).OR.(ID.GT.20)) GO TO 20	1MOD6320
	GO TO (30,50,70,80,100,110,130,140,160,170,180,200,220,230,240,250,270,290,300,320), ID	1MOD6330
	1,270,290,300,320), ID	1MOD6340
30	DO 40 I=1,2	1MOD6350
	DO 40 J=1,2	

40	WRITE (6,480) I,J	1MOD6360
	SPB(I,J)=CKDATA(SPB(I,J))	1MOD6370
	SFH(I,J)=SPB(I,J)	1MOD6380
	CONTINUE	1MOD6390
	GO TO 20	1MOD6400
50	DO 60 I=1,3	1MOD6410
	WRITE (6,490) I	1MOD6420
	CV(I)=CKDATA(CV(I))	1MOD6430
60	CONTINUE	1MOD6440
	GO TO 20	1MOD6450
70	WRITE (6,500)	1MOD6460
	CS=CKDATA(CS)	1MOD6470
	GO TO 20	1MOD6480
80	DO 90 I=1,2	1MOD6490
	WRITE (6,510) I	1MOD6500
	CH(I)=CKDATA(CH(I))	1MOD6510
90	CONTINUE	1MOD6520
	GO TO 20	1MOD6530
100	WRITE (6,520)	1MOD6540
	CF=CKDATA(CF)	1MOD6550
	GO TO 20	1MOD6560
110	DO 120 I=1,2	1MOD6570
	WRITE (6,530) I	1MOD6580
	CO(I)=CKDATA(CO(I))	1MOD6590
120	CONTINUE	1MOD6600
	GO TO 20	1MOD6610
130	WRITE (6,540) EP,EG,VP,VG	1MOD6620
	WRITE (6,550)	1MOD6630
	READ (5,*) VAL1,VAL2	1MOD6640
	IF (VAL1.NE.0.0) EP=VAL1	1MOD6650
	IF (VAL2.NE.0.0) EG=VAL2	1MOD6660
	WRITE (6,560)	1MOD6670
	READ (5,*) VAL1,VAL2	1MOD6680
	IF (VAL1.NE.0.0) VP=VAL1	1MOD6690
	IF (VAL1.NE.0.0) VG=VAL2	1MOD6700
	AP=(1.-VP*VP)/EP	1MOD6710

1MOD6720
 1MOD6730
 1MOD6740
 1MOD6750
 1MOD6760
 1MOD6770
 1MOD6780
 1MOD6790
 1MOD6800
 1MOD6810
 1MOD6820
 1MOD6830
 1MOD6840
 1MOD6850
 1MOD6860
 1MOD6870
 1MOD6880
 1MOD6890
 1MOD6900
 1MOD6910
 1MOD6920
 1MOD6930
 1MOD6940
 1MOD6950
 1MOD6960
 1MOD6970
 1MOD6980
 1MOD6990
 1MOD7000
 1MOD7010
 1MOD7020
 1MOD7030
 1MOD7040
 1MOD7050
 1MOD7060
 1MOD7070

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    AG=(1.-VG*VG)/EG
    A=4.*ATAN(1.)* (AP+AG)
    CP=SQRT(1./A)
    GO TO 20
140  DO 150 I=1,2
      WRITE (6,570) I
      CL(I)=CKDATA(CL(I))
      CONTINUE
      GO TO 20
150  WRITE (6,580)
      CH=CKDATA(CH)
      GO TO 20
160  WRITE (6,590)
      CT=CKDATA(CT)
      GO TO 20
170  DO 190 I=1,6
      WRITE (6,600) I
      CR(I)=CKDATA(CR(I))
      CONTINUE
      GO TO 20
180  DO 210 I=1,6
      WRITE (6,610) I
      SAC(I)=CKDATA(SAC(I))
      CONTINUE
      GO TO 20
200  WRITE (6,620)
      AKV=CKDATA(AKV)
      GO TO 20
210  WRITE (6,630)
      AKS=CKDATA(AKS)
      GO TO 20
220  WRITE (6,640)
      AKM=CKDATA(AKM)
      GO TO 20
230  DO 260 I=1,2
      WRITE (6,650) I
  
```

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260      AKO(I)=CKDATA(AKO(I))
      CONTINUE
      GO TO 20
270      DO 280 I=1,2
      WRITE (6,660) I
      AKL(I)=CKDATA(AKL(I))
      CONTINUE
      GO TO 20
280      WRITE (6,670)
      AKT=CKDATA(AKT)
      GO TO 20
300      DO 310 I=1,6
      WRITE (6,680) I
      AKR(I)=CKDATA(AKR(I))
      CONTINUE
      GO TO 20
320      DO 330 I=1,6
      WRITE (6,690) I
      SAT(I)=CKDATA(SAT(I))
      CONTINUE
      GO TO 20
      C
      C
      C
      C
340      FORMAT (1H1,4X,58HTHE FOLLOWING IS A LISTING OF THE PRE-PROGRAMMED
1      CONSTANTS,/,4X,55HUSED IN THE AGMA FORMULATIONS WITH APPROPRIATE
2      NOTES ON,/,4X,55HTHEIR APPLICATION. NOTE: THOSE STARTING WITH A
3      'C' ARE,/,4X,54HDURABILITY CONSTANTS AND THOSE WITH A 'K' ARE STRENGTH
4      FACTORS,/,4X,56HCONSTANTS. SERVICE FACTOR APPLIES TO BOTH FORMULATIONS
5      55,/)
350      FORMAT (4X,2HID,4X,5HCONST,4X,8HVALUE(S),3X,5HNOTES,/,4X,12H 1 SF1MOD7390
1      (1,1),5X,F4.2,5X,21HSERVICE FACTOR: A1,B1,/,4X,12H SF(1,2),5X,F1MOD7400
2      24.2,5X,21H A1,B2,/,4X,12H SF(2,1),5X,F4.2,5X,21H1MOD7410
3      3 A2,B1,/,4X,12H SF(2,2),5X,F4.2,5X,21H 1MOD7420
4      4 A2,B2,/)
1MOD7080
1MOD7090
1MOD7100
1MOD7110
1MOD7120
1MOD7130
1MOD7140
1MOD7150
1MOD7160
1MOD7170
1MOD7180
1MOD7190
1MOD7200
1MOD7210
1MOD7220
1MOD7230
1MOD7240
1MOD7250
1MOD7260
1MOD7270
1MOD7280
1MOD7290
1MOD7300
1MOD7310
1MOD7320
1MOD7330
1MOD7340
1MOD7350
1MOD7360
1MOD7370
1MOD7380
1MOD7390
1MOD7400
1MOD7410
1MOD7420
1MOD7430

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360 FORMAT (4X,12H 2 CV(1) ,5X,F4.2,5X,18HDYNAMIC FACTOR; C1,/4X,121MOD7440
1H CV(2) ,5X,F4.2,5X,18H C2,/4X,12H CV(3) 1MOD7450
2,5X,F4.2,5X,18H C3,/4X,12H 3 CS ,5X,F4.2,5X,1MOD7460
311HSIZE FACTOR, //4X,12H 4 CM(1) ,5X,F4.2,5X,28HLOAD DISTRIBUTIO1MOD7470
4N FACTOR; A1,/4X,12H CM(2) ,5X,F4.2,5X,28H 1MOD7480
5 A2,/4X,12H 5 CF ,5X,F4.2,5X,24HSURFACE CONDITION FAC1MOD7490
6TOR,/)

370 FORMAT (4X,12H 6 CO(1) ,5X,F4.2,5X,19HOVERLOAD FACTOR; A1,/4X,11MOD7510
12H CO(2) ,5X,F4.2,5X,19H A2,/4X,12H 7 C1MOD7520
2P ,4X,F6.1,4X,25HELASTIC PROPERTIES FACTOR, //4X,12H 8 CL(1) ,51MOD7530
3X,F4.2,5X,15HLIFE FACTOR; A1,/4X,12H CL(2) ,5X,F4.2,5X,15H 1MOD7540
4 A2,/4X,12H 9 CH ,5X,F4.2,5X,21HHARDNESS RATIO FAC1MOD7550
5TOR, //4X,12H10 CT ,5X,F4.2,5X,18HTEMPERATURE FACTOR, // 1MOD7560
380 FORMAT (4X,12H11 CR(1) ,5X,F4.2,5X,22HRELIABILITY FACTOR; D1,/41MOD7570
1X,12H CR(2) ,5X,F4.2,5X,22H D2,/4X,12H 1MOD7580
2 CR(3) ,5X,F4.2,5X,22H D3,/4X,12H CR(4) ,1MOD7590
35X,F4.2,5X,22H D4,/4X,12H CR(5) ,5X,F4.2,51MOD7600
4X,22H D5,/4X,12H CR(6) ,5X,F4.2,5X,22H 1MOD7610
5 D6,/)

390 FORMAT (4X,12H12 SAC(1) ,4X,F7.0,3X,28HALLOWABLE CONTACT STRESS;1MOD7630
1 D1,/4X,12H SAC(2) ,4X,F7.0,3X,28H D21MOD7640
2,/4X,12H SAC(3) ,4X,F7.0,3X,28H D3,/41MOD7650
3X,12H SAC(4) ,4X,F7.0,3X,28H D4,/4X,11MOD7660
42H SAC(5) ,4X,F7.0,3X,28H D5,/4X,12H 1MOD7670
5 SAC(6) ,4X,F7.0,3X,28H D6,/)

400 FORMAT (4X,12H13 KV ,5X,F4.2,5X,14HDYNAMIC FACTOR, //4X,12H141MOD7690
1 KS ,5X,F4.2,5X,11HSIZE FACTOR, //4X,12H15 KH ,5X,F4.2,1MOD7700
25X,24HLOAD DISTRIBUTION FACTOR, //4X,12H16 KO(1) ,5X,F4.2,5X,19H1MOD7710
3OVERLOAD FACTOR; E1,/4X,12H KO(2) ,5X,F4.2,5X,19H 1MOD7720
4 E2,/4X,12H17 KL(1) ,5X,F4.2,5X,15HLIFE FACTOR; A1,/4X,12H1MOD7730
5 KL(2) ,5X,F4.2,5X,15H A2,/)

410 FORMAT (4X,12H18 KT ,5X,F4.2,5X,18HTEMPERATURE FACTOR, //4X,11MOD7750
12H19 KR(1) ,5X,F4.2,5X,22HRELIABILITY FACTOR; D1,/4X,12H K1MOD7760
2R(2) ,5X,F4.2,5X,22H D2,/4X,12H KR(3) ,5X,1MOD7770
3F4.2,5X,22H D3,/4X,12H KR(4) ,5X,F4.2,5X,21MOD7780
42H D4,/4X,12H KR(5) ,5X,F4.2,5X,22H 1MOD7790

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5          D5, /4X, 12H      KR (6) ,5X, F4.2,5X, 22H      1MOD7800
6          D6, /)              1MOD7810
420  FORMAT (4X, 12H20      SAT (1), 4X, F6.0, 4X, 29HALLOWABLE MATERIAL STRESS1MOD7820
1: D1, /4X, 12H      SAT (2), 4X, F6.0, 4X, 29H      1MOD7830
202, /4X, 12H      SAT (3), 4X, F6.0, 4X, 29H      D31MOD7840
3, /4X, 12H      SAT (4), 4X, F6.0, 4X, 29H      D4, /1MOD7850
44X, 12H      SAT (5), 4X, F6.0, 4X, 29H      D5, /4X1MOD7860
5, 12H      SAT (6), 4X, F6.0, 4X, 29H      D6, /) 1MOD7870
430  FORMAT (5X, 38HDEFINITIONS OF CODED NOTES FROM ABOVE: /6X, 45HA1 NA1MOD7880
1VAL PROFILE - FULL POWER, 5 PERCENT MAX, /6X, 36HA2 OTHER - MAXIMUM1MOD7890
2 LOAD, CONTINUOUS, /6X, 35HB1 POWER SOURCE - TURBINE OR MOTOR, /6X, 1MOD7900
345HB2 POWER SOURCE - MULTICYLINDER I. C. ENGINE, /6X, 25HC1 FIRST1MOD7910
4 REDUCTION STAGE, /6X, 26HC2 SECOND REDUCTION STAGE, /6X, 25HC3 THIR1MOD7920
5D REDUCTION STAGE, /)      1MOD7930
440  FORMAT (6X, 33HD1 HARDNESS RANGE: 160 - 200 BHN, /6X, 33HD2 HARDNES1MOD7940
1S RANGE: 200 - 240 BHN, /6X, 33HD3 HARDNESS RANGE: 240 - 300 BHN, /61MOD7950
2X, 33HD4 HARDNESS RANGE: 300 - 360 BHN, /6X, 33HD5 HARDNESS RANGE: 1MOD7960
3360 - 400 BHN, /6X, 33HD6 HARDNESS RANGE: 400 - 640 BHN, /6X, 21HE1 1MOD7970
4 SINGLE POWER PATH, /6X, 21HE2 DOUBLE POWER PATH, /)      1MOD7980
450  FORMAT (4X, 58HDO YOU DESIRE TO CHANGE ANY OF THE ABOVE VALUES? (Y 1MOD7990
1OR N):)      1MOD8000
460  FORMAT (/6X, 61HTO CHANGE A CONSTANT ABOVE, ENTER THE ID NUMBER W1MOD8010
1HEN PROMPTED. /6X, 56HUSE ID NUMBER 99 WHEN NO FURTHER CHANGES ARE 1MOD8020
2TO BE MADE. /6X, 60HNOTE: WHEN ASKED FOR THE NEW VALUE OF THE CONS1MOD8030
3TANT, ENTERING, /6X, 57HA ZERO WILL CAUSE THE ORIGINAL VALUE TO REM1MOD8040
4AIN UNCHANGED. /6X, 59HTHIS IS USEFUL WHEN A CONSTANT HAS MULTIPLE1MOD8050
5 VALUES, PUT NOT, /6X, 30HALL OF THEM ARE TO BE CHANGED.)      1MOD8060
470  FORMAT (/6X, 51H** ENTER THE CONSTANT ID NUMBER (1-20, 99 TO STOP1MOD8070
1):)      1MOD8080
480  FORMAT (/6X, 12H** ENTER SF(, I1, 1H, , I1, 2H):)      1MOD8090
490  FORMAT (/6X, 12H** ENTER CV(, I1, 2H):)      1MOD8100
500  FORMAT (/6X, 12H** ENTER CS:)      1MOD8110
510  FORMAT (/6X, 12H** ENTER CM(, I1, 2H):)      1MOD8120
520  FORMAT (/6X, 12H** ENTER CF:)      1MOD8130
530  FORMAT (/6X, 12H** ENTER CO(, I1, 2H):)      1MOD8140
540  FORMAT (/6X, 31HCURRENT YOUNG'S MODULI ARE: EP=, 2PE9.1, 6H, EG=, E91MOD8150

```

```

1.1,/,4X,33HCURRENT POISSON'S RATIOS ARE: VP=,0PF5.3,6H,  VG=,F5.3) 1MOD8160
550  FORMAT (/,4X,52H** ENTER YOUNG'S MODULI FOR PINION AND GEAR (EP,EG1MOD8170
1):)
560  FORMAT (/,4X,53H** ENTER POISSON'S RATIO FOR PINION AND GEAR (VP,V1MOD8190
1G):)
570  FORMAT (/,4X,12H** ENTER CL(,I1,2H):) 1MOD8200
580  FORMAT (/,4X,12H** ENTER CH:) 1MOD8210
590  FORMAT (/,4X,12H** ENTER CT:) 1MOD8220
600  FORMAT (/,4X,12H** ENTER CR(,I1,2H):) 1MOD8230
610  FORMAT (/,4X,13H** ENTER SAC(,I1,2H):) 1MOD8240
620  FORMAT (/,4X,12H** ENTER KV:) 1MOD8250
630  FORMAT (/,4X,12H** ENTER KS:) 1MOD8260
640  FORMAT (/,4X,12H** ENTER KH:) 1MOD8270
650  FORMAT (/,4X,12H** ENTER KO(,I1,2H):) 1MOD8280
660  FORMAT (/,4X,12H** ENTER KL(,I1,2H):) 1MOD8290
670  FORMAT (/,4X,12H** ENTER KT:) 1MOD8300
680  FORMAT (/,4X,12H** ENTER KR(,I1,2H):) 1MOD8310
690  FORMAT (/,4X,13H** ENTER SAT(,I1,2H):) 1MOD8320
700  FORMAT (1A1) 1MOD8330
END 1MOD8340
1MOD8350

```



```
C
C SUBROUTINE PRLAN
C
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
C
C SUBPROGRAM TO PERFORM AN ANALYSIS OF A GIVEN PARALLEL
C AXIS GEAR SET
C
C EXTERNAL SUBPROGRAM(S) REQUIRED: SUBROUTINE GPI, SUBROUTINE GFJ,
C FUNCTION AGMAE1, FUNCTION ARCCOS, FUNCTION ARCSIN, FUNCTION FALFA,
C FUNCTION RTFNDR, FUNCTION SHRLD, FUNCTION THICK
C
C REAL MGOP,MGP
C COMMON /DESDAT/ PHRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),
C 1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3,
C 2,2),IOPRO,NPWHRIN,IPWRSR(2),NRRED,NPATH,NPLNT(3),NHHELX
C COMMON /DESPRL/ PWRFAC(2,3),MGOP(2),NGP(3,2),RPMP(6,2),PWRP(6,2),
C 1P(3,2),DG(3,2),FACEP(3,2),GEOMI(3,2),GEOMJP(3,2),GEOMJG(3,2),NP(
C 22),NG(3,2)
C
C INITIALIZATION
C
C NRRED=NRRED*2
C
C ENTER REQUIRED INFORMATION: DIAMETERS AND FACEWIDTHS
C
C DO 20 J=1,NDIFF
C DO 20 I=1,NRED
C WRITE (6,70) I,J
C IF ((J.EQ.2).AND.(I.EQ.NRED)) GO TO 10
C WRITE (6,80)
C READ (5,*) DP(I,J),DG(I,J)
C MGP(I,J)=DG(I,J)/DP(I,J)
```

2MOD0340
2MOD0350
2MOD0360
2MOD0370
2MOD0380
2MOD0390
2MOD0400
2MOD0410
2MOD0420
2MOD0430
2MOD0440
2MOD0450
2MOD0460
2MOD0470
2MOD0480
2MOD0490
2MOD0500
2MOD0510
2MOD0520
2MOD0530
2MOD0540
2MOD0550
2MOD0560
2MOD0570
2MOD0580
2MOD0590
2MOD0600
2MOD0610
2MOD0620
2MOD0630
2MOD0640
2MOD0650
2MOD0660
2MOD0670
2MOD0680
2MOD0690

```

NP(I,J)=INT(PD(I)*DP(I,J)+.5)
NG(I,J)=INT(PD(I)*DG(I,J)+.5)
WRITE(6,90)
READ(5,*)FACEP(I,J)
GO TO 20
WRITE(6,100)
READ(5,*)DP(NRED,2)
NP(NRED,2)=INT(PD(I)*DP(NRED,2)+.5)
NG(NRED,2)=NG(NRED,1)
DG(NRED,2)=DG(NRED,1)
FACEP(NRED,2)=FACEP(NRED,1)
MGP(NRED,2)=DG(NRED,2)/DP(NRED,2)
CONTINUE

```

10

20

C
C
C

COMPUTE RATIOS, SPEED AND POWER SPLITS, AND GEOMETRY FACTORS

```

DO 30 J=1,NDIFF
L=1
RPHP(1,J)=RPHIN(J)
RPHP(2,J)=RPHP(1,J)/MGP(L,J)
IF(NRED.EQ.1)GO TO 30
DO 30 I=3,NRED2,2
L=L+1
IM1=I-1
IP1=I+1
RPHP(I,J)=RPHP(IM1,J)
RPHP(IP1,J)=RPHP(I,J)/MGP(L,J)
CONTINUE
DO 50 J=1,NDIFF
PWR1=PHRIN(J)
L=0
DO 40 I=1,NRED2,2
IP1=I+1
L=L+1
PWRP(I,J)=PWR1*PWRFAC(NPATH,L)
PWRP(IP1,J)=PWRP(I,J)/FLOAT(L*NPATH)

```

30


```

MM=10*NDV
IF (NHXLX.EQ.2) FDP=2.25
IF (NHELX.EQ.2) SCALE(3)=150.
IRET=1
WRITE (6,440)
READ (5,*) RND

C
C C COMPUTE THE STAGE GEAR RATIOS FOR THE INITIAL DESIGN
C
DO 60 J=1,NDIFF
MGOP(J)=RPHIN(J)/RPMOUT
MGO=MGOP(J)
GO TO (10,20,30,40,50), LL
IF ((MGO.LE.1.0).OR.(MGO.GT.10.0)) GO TO 370
MQG(1,J)=MGO
GO TO 60
IF ((MGO.LE.2.24).OR.(MGO.GT.10.0)) GO TO 370
MQG(1,J)=MGO
GO TO 60
IF ((MGO.LE.2.0).OR.(MGO.GT.20.0)) GO TO 370
MQG(2,J)=SQRT(MGO)-1.
MQG(1,J)=MGO/MQG(2,J)
GO TO 60
IF ((MGO.LE.2.9).OR.(MGO.GT.48.4)) GO TO 370
MQG(2,J)=SQRT(MGO)+3.
MQG(1,J)=MGO/MQG(2,J)
GO TO 60
IF (MGO.LT.5) GO TO 370
MQG(2,J)=MGO**E
MQG(3,J)=MQG(2,J)+3.
MQG(1,J)=MGO/(MQG(2,J)*MQG(3,J))
CONTINUE
60
C
C C COMPUTE POWER AND SPEED SPLITS FOR THE INITIAL DESIGN
C
DO 80 J=1,NDIFF

```

```

RPM1=RPMIN(J)
PWR1=PWRIN(J)
DO 70 I=1,NRED
  SPDP(I,J)=RPM1
  SPDG(I,J)=RPM1/MGQ(I,J)
  RPM1=SPDG(I,J)
  HPP(I,J)=PWR1*PWRPAC(NPATH,I)
  HPG(I,J)=PWR1/FLOAT(NPATH*I)
CONTINUE
70 HPG(NRED,J)=PWR1
  HPG(NRED,J)=PWR1
CONTINUE
80
C
C
C
ESTIMATE INITIAL DESIGN AS START POINT FOR OPTIMIZATION
DO 90 J=1,NDIFF
DO 90 I=1,NRED
  IH=IHARD(I,1)
  BRAC=SAC(IH)*1.E-04/CR(IH)
  KK=BRAC*BRAC*REDFAC(I)*2.80/(CO(IOPRO)*CM(IOPRO))
  ANUN=126050.*HPP(I,J)*(MGQ(I,J)+1.)
  DEN=SPDP(I,J)*FDP*KK*MGQ(I,J)
  DPQ(I,J)=(ANUN/DEN)**E
  FACEQ(I,J)=FDP*DPQ(I,J)
CONTINUE
90
C
C
C
COMPUTE VALUES OF DEPENDENT VARIABLES
GO TO (110,120,130), NRED
C ***
110 SINGLE REDUCTION
  MGQ(1,1)=MGOP(1)
  DGQ(1,1)=MGQ(1,1)*DPQ(1,1)
  IF (NDIFF.EQ.1) GO TO 150
  MGQ(1,2)=MGOP(2)
  DGQ(1,2)=DGQ(1,1)
  DPQ(1,2)=DGQ(1,2)/MGQ(1,2)
  FACEQ(1,2)=FACEQ(1,1)

```

```

2MOD1780
2MOD1790
2MOD1800
2MOD1810
2MOD1820
2MOD1830
2MOD1840
2MOD1850
2MOD1860
2MOD1870
2MOD1880
2MOD1890
2MOD1900
2MOD1910
2MOD1920
2MOD1930
2MOD1940
2MOD1950
2MOD1960
2MOD1970
2MOD1980
2MOD1990
2MOD2000
2MOD2010
2MOD2020
2MOD2030
2MOD2040
2MOD2050
2MOD2060
2MOD2070
2MOD2080
2MOD2090
2MOD2100
2MOD2110
2MOD2120
2MOD2130

```

```

C ***
120 GO TO 150
    DOUBLE REDUCTION
    MGQ(1,1)=MGOP(1)/MGQ(2,1)
    DGQ(1,1)=MGQ(1,1)*DPQ(1,1)
    DGQ(2,1)=MGQ(2,1)*DPQ(2,1)
    IF (NDIFF.EQ.1) GO TO 150
    DGQ(2,2)=DGQ(2,1)
    MGQ(2,2)=DGQ(2,2)/DPQ(2,2)
    MGQ(1,2)=MGOP(2)/MGQ(2,2)
    DGQ(1,2)=MGQ(1,2)*DPQ(1,2)
    FACEQ(2,2)=FACEQ(1,2)
    GO TO 150
C ***
130 TRIPLE REDUCTION
    MGQ(1,1)=MGOP(1)/(MGQ(2,1)*MGQ(3,1))
    DO 140 I=1,3
    140 DGQ(I,1)=MGQ(I,1)*DPQ(I,1)
    IF (NDIFF.EQ.1) GO TO 150
    DGQ(3,2)=DGQ(3,1)
    MGQ(3,2)=DGQ(3,2)/DPQ(3,2)
    MGQ(1,2)=MGOP(2)/(MGQ(2,2)*MGQ(3,2))
    DGQ(2,2)=MGQ(2,2)*DPQ(2,2)
    DGQ(1,2)=MGQ(1,2)*DPQ(1,2)
    FACEQ(3,2)=FACEQ(3,1)
C
C
C
150 COMPUTE CONSTRAINTS AND OBJECTIVE FUNCTION
    VQ=0.0
    FLAGG=.FALSE.
    DO 190 J=1,NDIFF
    DO 190 I=1,NRED
    CALL GPI (GI(I,J),I,MGQ(I,J),DPQ(I,J),DGQ(I,J),0)
    APWRH=POWERH(SDP(I,J),FACEQ(I,J),DPQ(I,J),I,J,GI(I,J))
    CALL GFJ (GJP(I,J),I,DPQ(I,J),DGQ(I,J),1,0)
    APWRBP=POWERB(SDP(I,J),FACEQ(I,J),DPQ(I,J),I,J,1,GJP(I,J))
    CALL GFJ (GJG(I,J),I,DPQ(I,J),DGQ(I,J),2,0)
    APWRBG=POWERB(SDP(I,J),FACEQ(I,J),DPQ(I,J),I,J,2,GJG(I,J))

```

```

2MOD2140
2MOD2150
2MOD2160
2MOD2170
2MOD2180
2MOD2190
2MOD2200
2MOD2210
2MOD2220
2MOD2230
2MOD2240
2MOD2250
2MOD2260
2MOD2270
2MOD2280
2MOD2290
2MOD2300
2MOD2310
2MOD2320
2MOD2330
2MOD2340
2MOD2350
2MOD2360
2MOD2370
2MOD2380
2MOD2390
2MOD2400
2MOD2410
2MOD2420
2MOD2430
2MOD2440
2MOD2450
2MOD2460
2MOD2470
2MOD2480
2MOD2490

```

```

FLB=FOURPI/(PD(I)*TAN(HELIX(I)))
IF (FDP.EQ.2.25) FLB=DPQ(I,J)
FUB=FDP*DPQ(I,J)
G(1,I,J)=HPP(I,J)/APWRH-1.0
G(2,I,J)=HPP(I,J)/APWRBP-1.0
G(3,I,J)=HPG(I,J)/APWRBG-1.0
IF ((G(1,I,J).GT.0.) .OR. (G(2,I,J).GT.0.) .OR. (G(3,I,J).GT.0.)) FLAG=0.0
1G=.TRUE.
G(4,I,J)=DGQ(I,J)/200.-1.0
G(5,I,J)=FLB/FACEQ(I,J)-1.0
G(6,I,J)=FACEQ(I,J)/FUB-1.0
GO TO (180,160,170), NRED
160 IF (NPATH.EQ.1) G(7,I,J)=DGQ(1,J)/DGQ(2,J)-1.0
IF (NPATH.EQ.2) G(7,I,J)=MGQ(1,J)/MGQ(2,J)-1.0
G(8,I,J)=DPQ(1,J)/DPQ(2,J)-1.0
GO TO 180
170 G(7,I,J)=MGQ(1,J)/MGQ(2,J)-1.0
G(8,I,J)=MGQ(2,J)/MGQ(3,J)-1.0
G(9,I,J)=DPQ(1,J)/DPQ(2,J)-1.0
G(10,I,J)=DPQ(2,J)/DPQ(3,J)-1.0
VQ=VQ+.25*(MGQ(I,J)+1.)*(MGQ(I,J)+1.)*DPQ(I,J)*DPQ(I,J)*FACEQ(I,J)
180 VQ=VQ+.25*(MGQ(I,J)+1.)*(MGQ(I,J)+1.)*DPQ(I,J)*DPQ(I,J)*FACEQ(I,J)
190 CONTINUE
C
C CHECK FOR CONSTRAINT VIOLATIONS (CONSTRAINTS VIOLATED IF AT
C LEAST ONE HAS A VALUE GREATER THAN ZERO)
C
GMAX=-1.0E+20
IG=IGG(NRED)
DO 200 K=1,IG
DO 200 I=1,NRED
DO 200 J=1,NDIFF
GMAX=AMAX1(GMAX,G(K,I,J))
200 CONTINUE
GO TO (210,300), IRET
C
C SAVE THIS ITERATION'S DESIGN
C

```



```

C      210      GMYSTR=GMAX
      220      FLAG=.FALSE.
            IF (GMAX.GT.0.0) FLAG=.TRUE.
            VSTR=VQ
            KS=1
            DO 230 J=1,NDIFF
            L=0
            DO 230 I=1,NRED
            MGP(I,J)=MGQ(I,J)
            DP(I,J)=DPQ(I,J)
            DG(I,J)=DGQ(I,J)
            FACEP(I,J)=FACEQ(I,J)
            NP(I,J)=INT(DP(I,J)*PD(I)+.5)
            NG(I,J)=INT(DG(I,J)*PD(I)+.5)
            GEOMI(I,J)=GI(I,J)
            GEOMJP(I,J)=GJP(I,J)
            GEOMJG(I,J)=GJG(I,J)
            L=L+1
            RPHP(L,J)=SPDP(I,J)
            PWRP(L,J)=HPP(I,J)
            L=L+1
            RPHP(L,J)=SPDG(I,J)
            PWRP(L,J)=HPG(I,J)
            CONTINUE
      230      IF (IRET.EQ.2) GO TO 280
C
C      C      PERFORM LOCAL RANDOM SEARCHES NEAR INITIAL/HOST RECENT DESIGN
C
      240      IRET=2
            M=M+1
            IF (M.LT.NM) GO TO 250
            ALPHA=BB*ALPHA
            IF (ALPHA.LT.1.E-04) GO TO 340
            M=0
            SMAX=-1.E+10
      250

```

```

2MOD2860
2MOD2870
2MOD2880
2MOD2890
2MOD2900
2MOD2910
2MOD2920
2MOD2930
2MOD2940
2MOD2950
2MOD2960
2MOD2970
2MOD2980
2MOD2990
2MOD3000
2MOD3010
2MOD3020
2MOD3030
2MOD3040
2MOD3050
2MOD3060
2MOD3070
2MOD3080
2MOD3090
2MOD3100
2MOD3110
2MOD3120
2MOD3130
2MOD3140
2MOD3150
2MOD3160
2MOD3170
2MOD3180
2MOD3190
2MOD3200
2MOD3210

```

```

IS=0
DO 260 JJ=1,NDVP
DO 260 II=1,3
IS=IS+1
RND=RDNGEN(RND)
S(IS)=(2.*RND-1.)*SCALE(II)
SHAY=AMAX1(SHAY,ABS(S(IS)))
260 DO 270 IS=1,NDVP3
S(IS)=S(IS)/SHAY
KS=0
270 L=0
DO 290 JJ=1,NDIFF
DO 290 II=1,NRED
L=L+1
IF (FLAGG) S(L)=ABS(S(L))
DPQ(II,JJ)=DP(II,JJ)+ALPHA*S(L)
L=L+1
IF (FLAGG) S(L)=ABS(S(L))
HGQ(II,JJ)=HGP(II,JJ)+ALPHA*S(L)
L=L+1
290 PACEQ(II,JJ)=PACEP(II,JJ)+ALPHA*S(L)
GO TO 100
300 IQ=IQ+1
IF (IQ.GT.IQMAX) GO TO 340
IF (GMAX.GT.0.0) GO TO 330
IK=IK+1
IF (IK.EQ.1) ALPHA=1.0
IF (VQ.LT.VSTR) GO TO 220
IF (KS.EQ.1) GO TO 240
310 DO 320 IS=1,NDVP3
S(IS)=-S(IS)
KS=1
320 GO TO 280
330 IF (GMAX.GT.GMXSTR) GO TO 310
GMXSTR=GMAX
GO TO 220
2MOD3220
2MOD3230
2MOD3240
2MOD3250
2MOD3260
2MOD3270
2MOD3280
2MOD3290
2MOD3300
2MOD3310
2MOD3320
2MOD3330
2MOD3340
2MOD3350
2MOD3360
2MOD3370
2MOD3380
2MOD3390
2MOD3400
2MOD3410
2MOD3420
2MOD3430
2MOD3440
2MOD3450
2MOD3460
2MOD3470
2MOD3480
2MOD3490
2MOD3500
2MOD3510
2MOD3520
2MOD3530
2MOD3540
2MOD3550
2MOD3560
2MOD3570

```



```

CDP(I,J) = (DP(I,J) + DG(I,J)) / 2.
RPP(I,J) = NP(I,J) * RPP(M,J) / 60.
WTP(M,J) = 126050. * PWRP(M,J) / (RPP(M,J) * DP(I,J))
C1 = WTP(M,J) * CO(IOPRO) / CV(I)
C2 = CS / (FACEP(I,J) * DP(I,J))
C3 = CH(IOPRO) * CF / GEOMI(I,J)
SIGHP(I,J) = CP * SQRT(C1 * C2 * C3)
NPTH = NPATH
IF ((NRED.EQ.3) .AND. (I.GE.2)) NPTH=2
C1 = AKO(NPTH) / AKV
C2 = PD(I) / FACEP(I,J)
C3 = AKS * AKM
SIG = C1 * C2 * C3
SIGBP(M,J) = WTP(M,J) * SIG / GEOMJP(I,J)
TORQP(M,J) = WTP(M,J) * DP(I,J) / 2000.
TLPIP(M,J) = WTP(M,J) / FACEP(I,J)
UNTLDP(M,J) = TLPIP(M,J) * PND(I)
KFCTRP(M,J) = WTP(M,J) / DEN
HTHP(M,J) = NP(I,J)
PDIAMP(M,J) = DP(I,J)
M = M + 1
WTP(M,J) = 126050. * PWRP(M,J) / (RPP(M,J) * DG(I,J))
SIGBP(M,J) = WTP(M,J) * SIG / GEOMJP(I,J)
TORQP(M,J) = WTP(M,J) * DP(I,J) / 2000.
TLPIP(M,J) = WTP(M,J) / FACEP(I,J)
UNTLDP(M,J) = TLPIP(M,J) * PND(I)
KFCTRP(M,J) = WTP(M,J) / DEN
HTHP(M,J) = NG(I,J)
PDIAMP(M,J) = DG(I,J)
CONTINUE
IF (NPWRIN.EQ.1) RETURN

```

10

C

COMPUTE SOURCE CENTERLINE DISTANCE LIMITS

A MINIMUM 12.0 INCH CLEARANCE IS USED BETWEEN EACH POWER

TRAINS. FIRST REDUCTION GEARS' PITCH DIAMETERS.

C

2MOD4660
2MOD4670
2MOD4680
2MOD4690
2MOD4700
2MOD4710
2MOD4720
2MOD4730
2MOD4740
2MOD4750
2MOD4760
2MOD4770
2MOD4780
2MOD4790
2MOD4800
2MOD4810
2MOD4820
2MOD4830
2MOD4840
2MOD4850
2MOD4860
2MOD4870
2MOD4880
2MOD4890
2MOD4900
2MOD4910
2MOD4920
2MOD4930
2MOD4940
2MOD4950
2MOD4960
2MOD4970
2MOD4980
2MOD4990
2MOD5000
2MOD5010

```

C ***
20 GO TO (20,30,40,50,60,70), L
   SINGLE REDUCTION, SINGLE POWER PATH
   SCDMIN=DP(1,1)/2.+6.
   SCDMAX=SQRT(CDP(1,1)*CDP(1,1)-SCDMIN*SCDMIN)
   RETURN
C ***
30 SINGLE REDUCTION, DUAL POWER PATH
   A=DP(1,1)/2.+6.
   A1=ARCSIN(A,CDP(1,1))
   ARG=DP(1,1)/CDP(1,1)
   G1=A1+ATAN(ARG)
   CSTR=SQRT(CDP(1,1)*CDP(1,1)+DP(1,1)*DP(1,1))
   SCDMIN=CSTR*SIN(G1)
   SCDMAX=CSTR*COS(G1)
   RETURN
C ***
40 DOUBLE REDUCTION, SINGLE POWER PATH
   SCDMIN=DG(1,1)/2.+6.
   A1=ARCSIN(SCDMIN,CDP(2,1))
   SCDMAX=CDP(1,1)+CDP(2,1)*COS(A1)
   RETURN
C ***
50 DOUBLE REDUCTION, DUAL POWER PATH
   A=DG(1,1)/2.+6.
   A1=ARCSIN(A,CDP(2,1))
   ARG=CDP(1,1)/CDP(2,1)
   G1=A1+ATAN(ARG)
   CSTR=SQRT(CDP(1,1)*CDP(1,1)+CDP(2,1)*CDP(2,1))
   SCDMIN=CSTR*SIN(G1)
   SCDMAX=CSTR*COS(G1)
   IF (G1.GE.ATAN(1.)) SCDMAX=SCDMIN
   RETURN
C ***
60 TRIPLE REDUCTION, FIRST RED. HAS SINGLE POWER PATH
   A=DP(1,1)/2.+6.
   B=DG(1,1)/2.+6.
   B1=ARCSIN(B,CDP(3,1))
   ARG=CDP(2,1)/CDP(3,1)
   G1=B1+ATAN(ARG)
   CSTR=SQRT(CDP(2,1)*CDP(2,1)+CDP(3,1)*CDP(3,1))

```

```

2MOD5020
2MOD5030
2MOD5040
2MOD5050
2MOD5060
2MOD5070
2MOD5080
2MOD5090
2MOD5100
2MOD5110
2MOD5120
2MOD5130
2MOD5140
2MOD5150
2MOD5160
2MOD5170
2MOD5180
2MOD5190
2MOD5200
2MOD5210
2MOD5220
2MOD5230
2MOD5240
2MOD5250
2MOD5260
2MOD5270
2MOD5280
2MOD5290
2MOD5300
2MOD5310
2MOD5320
2MOD5330
2MOD5340
2MOD5350
2MOD5360
2MOD5370

```



```

60      D2F2=FNP*DG(1,J)*DG(1,J)*FACEP(1,J)
      D2F3=2.*DP(2,J)*DP(2,J)*FACEP(2,J)
      D2F4=4.*DG(2,J)*DG(2,J)*FACEP(2,J)
      D2F5=4.*DP(3,J)*DP(3,J)*FACEP(3,J)
      D2F=D2F+D2F1+D2F2+D2F3+D2F4+D2F5
      D2F=D2F+DG(3,1)*DG(3,1)*FACEP(3,1)
      WHTP=1196.*(D2F)**0.34
      IP=INT(ALOG10(WHTP))-2
      WHTP=AINT(WHTP/(10.**IP))*(10**IP)
      SPCWTP=WHTP/SHF
      C
      C
      C
      DIMENSIONS ESTIMATE
      DO 80 I=1,NRED
      IF (NPRIN.EQ.1) SF=SF+FACEP(I,1)
      IF (NPRIN.EQ.2) SF=SF+AMAX1(FACEP(I,1),FACEP(I,2))
      CONTINUE
      ISIZEP(1)=INT(2.26*SF+.5)
      ISIZEP(2)=INT(1.28*DG(NRED,1)+.5)
      WC=1.20
      IF (NPRIN.EQ.2) WC=1.37
      ISIZEP(3)=INT(WC*DG(NRED,1)+.5)
      RETURN
      END
      C
      C
      C
      SUBROUTINE PRLOUT
      C
      C
      C
      CODED BY:  LT J.L. PAQUETTE, USN      JAN 1982
               NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
      C
      C
      C
      SUBPROGRAM TO PRESENT ALL RESULTS FROM THE DESIGN/ANALYSIS
      FOR PARALLEL AXIS GEARS
      C
      C
      REAL MGOP,MGP,MFP,KFCTRP
      2MOD6100
      2MOD6110
      2MOD6120
      2MOD6130
      2MOD6140
      2MOD6150
      2MOD6160
      2MOD6170
      2MOD6180
      2MOD6190
      2MOD6200
      2MOD6210
      2MOD6220
      2MOD6230
      2MOD6240
      2MOD6250
      2MOD6260
      2MOD6270
      2MOD6280
      2MOD6290
      2MOD6300
      2MOD6310
      2MOD6320
      2MOD6330
      2MOD6340
      2MOD6350
      2MOD6360
      2MOD6370
      2MOD6380
      2MOD6390
      2MOD6400
      2MOD6410
      2MOD6420
      2MOD6430
      2MOD6440
      2MOD6450

```

```

DIMENSION KHard(6,2),MHard(12)
COMMON /DESdat/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),2MOD6460
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(32MOD6470
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
COMMON /DESPRL/ PWRPAC(2,3),MGOP(2),MGP(3,2),RPM(6,2),D2MOD6490
1P(3,2),DG(3,2),FACEF(3,2),GEOMI(3,2),GEOMJP(3,2),NP(3,2MOD6500
22),NG(3,2)
COMMON /RESPRL/ PLVP(3,2),FBYDP(3,2),CDP(3,2),WTP(6,2),TLPIP(6,2),2MOD6520
1UNTLDP(6,2),MFP(3,2),KFCTRP(6,2),SIGH(3,2),SIGBP(6,2),TORQP(6,2),2MOD6530
2PDAMP(6,2),SCDMIN,SCDMAX,SHP,WGHTP,SPCWTP,TRQOUT,MTHP(6,2),ISIZEP2MOD6550
3(3)
INITIALIZATION
DATA KHard/160,200,240,300,360,400,200,240,300,360,400,640/,M/0/
NRED2=2*NRED
NRED4=4*NRED
DO 10 II=1,NRED
DO 10 JJ=1,2
M=M+1
I=IHARD(II,JJ)
MHard(M)=KHard(I,1)
M=M+1
MHard(M)=KHard(I,2)
PRINT OUTPUT
DO 20 J=1,NDIFF
WRITE(6,30)
IF ((NPWRIN.EQ.2).AND.(NDIFF.EQ.1)) WRITE(6,70)
IF (IPWRSR(J).EQ.1) WRITE(6,40) J
IF (IPWRSR(J).EQ.2) WRITE(6,50) J
WRITE(6,60) PWRIN(J),RPMIN(J)
WRITE(6,80) NPWRIN,NPATH,NRED
WRITE(6,90) SHP,RPMOUT,MGOP(J),TRQOUT
IF (NPWRIN.EQ.2) WRITE(6,100) SCDMIN,SCDMAX

```

C
C
C

10
C
C
C

2MOD6820
2MOD6830
2MOD6840
2MOD6850
2MOD6860
2MOD6870
2MOD6880
2MOD6890
2MOD6900
2MOD6910
2MOD6920
2MOD6930
2MOD6940
2MOD6950
2MOD6960
2MOD6970
2MOD6980
2MOD6990
2MOD7000
2MOD7010
2MOD7020
2MOD7030
2MOD7040
2MOD7050
2MOD7060
2MOD7070
2MOD7080
2MOD7090
2MOD7100
2MOD7110
2MOD7120
2MOD7130
2MOD7140
2MOD7150
2MOD7160
2MOD7170

```

WRITE (6,110) WHTP,SPCWT,(ISIZEP(I),I=1,3)
IF (NRED.EQ.1) WRITE (6,120)
IF (NRED.EQ.2) WRITE (6,130)
IF (NRED.EQ.3) WRITE (6,140)
WRITE (6,150) (PWRP(I,J),I=1,NRED2)
WRITE (6,160) (RPMP(I,J),I=1,NRED2)
WRITE (6,170) (MTHP(I,J),I=1,NRED2)
WRITE (6,180) (PND(I),I=1,NRED)
WRITE (6,190) (PD(I),I=1,NRED)
WRITE (6,200) (DPHN(I),I=1,NRED)
WRITE (6,210) (DPHI(I),I=1,NRED)
WRITE (6,220) (DHELIX(I),I=1,NRED)
WRITE (6,230) (MGP(I,J),I=1,NRED)
WRITE (6,240) (PDAMP(I,J),I=1,NRED2)
WRITE (6,250) (FACEP(I,J),I=1,NRED)
WRITE (6,260) (FBYDP(I,J),I=1,NRED)
WRITE (6,270) (CDP(I,J),I=1,NRED)
WRITE (6,280) (PLVP(I,J),I=1,NRED)
WRITE (6,290) (WTP(I,J),I=1,NRED2)
WRITE (6,300) (TLPIP(I,J),I=1,NRED2)
WRITE (6,310) (UNTLDP(I,J),I=1,NRED2)
WRITE (6,320) (MFP(I,J),I=1,NRED)
WRITE (6,330) (KFCTRP(I,J),I=1,NRED2)
WRITE (6,340) (SIGHP(I,J),I=1,NRED)
WRITE (6,350) (SIGBP(I,J),I=1,NRED2)
WRITE (6,360) (TORQP(I,J),I=1,NRED2)
WRITE (6,370) (MHARD(I),I=1,NRED4)
CONTINUE
WRITE (6,30)
RETURN

```

FORMAT STATEMENTS

```

FORMAT (/,1X,72(1H*),/)
FOENAT (2X,12HPower SOURCE,I2,19H: TURBINE OR MOTOR)

```

20
C
C
C
C
30
40

```

50  FORMAT (2X,12HPOWER SOURCE,I2,43H:  MULTICYLINDER INTERNAL COMBUST2MOD7180
    1ION ENGINE) 2MOD7190
60  FORMAT (6X,18HINPUT POWER (HP): ,F7.0,4X,19HINPUT SPEED (RPM): ,F62MOD7200
    1.0,/) 2MOD7210
70  FORMAT (2X,57HNOTE: POWER SOURCES 1 AND 2 ARE IDENTICAL, THEREFORE2MOD7220
    1, THE,/,2X,56HTABULATED INFORMATION BELOW APPLIES TO EACH POWER TR2MOD7230
    2AIN,./) 2MOD7240
80  FORMAT (2X,27HARRANGEMENT: PARALLEL AXIS,,I2,10H INPUT(S),,I2,15H 2MOD7250
    1POWER PATH(S),,I2,13H REDUCTION(S)) 2MOD7260
90  FORMAT (6X,19HOUTPUT POWER (HP): ,F8.1,3X,20HOUTPUT SPEED (RPM): ,2MOD7270
    1F5.0,/,6X,7HRRATIO: ,F6.3,16X,25HOUTPUT TORQUE (K IN-LB): ,F8.1,/) 2MOD7280
100  FORMAT (2X,34HSOURCE CENTER DISTANCE (IN): MIN= ,F5.1,3X,5HMAX= ,F2MOD7290
    15.1,/) 2MOD7300
110  FORMAT (2X,46HSIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:,,/6X,2MOD7310
    113HWEIGHT (LB): ,F7.0,5X,25HSPECIFIC WEIGHT (LB/HP): ,F6.2,/,6X,132MOD7320
    2HLENGTH (IN): ,I3,3X,12HWIDTH (IN): ,I3,3X,13HHEIGHT (IN): ,I3,/,2MOD7330
    120  FORMAT (27X,11HREDUCTION 1,/,24X,17H|PINION | GEAR 1,/,24X,1H|,152MOD7340
    1(1H-),1H|) 2MOD7350
130  FORMAT (27X,11HREDUCTION 1,5X,11HREDUCTION 2,/,24X,33H|PINION | GE2MOD7360
    1AR |PINION | GEAR 1,/,24X,1H|,15(1H-),1H|) 2MOD7370
140  FORMAT (27X,11HREDUCTION 1,5X,11HREDUCTION 2,5X,11HREDUCTION 3,/,22MOD7380
    14X,49H|PINION | GEAR |PINION | GEAR |PINION | GEAR 1,/,24X,1H|,2MOD7390
    215(1H-),1H|,15(1H-),1H|,15(1H-),1H|) 2MOD7400
150  FORMAT (1X,24HPOWER SPLIT HP 1,6(F7.0,1H|)) 2MOD7410
160  FORMAT (1X,24HSPEED RPM 1,3(F6.0,1X,1H|,1X,F6.0,1H|)) 2MOD7420
170  FORMAT (1X,24HNUMBER OF TEETH 1,6(2X,I4,1X,1H|)) 2MOD7430
180  FORMAT (1X,24HNORMAL DIAMETRAL PITCH 1,3(5X,F6.3,4X,1H|)) 2MOD7440
190  FORMAT (1X,24HTRANS. DIAMETRAL PITCH 1,3(5X,F6.3,4X,1H|)) 2MOD7450
200  FORMAT (1X,24HNORMAL PRESSURE ANGLE 1,3(6X,F4.1,5X,1H|)) 2MOD7460
210  FORMAT (1X,24HTRANS. PRESSURE ANGLE 1,3(6X,F4.1,5X,1H|)) 2MOD7470
220  FORMAT (1X,24HHELIX ANGLE 1,3(6X,F4.1,5X,1H|)) 2MOD7480
230  FORMAT (1X,24HGEAR RATIO 1,3(5X,F6.3,4X,1H|)) 2MOD7490
240  FORMAT (1X,24HPITCH DIAMETER IN 1,3(F6.2,1X,1H|,1X,F6.2,1H|)) 2MOD7500
250  FORMAT (1X,24HEFFECTIVE FACEWIDTH IN 1,3(5X,F5.2,5X,1H|)) 2MOD7510
260  FORMAT (1X,24HF/DP 1,3(6X,F4.2,5X,1H|)) 2MOD7520
270  FORMAT (1X,24HCENTER DISTANCE IN 1,3(5X,F6.2,4X,1H|)) 2MOD7530

```

```

280  FORMAT (1X,24HPITCHLINE VELOCITY FPM 1,3(5X,F6.0,4X,1H1)) 2MOD7540
290  FORMAT (1X,24HTANGENTIAL LOAD LB 1,3(F6.0,1X,1H1,1X,F6.0,1H1)) 2MOD7550
300  FORMAT (1X,24HTOOTH LOAD/IN LB/IN 1,6(1X,F5.0,1X,1H1)) 2MOD7560
310  FORMAT (1X,24HUNIT LOAD PSI 1,3(F6.0,1X,1H1,1X,F6.0,1H1)) 2MOD7570
320  FORMAT (1X,24HMESH FREQUENCY HZ 1,3(5X,F6.0,4X,1H1)) 2MOD7580
330  FORMAT (1X,24HK FACTOR (COMPUTED) 1,6(1X,F5.0,1X,1H1)) 2MOD7590
340  FORMAT (1X,24HCONTACT STRESS PSI 1,3(4X,F7.0,4X,1H1)) 2MOD7600
350  FORMAT (1X,24HBENDING STRESS PSI 1,6(F7.0,1H1)) 2MOD7610
360  FORMAT (1X,24HTORQUE K IN-LB 1,6(F7.1,1H1)) 2MOD7620
370  FORMAT (1X,24HHARDNESS RANGE BHN 1,6(I3,1H-,I3,1H1)) 2MOD7630
      END 2MOD7640

```

```

SUBROUTINE EPCANL
CODED BY:  LT J.L. PAQUETTE, USN      JAN 1982
          NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940

SUBPROGRAM TO PERFORM AN ANALYSIS OF A GIVEN EPICYCLIC GEAR SET

EXTERNAL SUBPROGRAM(S) REQUIRED: SUBROUTINE GPI, SUBROUTINE GPJ,
FUNCTION AGMAE1, FUNCTION ARCCOS, FUNCTION ARCSIN, FUNCTION FALFA,
FUNCTION RTFNDR, FUNCTION SHRLD, FUNCTION THICK

REAL MGOE,MGE,MG1
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IIEPIC(3),IHARD(3)
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
COMMON /DESEPC/ MGOE,MGE(3),RPMI(3),RPMPL(3),RPMO(3),PWRE(3),DS(3)
1,DPLN(3),DR(3),FACEE(3),GI(3),GJS(3),GJPL(3),NS(3),NPLN(3),NR(3)
3MOD0180
3MOD0190
3MOD0200
3MOD0210
3MOD0220
3MOD0230
3MOD0240
3MOD0250
3MOD0260
3MOD0270
3MOD0280
3MOD0290
3MOD0300
3MOD0310
3MOD0320
3MOD0330

ENTER REQUIRED INFORMATION: TOOTH NUMBERS, DIAMETERS, FACEWIDTHS

DO 10 I=1,NRED
WRITE (6,30) I
WRITE (6,40)
READ (5,*) DS(I),DPLN(I),DR(I)
WRITE (6,50)
READ (5,*) FACEE(I)
CONTINUE

10
COMPUTE RATIOS, SPEED AND POWER SPLITS, AND GEOMETRY FACTORS

RPM1=RPMIN(1)
MGOE=RPMIN(1)/RPMOUT
DO 20 I=1,NRED

```

```

20      NS(I)=INT(PD(I)*DS(I)+.5)
      NPLN(I)=INT(PD(I)*DPLN(I)+.5)
      NR(I)=INT(PD(I)*DR(I)+.5)
      MGE(I)=DR(I)/DS(I)
      IF (IEPIC(I).EQ.1) MGE(I)=MGE(I)+1.
      RPMI(I)=RPM1
      KPMO(I)=RPM1/MGE(I)
      RPM1=RPMO(I)
      RPML(I)=RPM1*DR(I)/DPLN(I)
      PWRE(I)=PWRLN(1)/NPLNT(I)
      MG1=DPLN(I)/DS(I)
      CALL GFI (GI(I),I,MG1,DS(I),DPLN(I),0)
      CALL GPJ (GJS(I),I,DS(I),DPLN(I),1,0)
      CALL GPJ (GJPL(I),I,DS(I),DPLN(I),2,0)
      CONTINUE
      RETURN
20
C
C
C
C
30      FORMAT (//,4X,54HTHE INFORMATION REQUESTED BELOW IS FOR REDUCTION
      1STAGE,I2,1H.)
40      FORMAT (//,1X,61H** ENTER DIAMETERS, IN INCHES, OF SUN, PLANET, AND
      1 RING GEARS,/,1X,18H (DS, DPLN, DR):)
50      FORMAT (//,1X,39H** ENTER FACEWIDTH OF GEARS, IN INCHES:)
      END
C
C
C
C
      SUBROUTINE EPCDES
C
C
C
C
      CODED BY:  LT J.L. PAQUETTE, USN          JAN 1982
                NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
C
C
C
C
      SUBPROGRAM TO PERFORM DESIGN CALCULATIONS FOR EPICYCLIC
      REDUCTION GEARS USING A BASIC RANDOM SEARCH OPTIMIZATION
C

```


[illegible]

3MOD1060
3MOD1070
3MOD1080
3MOD1090
3MOD1100
3MOD1110
3MOD1120
3MOD1130
3MOD1140
3MOD1150
3MOD1160
3MOD1170
3MOD1180
3MOD1190
3MOD1200
3MOD1210
3MOD1220
3MOD1230
3MOD1240
3MOD1250
3MOD1260
3MOD1270
3MOD1280
3MOD1290
3MOD1300
3MOD1310
3MOD1320
3MOD1330
3MOD1340
3MOD1350
3MOD1360
3MOD1370
3MOD1380
3MOD1390
3MOD1400
3MOD1410

```

NDV=NRD3
MM=10*NDV
IF (NHELX.EQ.2) FDP=2.25
IF (NHELX.EQ.3) SCALE(3)=75.
IRET=1
WRITE (6,340)
READ (5,*) RND
RND=NRDGEN(RND)

C
C COMPUTE THE OVERALL GEAR RATIO, INITIAL STAGE GEAR RATIOS,
C AND POWER AND SPEED SPLITS
C
MGOE=RPMIN(1)/RPMOUT
MGMAX=8.*NRD
IF (MGOE.GT.MGMAX) GO TO 320
RPM1=RPMIN(1)
DO 20 I=1,NRED
  MGQ(I)=MGOE**E
  SPD(I)=RPM1
  RPM1=SPD(I)/MGQ(I)
  HP(I,1)=PWRIN(1)
  HP(I,2)=HP(I,1)/NPLNT(I)
CONTINUE

20
C
C ESTIMATE INITIAL DESIGN AS START POINT FOR OPTIMIZATION
C
DO 30 I=1,NRED
  MG1=(1.+RND)*1.5
  IH=IHARD(I,1)
  BRAC=SAC(IH)*1.E-04/CR(IH)
  KK=BRAC*BRAC*3.36/(CO(IOPRO)*CM(IOPRO))
  ANUM=126050.*HP(I,1)*(MG1+1.)
  DEN=SPD(I)*FDP*KK*MG1
  DSQ(I)=(ANUM/DEN)**E3
  FACEQ(I)=FDP*DSQ(I)
CONTINUE

30

```

```

C
C      COMPUTE VALUES OF DEPENDENT VARIABLES
C
40      GO TO (50,60,70), NKED
C ***  SINGLE REDUCTION
50      MGQ(1)=MGOE
      GO TO 80
C ***  DOUBLE REDUCTION
60      MGQ(1)=MGOE/MGQ(2)
      GO TO 80
C ***  TRIPLE REDUCTION
70      MGQ(1)=MGOE/(MGQ(2)*MGQ(3))
80      RPM1=RPMIN(1)
      DO 130 I=1,NRED
      PNSQ=PD(I)*DSQ(I)
      NSQ(I)=INT(PNSQ+.5)
      IEP=IEPIC(I)
      GO TO (90,100), IEP
C ***  PLANETARY GEAR CONFIGURATION
90      PNRQ=FLOAT(NSQ(I))*MGQ(I)-1.)
      NRQ(I)=INT(PNRQ+.5)
      PKCON=FLOAT(NRQ(I))*MGQ(I)/(FLOAT(NPLNT(I))*MGQ(I)-1.)
      KCON=INT(PKCON+.5)
      GO TO 110
C ***  STAR GEAR ARRANGEMENT
100     PNRQ=FLOAT(NSQ(I))*MGQ(I)
      NRQ(I)=INT(PNRQ+.5)
      PKCON=FLOAT(NRQ(I))*MGQ(I)+1.)/(MGQ(I)*FLOAT(NPLNT(I)))
      KCON=INT(PKCON+.5)
110     NRQ(I)=KCON*NPLNT(I)-NSQ(I)
      PLNTQ=(FLOAT(NRQ(I))-FLOAT(NSQ(I)))/2.
      IF (PLNTQ.EQ.AINT(PLNTQ)) GO TO 120
      KCON=KCON+1
      GO TO 110
120     NPLNQ(I)=INT(PLNTQ)
      MGQ(I)=FLOAT(NRQ(I))/FLOAT(NSQ(I))

```

```

3MOD1420
3MOD1430
3MOD1440
3MOD1450
3MOD1460
3MOD1470
3MOD1480
3MOD1490
3MOD1500
3MOD1510
3MOD1520
3MOD1530
3MOD1540
3MOD1550
3MOD1560
3MOD1570
3MOD1580
3MOD1590
3MOD1600
3MOD1610
3MOD1620
3MOD1630
3MOD1640
3MOD1650
3MOD1660
3MOD1670
3MOD1680
3MOD1690
3MOD1700
3MOD1710
3MOD1720
3MOD1730
3MOD1740
3MOD1750
3MOD1760
3MOD1770

```



```

180      CONTINUE
      IF (IRET.EQ.2) GO TO 230
C
C      PERFORM LOCAL RANDOM SEARCHES NEAR INITIAL/MOST RECENT DESIGN
C
      IRET=2
      H=H+1
      IF (H.LT.MM) GO TO 200
      ALPHA=BB*ALPHA
      IF (ALPHA.LT.1.E-04) GO TO 290
      H=0
      SMAX=-1.E+10
      IS=0
      DO 210 JJ=1,NRED
      DO 210 II=1,3
      IS=IS+1
      RND=RDGEN(RND)
      S(IS)=(2.*RND-1.)*SCALE(II)
      SMAX=AMAX1(SMAX,ABS(S(IS)))
      DO 220 IS=1,NRED3
      S(IS)=S(IS)/SMAX
      KS=0
      L=0
      DO 240 II=1,NRED
      L=L+1
      IF (FLAGG) S(L)=ABS(S(L))
      DSQ(II)=DS(II)+ALPHA*S(L)
      L=L+1
      IF (FLAGG) S(L)=ABS(S(L))
      HGQ(II)=HGE(II)+ALPHA*S(L)
      L=L+1
      FACEQ(II)=FACEE(II)+ALPHA*S(L)
      GO TO 40
      IQ=IQ+1
      IF (IQ.GT.IQMAX) GO TO 290
      IF (GMAX.GT.0.0) GO TO 280
C
240
250
3MOD2500
3MOD2510
3MOD2520
3MOD2530
3MOD2540
3MOD2550
3MOD2560
3MOD2570
3MOD2580
3MOD2590
3MOD2600
3MOD2610
3MOD2620
3MOD2630
3MOD2640
3MOD2650
3MOD2660
3MOD2670
3MOD2680
3MOD2690
3MOD2700
3MOD2710
3MOD2720
3MOD2730
3MOD2740
3MOD2750
3MOD2760
3MOD2770
3MOD2780
3MOD2790
3MOD2800
3MOD2810
3MOD2820
3MOD2830
3MOD2840
3MOD2850

```

260	IF (VQ.LT.VSTR) GO TO 170	3MOD2870
	IF (KS.EQ.1) GO TO 190	3MOD2880
270	DO 270 IS=1,NRED3	3MOD2890
	S(IS)=-S(IS)	3MOD2900
	KS=1	3MOD2910
	GO TO 230	3MOD2920
280	IF (GMAX.GT.GMXSTR) GO TO 260	3MOD2930
	GMXSTR=GMAX	3MOD2940
	GO TO 170	3MOD2950
C		3MOD2960
C	COMPUTE ACTUAL OVERALL GEAR RATIOS AND SPEEDS TO BE USED	3MOD2970
C		3MOD2980
290	MGOE=1.	3MOD2990
	DO 300 I=1,NRED	3MOD3000
300	MGOE=MGOE*MGE(I)	3MOD3010
C		3MOD3020
C	END OF DESIGN ITERATIONS	3MOD3030
C		3MOD3040
	IF (FLAG) GO TO 310	3MOD3050
	RETURN	3MOD3060
C		3MOD3070
C	ERROR CONDITION HANDLING	3MOD3080
C		3MOD3090
310	WRITE (6,330)	3MOD3100
	RETURN	3MOD3110
320	NP1=NRED+1	3MOD3120
	WRITE (6,350) MGOE,NRED,NP1	3MOD3130
	NRED=NP1	3MOD3140
	GO TO 10	3MOD3150
C		3MOD3160
C	FORMAT STATEMENTS	3MOD3170
C		3MOD3180
C		3MOD3190
330	FORMAT (//,4X,23H*****,WARNING *****,//,4X,54HSIZE AND/OR ALLOW3MOD3200	3MOD3200
		3MOD3210

```

1ABLE POWER CONSTRAINTS WERE VIOLATED.,/,4X,32HTHIS DESIGN MAY NOT 3MOD3220
2BE FEASIBLE.,/,4X,34H*** PROGRAM CONTINUING ***//) 3MOD3230
340 FORMAT (/,2X,49H* ENTER SEED FOR RANDOM NUMBER GENERATOR (X.XX):3MOD3240
1) 3MOD3250
350 FORMAT (/,4X,28HTHE OVERALL REDUCTION RATIO,,F7.3,17H, IS TO LARG3MOD3260
1E FOR,12,,4X,30HREDUCTION STAGE(S); THEREFORE,,12,33H REDUCTION 3MOD3270
2TAGE(S) WILL BE USED.) 3MOD3280
END 3MOD3290
3MOD3300
C 3MOD3310
C 3MOD3320
C 3MOD3330
C 3MOD3340
C 3MOD3350
C 3MOD3360
C 3MOD3370
C 3MOD3380
C 3MOD3390
C 3MOD3400
C 3MOD3410
C 3MOD3420
C 3MOD3430
C 3MOD3440
C 3MOD3450
C 3MOD3460
C 3MOD3470
C 3MOD3480
C 3MOD3490
C 3MOD3500
C 3MOD3510
C 3MOD3520
C 3MOD3530
C 3MOD3540
C 3MOD3550
C 3MOD3560
C 3MOD3570

SUBROUTINE EPCRES
CODED BY: LT J.L. PAQUETTE, USN JAN 1982
NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
SUBPROGRAM TO COMPUTE ALL OUTPUT PARAMETERS FOR EPICYCLIC GEARS
REAL MGOE,MGE,MFE,KFCTRE,MG1
COMMON /AGMAB/ SFB(2,2),AKV,AKS,AKM,AKO(2),SAT(6),AKL(2),AKR(6),AK 3MOD3410
1T 3MOD3420
COMMON /AGMAH/ SFH(2,2),CV(3),CS,CM(2),CF,CO(2),SAC(6),CP,CL(2),CH3MOD3430
1,CT,CR(6) 3MOD3440
COMMON /DESDAT/ PWRIN(2),REFIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),3MOD3450
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(33MOD3460
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX 3MOD3470
COMMON /DESEPC/ MGOE,MGE(3),RPMI(3),RPMPL(3),RPMO(3),PWRE(3),DS(3) 3MOD3480
1,DPLN(3),DR(3),FACEE(3),GI(3),GJS(3),GJPL(3),NS(3),NPLN(3),NR(3) 3MOD3490
COMMON /RESEPC/ PLVE(3),FBYDE(3),CDE(3),WTE(3),TLPIE(3),UNTLDE(3),3MOD3500
1MFE(3,3),KFCTRE(3),SIGHE(3),SIGBE(3),TORQE(3,3),RPME(3,3),PDIAME(33MOD3510
2,3),WGHTe,SPCWTE,MTHE(3,3),ISIZEE(3) 3MOD3520
3MOD3530
INITIALIZE 3MOD3540
3MOD3550
PI=4.*ATAN(1.) 3MOD3560
3MOD3570

```



```

C
C
      COMPUTE ALL OUTPUT PARAMETERS

      DO 40 I=1,NRED
      PLVE(I)=PI*DS(I)*RPMI(I)/12.
      FBVDE(I)=FACEE(I)/DS(I)
      CDE(I)=(DS(I)+DPLN(I))/2.
      WTE(I)=126050.*PWRIN(1)/(RPMI(I)*DS(I))
      TLPTE(I)=WTE(I)/FACEE(I)
      UNTLDE(I)=TLPTE(I)*PND(I)
      ANPLNT=FLOAT(NPLNT(I))
      ANR=FLOAT(NR(I))
      ANPLN=FLOAT(NPLN(I))
      ANS=FLOAT(NS(I))
      IE=IEPIC(I)
      GO TO (10,20), IE
10    ANRPNS=ANR+ANS
      MFE(I,1)=ANPLNT*ANR*RPMI(I)/ANRPNS
      MFE(I,2)=(ANR/ANPLN)*ANS*RPMI(I)/ANRPNS
      MFE(I,3)=ANPLNT*ANS*RPMI(I)/ANRPNS
      GO TO 30
20    MFE(I,1)=ANPLNT*RPMI(I)
      MFE(I,2)=2.*ANS*RPMI(I)/ANPLN
      MFE(I,3)=ANPLNT*ANS*RPMI(I)/ANR
      MG1=DPLN(I)/DS(I)
      KPCTRE(I)=WTE(I)*(MG1+1.)/(FACEE(I)*DS(I)*MG1)
      C1=WTE(I)*CO(IOPRO)/CV(1)
      C2=CS/DS(I)*FACEE(I)
      C3=CH(ICPRO)*CF/GI(I)
      SIGHE(I)=CP*SQRT(C1*C2*C3)
      C1=WTE(I)*AKO(2)/AKV
      C2=PD(I)/FACEE(I)
      C3=AKS*AKH/AMIN1(GJS(I),GJPL(I))
      SIGBE(I)=C1*C2*C3
      TW=WTE(I)/2000.
      TORQE(I,1)=TW*DS(I)
      TORQE(I,2)=TW*DPLN(I)
3MOD3580
3MOD3590
3MOD3600
3MOD3610
3MOD3620
3MOD3630
3MOD3640
3MOD3650
3MOD3660
3MOD3670
3MOD3680
3MOD3690
3MOD3700
3MOD3710
3MOD3720
3MOD3730
3MOD3740
3MOD3750
3MOD3760
3MOD3770
3MOD3780
3MOD3790
3MOD3800
3MOD3810
3MOD3820
3MOD3830
3MOD3840
3MOD3850
3MOD3860
3MOD3870
3MOD3880
3MOD3890
3MOD3900
3MOD3910
3MOD3920
3MOD3930

```



```

COMMON /RESEPC/ PLVE(3), FBYDE(3), CDE(3), WTE(3), TLP(3), UNTLDE(3), 3MOD4300
1MFE(3,3), KFCRTRE(3), SIGHE(3), SIGBE(3), TORQE(3,3), RPME(3,3), PDIAME(33MOD4310
2,3), WGTE, SPCWTE, MTHE(3,3), ISIZEE(3) 3MOD4320
C 3MOD4330
C 3MOD4340
C 3MOD4350
C 3MOD4360
C 3MOD4370
C 3MOD4380
C 3MOD4390
C 3MOD4400
C 3MOD4410
C 3MOD4420
C 3MOD4430
C 3MOD4440
C 3MOD4450
C 3MOD4460
C 3MOD4470
C 3MOD4480
C 3MOD4490
C 3MOD4500
C 3MOD4510
C 3MOD4520
C 3MOD4530
C 3MOD4540
C 3MOD4550
C 3MOD4560
C 3MOD4570
C 3MOD4580
C 3MOD4590
C 3MOD4600
C 3MOD4610
C 3MOD4620
C 3MOD4630
C 3MOD4640
C 3MOD4650

INITIALIZATION
D2P=0.0
SP=0.0
PNP=FLOAT(NPATH)
DRMAX=-1.E-04

COMPUTE WEIGHT ESTIMATE
DO 10 I=1,NRED
DRMAX=AMAX1(DRMAX, DR(I))
SF=SF+FACEE(I)
D2F1=DS(I)*DS(I)*FACEE(I)
D2F2=NPLNT(I)*DPLN(I)*DPLN(I)*FACEE(I)
USE 0.7 DR TO ACCOUNT FOR THE CARRIER
D2F3=.49*DR(I)*DR(I)*FACEE(I)
D2F=D2F1+D2F2+D2F3
WGTE=.905*(D2P)**0.89
IP=INT(ALOG10(WGTE))-2
WGTE=AINT(WGTE/(10.**IP))*(10.**IP)
SPCWTE=WGTE/PWRIN(1)

DIMENSIONS ESTIMATE
ISIZEE(1)=INT(2.85*SP+.5)
ISIZEE(2)=INT(1.30*DRMAX+.5)
ISIZEE(3)=INT(1.20*DRMAX+.5)
RETURN
END

```

```

SUBROUTINE EPCOUT
CODED BY:  LT J.L. PAQUETTE, USN          JAN 1982
          NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940

SUBPROGRAM TO PRESENT ALL RESULTS FROM THE DESIGN/ANALYSIS
FOR EPICYCLIC GEARS

REAL MGOF,MGE,MFE,KFCTRE
DIMENSION KHard(6,2),MHard(4)
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3)
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
COMMON /DESEPC/ MGOF,MGE(3),RPMI(3),RPMPL(3),RPMO(3),PWRE(3),DS(3)
1,DPLN(3),DR(3),FACEE(3),GI(3),GJS(3),GJPL(3),NS(3),NPLN(3),NR(3)
COMMON /RESEPC/ PLVE(3),FBYDE(3),CDE(3),WTE(3),TLPIE(3),UNTLDE(3),
1MFE(3,3),KFCTRE(3),SIGHE(3),SIGBE(3),TORQE(3,3),RPME(3,3),PDIAHE(3)
2,3),WGHE,SPCWTE,MTHE(3,3),ISIZEE(3)

INITIALIZATION

DATA KHard/160,200,240,300,360,400,200,240,300,360,400,640/

PRINT OUTPUT

WRITE (6,30)
IF (IPWRSR(1).EQ.1) WRITE (6,40)
IF (IPWRSR(1).EQ.2) WRITE (6,50)
WRITE (6,60) PWRIN(1),RPMIN(1)
WRITE (6,70) NRED
WRITE (6,80) PWRIN(1),RPMOUT,MGOF,TORQE(NRED,3)
WRITE (6,90) WGHE,SPCWTE,(ISIZEE(I),I=1,3)
DO 20 I=1,NRED
WRITE (6,30)
M=0
DO 10 J=1,2

```

C
C
C
C
C
C
C

C
C
C
C
C
C

3MOD4660
3MOD4670
3MOD4680
3MOD4690
3MOD4700
3MOD4710
3MOD4720
3MOD4730
3MOD4740
3MOD4750
3MOD4760
3MOD4770
3MOD4780
3MOD4790
3MOD4800
3MOD4810
3MOD4820
3MOD4830
3MOD4840
3MOD4850
3MOD4860
3MOD4870
3MOD4880
3MOD4890
3MOD4900
3MOD4910
3MOD4920
3MOD4930
3MOD4940
3MOD4950
3MOD4960
3MOD4970
3MOD4980
3MOD4990
3MOD5000
3MOD5010

3MOD5020
3MOD5030
3MOD5040
3MOD5050
3MOD5060
3MOD5070
3MOD5080
3MOD5090
3MOD5100
3MOD5110
3MOD5120
3MOD5130
3MOD5140
3MOD5150
3MOD5160
3MOD5170
3MOD5180
3MOD5190
3MOD5200
3MOD5210
3MOD5220
3MOD5230
3MOD5240
3MOD5250
3MOD5260
3MOD5270
3MOD5280
3MOD5290
3MOD5300
3MOD5310
3MOD5320
3MOD5330
3MOD5340
3MOD5350
3MOD5360
3MOD5370

```

10  IH=IHARD(I,J)
    M=M+1
    MHARD(M)=KHARD(IH,1)
    M=M+1
    MHARD(M)=KHARD(IH,2)
    WRITE(6,100) I
    IF (IEPIC(I).EQ.1) WRITE(6,110)
    IF (IEPIC(I).EQ.2) WRITE(6,120)
    WRITE(6,130) NPLNT(I)
    WRITE(6,140) PWRIN(1),PWRE(I),PWRIN(1)
    WRITE(6,150) (RPME(I,J),J=1,3)
    WRITE(6,160) (MTHE(I,J),J=1,3)
    WRITE(6,170) PND(I)
    WRITE(6,180) PD(I)
    WRITE(6,190) DPHIN(I)
    WRITE(6,200) DPHI(I)
    WRITE(6,210) DHELIX(I)
    WRITE(6,220) MGE(I)
    WRITE(6,230) (PDIAHE(I,J),J=1,3)
    WRITE(6,240) FACEE(I)
    WRITE(6,250) FBIDE(I)
    WRITE(6,260) CDE(I)
    WRITE(6,270) PLVE(I)
    WRITE(6,280) WTE(I)
    WRITE(6,290) TLPIE(I)
    WRITE(6,300) UNTLDE(I)
    WRITE(6,310) (HFE(I,J),J=1,3)
    WRITE(6,320) KPCTRE(I)
    WRITE(6,330) SIGHE(I)
    WRITE(6,340) SIGBE(I)
    WRITE(6,350) (TORQE(I,J),J=1,3)
    WRITE(6,360) (MHARD(J),J=1,2),(MHARD(J),J=1,4)
    CONTINUE
    WRITE(6,30)
    RETURN
20
C

```

C	FORMAT STATEMENTS			3MOD5380
C				3MOD5390
C				3MOD5400
30	FORMAT (//,1X,72(1H*),/)			3MOD5410
40	FORMAT (2X,31HPower SOURCE: TURBINE OR MOTOR)			3MOD5420
50	FORMAT (2X,55HPower SOURCE: MULTICYLINDER INTERNAL COMBUSTION ENG			3MOD5430
	1INE)			3MOD5440
60	FORMAT (6X,18INPUT POWER (HP): ,F7.0,4X,19INPUT SPEED (RPM): ,F6			3MOD5450
	1.0,/)			3MOD5460
70	FORMAT (2X,23HARRANGEMENT: EPICYCLIC,,I2,13H REDUCTION(S))			3MOD5470
80	FORMAT (6X,19HOUTPUT POWER (HP): ,F7.0,3X,20HOUTPUT SPEED (RPM): ,			3MOD5480
	1F5.0,/6X,7HRATIO: ,F6.3,16X,25HOUTPUT TORQUE (K IN-LB): ,F8.1,/)			3MOD5490
90	FORMAT (2X,46HSIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:,,/6X,3MOD5500			3MOD5510
	113HHEIGHT (LB): ,F7.0,5X,25HSPECIFIC WEIGHT (LB/HP): ,F6.2,/6X,133MOD5510			3MOD5520
	2HLENGTH (IN): ,I3,3X,12HWIDTH (IN): ,I3,3X,13HHEIGHT (IN): ,I3,/)			3MOD5530
100	FORMAT (43X,9HREDUCTION,I2,/24X,1H,6X,3HSUN,6X,1H,4X,7HPLANETS,3MOD5530			3MOD5540
	14X,1H,3X,9HRING-CAGE,3X,1H,/,24X,1H,15(1H-),1H,15(3MOD5540			3MOD5550
	21H-),1H))			3MOD5560
110	FORMAT (1X,24HGEAR ARRANGEMENT		1,19X,9HPLANETARY,19X,1H))	3MOD5570
120	FORMAT (1X,24HGEAR ARRANGEMENT		1,22X,4HSTAR,21X,1H))	3MOD5580
130	FORMAT (1X,24HNUMBER OF PLANETS		1,23X,I1,23X,1H))	3MOD5590
140	FORMAT (1X,24HPower SPLIT	HP	1,3(4X,F7.0,4X,1H))	3MOD5600
150	FORMAT (1X,24HSPEED	RPM	1,3(4X,F7.0,4X,1H))	3MOD5610
160	FORMAT (1X,24HNUMBER OF TEETH		1,3(6X,I4,5X,1H))	3MOD5620
170	FORMAT (1X,24HNORMAL DIAMETRAL PITCH		1,20X,F6.3,21X,1H))	3MOD5630
180	FORMAT (1X,24HTRANS. DIAMETRAL PITCH		1,20X,F6.3,21X,1H))	3MOD5640
190	FORMAT (1X,24HNORMAL PRESSURE ANGLE		1,21X,F4.1,22X,1H))	3MOD5650
200	FORMAT (1X,24HTRANS. PRESSURE ANGLE		1,21X,F4.1,22X,1H))	3MOD5660
210	FORMAT (1X,24HHELIX ANGLE		1,21X,F4.1,22X,1H))	3MOD5670
220	FORMAT (1X,24HGEAR RATIO		1,20X,F6.3,21X,1H))	3MOD5680
230	FORMAT (1X,24HPITCH DIAMETER	IN	1,3(4X,F6.2,5X,1H))	3MOD5690
240	FORMAT (1X,24HEFFECTIVE FACEWIDTH	IN	1,21X,F5.2,21X,1H))	3MOD5700
250	FORMAT (1X,24HP/Dp		1,21X,F4.2,22X,1H))	3MOD5710
260	FORMAT (1X,24HCENTER DISTANCE	IN	1,20X,F6.2,21X,1H))	3MOD5720
270	FORMAT (1X,24HPITCHLINE VELOCITY FPM		1,20X,F6.0,21X,1H))	3MOD5730
280	FORMAT (1X,24HTANGENTIAL LOAD	LB	1,20X,F6.0,21X,1H))	

290	FORMAT	(1X,24HTOOTH LOAD/IN	LB/IN	1,21X,F5.0,21X,1H))	3MOD5740
300	FORMAT	(1X,24HUNIT LOAD	PSI	1,20X,F6.0,21X,1H))	3MOD5750
310	FORMAT	(1X,24HMESH FREQUENCY	HZ	1,3(4X,F6.0,5X,1H))	3MOD5760
320	FORMAT	(1X,24HK FACTOR (COMPUTED)		1,21X,F5.0,21X,1H))	3MOD5770
330	FORMAT	(1X,24HCONTACT STRESS	PSI	1,20X,F7.0,20X,1H))	3MOD5780
340	FORMAT	(1X,24HBENDING STRESS	PSI	1,20X,F7.0,20X,1H))	3MOD5790
350	FORMAT	(1X,24HTORQUE	K IN-LB	1,3(4X,F7.1,4X,1H))	3MOD5800
360	FORMAT	(1X,24HHARDNESS RANGE	BHN	1,3(2X,I3,5H - ,I3,2X,1H))	3MOD5810
	END				3MOD5820

Module Four

```

C      FUNCTION AGMAE1 (PHIN,II)
C
C      CODED BY:  LT J.L. PAQUETTE, USN      JAN 1982
C                NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
C
C      SUBPROGRAM TO PROVIDE INTERPOLATION OF TABLE E-1 IN
C      AGMA 226.01, AUG 1970 FOR VALUES OF H, L, AND M USED
C      TO COMPUTE KP -- LAGRANGE INTERPOLATION USED
C
C      PHIN  NORMAL PRESSURE ANGLE IN RADIANS
C      II=1  INTERPOLATION OF H -- F(1) TO F(3)
C      II=2  INTERPOLATION OF L -- F(4) TO F(6)
C      II=3  INTERPOLATION OF M -- F(7) TO F(9)
C
C      DIMENSION F(9),A(3)
C
C      INITIALIZATION:
C      ARRAY F CONTAINS THE VALUES OF H, L, M FROM TABLE E-1
C
C      DATA F/0.22,0.18,0.14,0.20,0.15,0.11,0.40,0.45,0.50/
C      SUM=0.0
C      L=0
C
C      CONVERT PHIN TO DEGREES FOR USE IN INTERPOLATION
C
C      X=PHIN*180./(4.*ATAN(1.))
C      A(1)=(X-20.)*(X-25.)/57.75
C      A(2)=(X-14.5)*(X-25.)/(-27.5)
C      A(3)=(X-14.5)*(X-20.)/52.5
C      I=(II-1)*3+1
C      J=3*II
C      DO 10 K=I,J
C      L=L+1

```



```

C 30
CNST (1) = XC
CNST (2) = YC
CNST (3) = RF
CNST (4) = RV
CNST (5) = FLOAT (I II)
ALPHA = RTFNDR (0., 0.7, FALFA, CNST, 0.0001)
CAPT = XC - RF * COS (ALPHA)
T = 2. * CAPT
H = RV - YC + RF * SIN (ALPHA)
IF (III.EQ.1) H = YC - RV + RF * SIN (ALPHA)
ARG = H / CAPT
GAMMA1 = (PI / 2.) - ATAN (ARG)
X = CAPT * TAN (GAMMA1)

C
C CSMH = AGMA HELICAL FACTOR; ANU = ARGUMENT FOR CSMH IN DEGREES
C AND DIVIDED BY 100
C
ARG = TAN (HELIX (IRED)) * SIN (PHIN (IRED))
ANU = ATAN (ARG) * 1.8 / PI
CSMH = 1. / (1. - SQRT (ANU * (1. - ANU)))

C
C YSMC = AGMA TOOTH FORM FACTOR
C
ARG = (1.5 / (X * CSMH)) - (TAN (PHILN) / T)
ARG1 = COS (PHILN) / COS (PHIN (IRED))
YSMC = PND (IRED) / (ARG * ARG1)

C
C KP = THEORETICAL STRESS CORRECTION FACTOR; AH, AL, AND AM ARE
C THE H, L, AND M CONSTANTS FROM AGMA 226.01, APPENDIX E, USED
C TO COMPUTE KF
C
AH = AGMAE1 (PHIN (IRED), 1)
AL = AGMAE1 (PHIN (IRED), 2)
AM = AGMAE1 (PHIN (IRED), 3)
RAT1 = T / RF

```

```

4MOD2500
4MOD2510
4MOD2520
4MOD2530
4MOD2540
4MOD2550
4MOD2560
4MOD2570
4MOD2580
4MOD2590
4MOD2600
4MOD2610
4MOD2620
4MOD2630
4MOD2640
4MOD2650
4MOD2660
4MOD2670
4MOD2680
4MOD2690
4MOD2700
4MOD2710
4MOD2720
4MOD2730
4MOD2740
4MOD2750
4MOD2760
4MOD2770
4MOD2780
4MOD2790
4MOD2800
4MOD2810
4MOD2820
4MOD2830
4MOD2840
4MOD2850

```



```

C MOD3220
C MOD3230
C MOD3240
C MOD3250
C MOD3260
C MOD3270
C MOD3280
C MOD3290
C MOD3300
C MOD3310
C MOD3320
C MOD3330
C MOD3340
C MOD3350
C MOD3360
C MOD3370
C MOD3380
C MOD3390
C MOD3400
C MOD3410
C MOD3420
C MOD3430
C MOD3440
C MOD3450
C MOD3460
C MOD3470
C MOD3480
C MOD3490
C MOD3500
C MOD3510
C MOD3520
C MOD3530
C MOD3540
C MOD3550
C MOD3560
C MOD3570

INITIALIZATION

IH=IHARD(IRED,IGR)
IP=IPWRSR(IPWR)
NPTH=NPETH
IF ((NRED.EQ.3) .AND. (IRED.GE.2)) NPTH=2

COMPUTE ALLOWABLE SERVICE POWER

ANUM=RPM*D*AKV*GEOMJ*SAT(IH)*AKL(IOPRO)
DEN=SPB(IOPRO,IP)*AKO(NPTH)*AKM*AKS*AKR(IH)*AKT*PD(IRED)
POWERB=FACE*ANUM/(126050.*DEN)
RETURN
END

FUNCTION POWERH (RPM,FACE,D,IRED,IPWR,GEOMI)
C MOD3400
C MOD3410
C MOD3420
C MOD3430
C MOD3440
C MOD3450
C MOD3460
C MOD3470
C MOD3480
C MOD3490
C MOD3500
C MOD3510
C MOD3520
C MOD3530
C MOD3540
C MOD3550
C MOD3560
C MOD3570

CODED BY: LT J.L. PAQUETTE, USN JAN 1982
          NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940

SUBPROGRAM TO COMPUTE THE ALLOWABLE SERVICE POWER OF A GEAR PAIR
BASED ON AGMA'S DURABILITY RATING (AGMA 211.02 SEPT 1966)

RPM PINION SPEED IN RPM
FACE FACEWIDTH OF THE GEAR PAIR IN INCHES
D PITCH DIAMETER OF THE PINION IN INCHES
IRED REDUCTION STAGE UNDER CONSIDERATION
IPWR POWER SOURCE IDENTIFICATION
GEOMI DURABILITY GEOMETRY FACTOR, I

COMMON /AGMAH/ SPH(2,2),CV(3),CS,CM(2),CP,CO(2),SAC(6),CP,CL(2),CH
1,CT,CR(6)
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),

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C C SUBPROGRAM TO FIND A ROOT OF F(X) IN THE INTERVAL AX,BX
C C
C C AX LEFT ENDPOINT OF INITIAL INTERVAL
C C BX RIGHT ENDPOINT OF INITIAL INTERVAL
C C F FUNCTION SUBPROGRAM FOR EVALUATION OF F(X)
C C CNST PARAMETER LIST OF CONSTANTS REQUIRED IN F(X)
C C TOL DESIRED LENGTH OF INTERVAL OF UNCERTAINTY OF
C C THE FINAL RESULT
C C
C C IT IS ASSUMED THAT F(AX) AND F(BX) HAVE OPPOSITE SIGNS
C C WITHOUT A CHECK. RTPNDR RETURNS A ROOT, X, IN THE
C C INTERVAL AX,BX TO WITHIN A TOLERANCE OF 4*EPS*ABS(X)+TOL
C C WHERE EPS IS THE RELATIVE MACHINE PRECISION.
C C
C C DIMENSION CNST(5)
C C
C C COMPUTE EPS, THE RELATIVE MACHINE PRECISION
C C
C C EPS= 1.0
C C EPS=EPS/2.0
C C TOL1=1.0+EPS
C C IF (TOL1.GT.1.0) GO TO 10
C C
C C INITIALIZATION
C C
C C A=AX
C C B=BX
C C FA=F(A,CNST)
C C FB=F(B,CNST)
C C
C C BEGIN STEP
C C
C C C=A
C C FC=FA
C C D=B-A
C C
C C 10
C C
C C 4MOD3940
C C 4MOD3950
C C 4MOD3960
C C 4MOD3970
C C 4MOD3980
C C 4MOD3990
C C 4MOD4000
C C 4MOD4010
C C 4MOD4020
C C 4MOD4030
C C 4MOD4040
C C 4MOD4050
C C 4MOD4060
C C 4MOD4070
C C 4MOD4080
C C 4MOD4090
C C 4MOD4100
C C 4MOD4110
C C 4MOD4120
C C 4MOD4130
C C 4MOD4140
C C 4MOD4150
C C 4MOD4160
C C 4MOD4170
C C 4MOD4180
C C 4MOD4190
C C 4MOD4200
C C 4MOD4210
C C 4MOD4220
C C 4MOD4230
C C 4MOD4240
C C 4MOD4250
C C 4MOD4260
C C 4MOD4270
C C 4MOD4280
C C 4MOD4290

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	S=PB/PA	4MOD4660
	P=S*(2.*YM*Q*(Q-R)-(E-A)*(R-1.))	4MOD4670
	Q=(Q-1.)*(R-1.)*(S-1.)	4MOD4680
C		4MOD4690
C	ADJUST SIGNS	4MOD4700
C		4MOD4710
60	IF (P.GT.0.0) Q=-Q	4MOD4720
	P=ABS(P)	4MOD4730
C		4MOD4740
C	INTERPOLATION ACCEPTABILITY TEST	4MOD4750
C		4MOD4760
	T1=2.*P	4MOD4770
	T2=3.*YM*Q-ABS(TOL1*Q)	4MOD4780
	T3=ABS(.5*E*Q)	4MOD4790
	IF (T1.GE.T2) GO TO 70	4MOD4800
	IF (P.GE.T3) GO TO 70	4MOD4810
	E=D	4MOD4820
	D=P/Q	4MOD4830
	GO TO 80	4MOD4840
C		4MOD4850
C	BISECTION	4MOD4860
C		4MOD4870
70	D=YM	4MOD4880
	E=D	4MOD4890
C		4MOD4900
C	COMPLETE STEP	4MOD4910
C		4MOD4920
80	A=B	4MOD4930
	PA=PB	4MOD4940
	IF (ABS(D).GT.TOL1) B=B+D	4MOD4950
	IF (ABS(D).LE.TOL1) B=B+SIGN(TOL1, YM)	4MOD4960
	PB=P(B, CNST)	4MOD4970
	T1=PB*(PC/ABS(PC))	4MOD4980
	IF (T1.GT.0.0) GO TO 20	4MOD4990
	GO TO 30	4MOD5000
C		4MOD5010

```

C          ROUTINE COMPLETED                                4MOD5020
C          90 RTFNDNR=B                                     4MOD5030
C          RETURN                                           4MOD5040
C          END                                              4MOD5050
C          4MOD5060
C          4MOD5070
C          4MOD5080
C          4MOD5090
C          FUNCTION SHRLD (IRED,DP,DG)
C          CODED BY:  LT J.L. PAQUETTE, USN                JAN 1982
C          NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
C          SUBPROGRAM TO COMPUTE THE LOAD SHARING RATIO, MN=PN/(.95*Z)
C          IRED  REDUCTION STAGE UNDER CONSIDERATION
C          DP    DIAMETER OF THE PINION
C          DG    DIAMETER OF THE GEAR
C          COMMON /DES DAT/ PWRIN(2),RPHIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),4MOD5210
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFP,IARR,IEPIC(3),IHARD(3)4MOD5220
2,2),IOPRO,NPWRIN,IPWRSR(2),NRKD,NPATH,NPLNT(3),NHELX
PI=4.*ATAN(1.)
PN=(PI/PND(IRED))*COS(PHIN(IRED))
DGB=DG*COS(PHI(IRED))
DPB=DP*COS(PHI(IRED))
DGO=DG*(2./PD(IRED))
DPO=DP*(2./PD(IRED))
Z=.5*(SQRT(DGO*DGO-DGB*DGB)+SQRT(DPO*DPO-DPB*DPB))-SQRT(DG*DG-DGB*DGB)4MOD5300
1GB)-SQRT(DP*DP-DPB*DPB))
SHRLD=PN/(.95*Z)
C          RETURN
C          END
C          4MOD5330
C          4MOD5340
C          4MOD5350
C          4MOD5360
C          4MOD5370

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C      FUNCTION THICK (D,DB,DT,T,III)
C
C      CODED BY:  LT J.L. PAQUETTE, USN          JAN 1982
C                NAVAL POSTGRADUATE SCHOOL    MONTEREY, CA 93940
C
C      SUBPROGRAM TO COMPUTE THE THICKNESS OF A TOOTH BASED ON
C      A THICKNESS KNOWN AT A SPECIFIED DIAMETER
C
C      D      REFERENCE DIAMETER OF KNOWN THICKNESS (USUALLY PITCH DIAM.)
C      DB     BASE DIAMETER OF THE GEAR
C      DT     DIAMETER AT WHICH THE THICKNESS IS TO BE COMPUTED
C      T      KNOWN THICKNESS AT REFERENCE DIAMETER
C      III    0 FOR EXTERNAL GEARS; 1 FOR INTERNAL GEARS
C
C      EXTERNAL SUBPROGRAM(S) REQUIRED: FUNCTION ARCCOS
C
C      REAL INV
C      INV(X)=TAN(X)-X
C      PHI=ARCCOS(DB,D)
C      PHIT=ARCCOS(DB,DT)
C      IF (III.EQ.1) GO TO 10
C      THICK=DT*((T/D)+INV(PHI)-INV(PHIT))
C      RETURN
C      THICK=DT*((T/D)-INV(PHI)+INV(PHIT))
C      RETURN
C      END
C
C      10

```

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4MOD5380
4MOD5390
4MOD5400
4MOD5410
4MOD5420
4MOD5430
4MOD5440
4MOD5450
4MOD5460
4MOD5470
4MOD5480
4MOD5490
4MOD5500
4MOD5510
4MOD5520
4MOD5530
4MOD5540
4MOD5550
4MOD5560
4MOD5570
4MOD5580
4MOD5590
4MOD5600
4MOD5610
4MOD5620
4MOD5630

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APPENDIX E

FIGURES

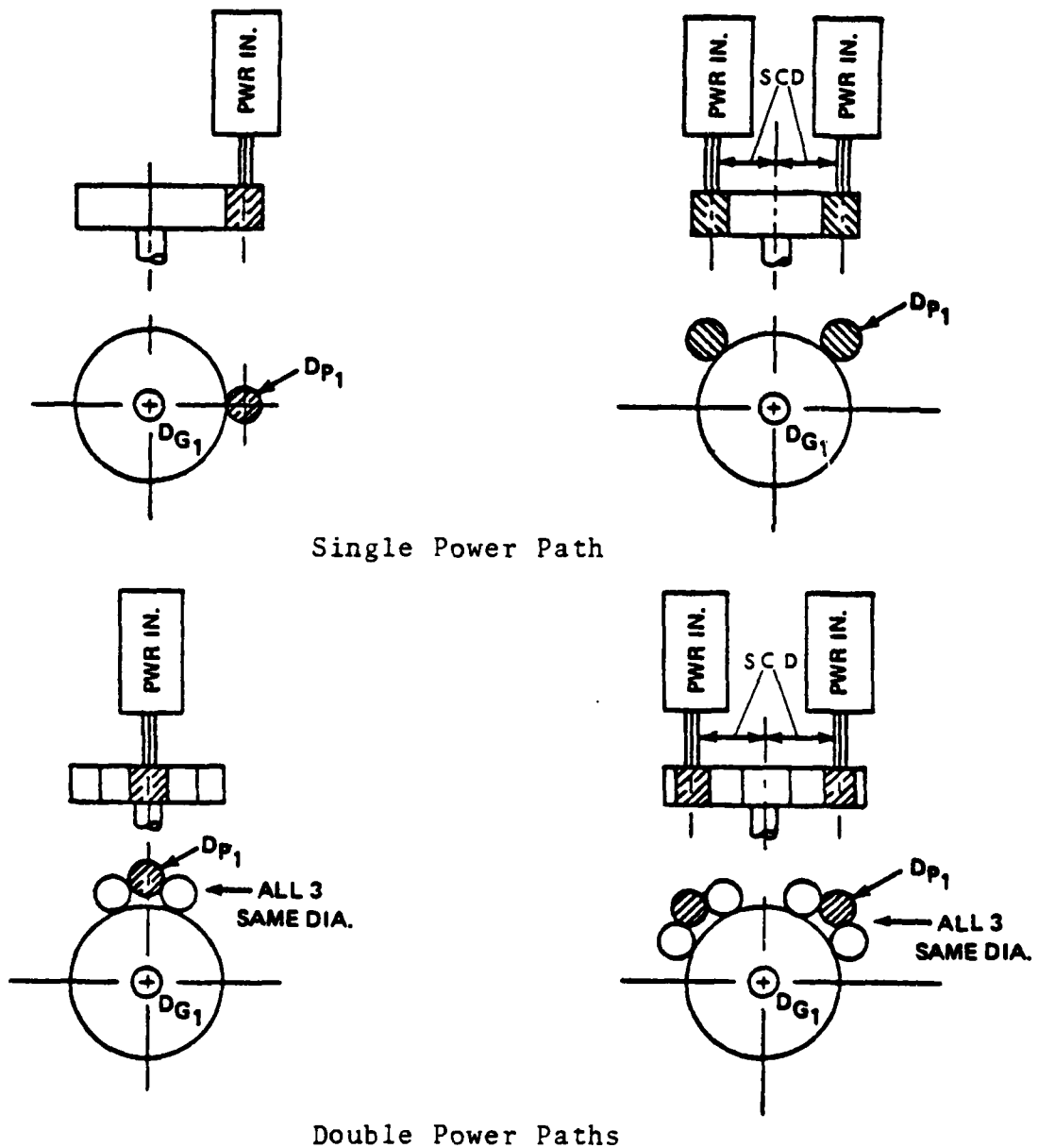
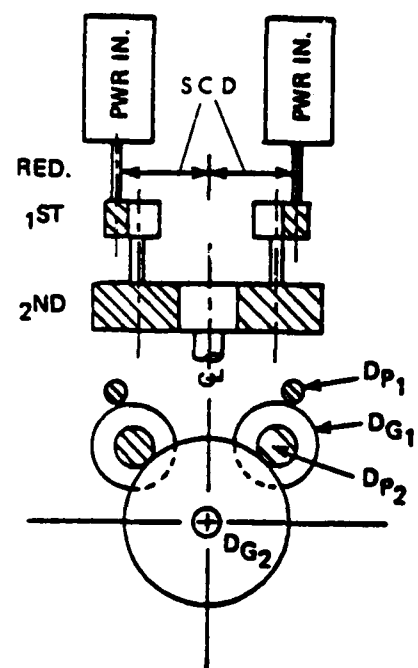
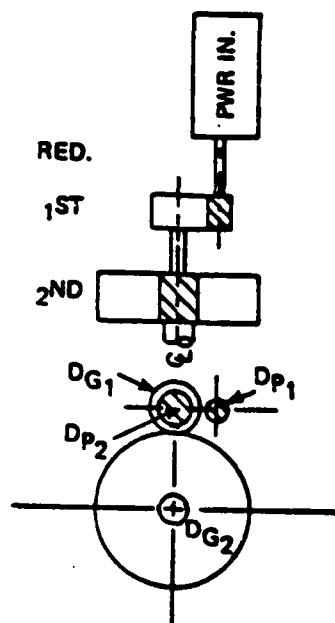
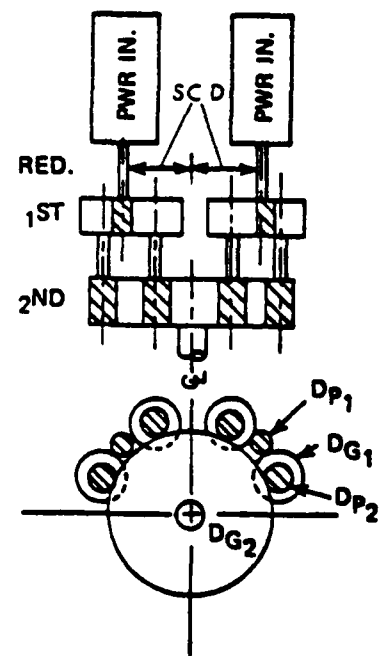
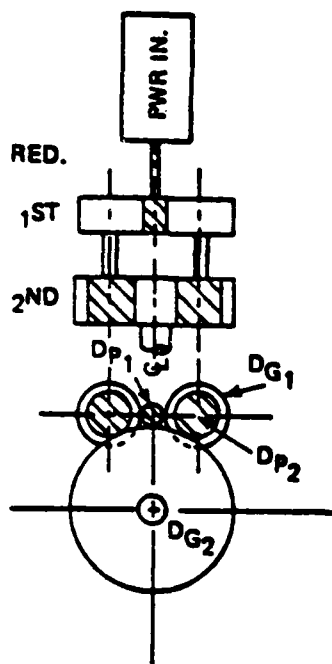


Figure 1: Single Reduction Parallel Axis Arrangements

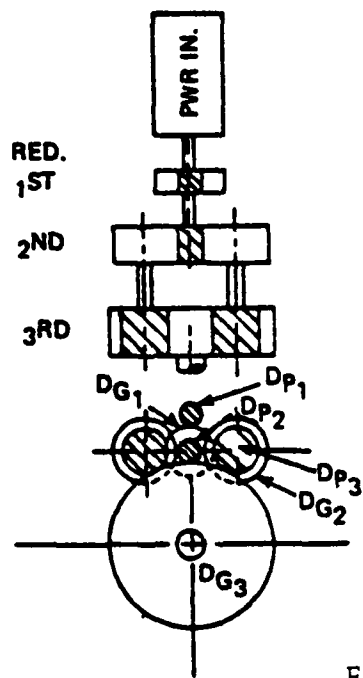


Single Power Path

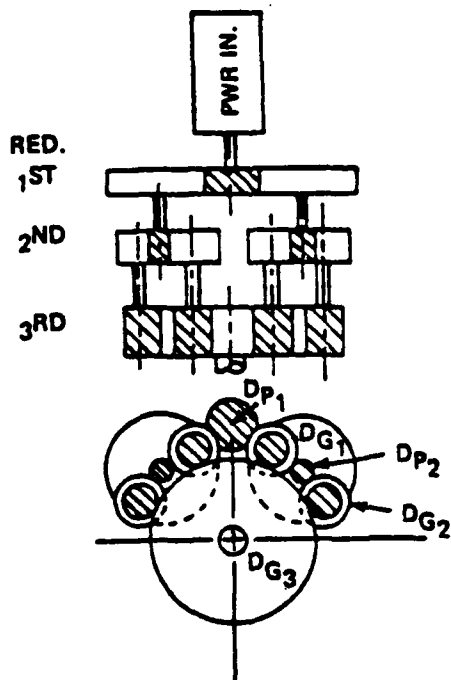
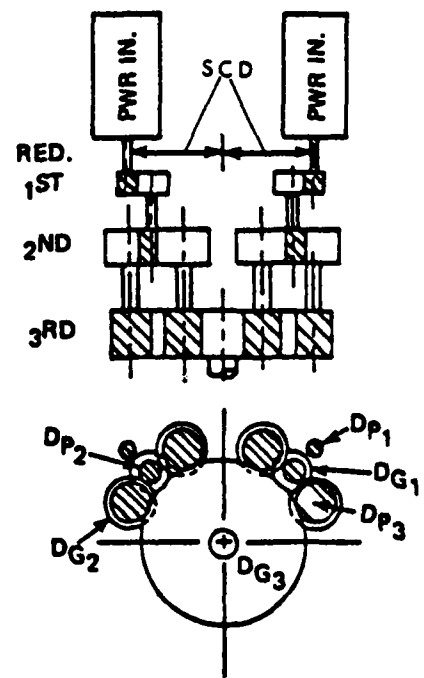


Double Power Path

Figure 2: Double Reduction Parallel Axis Arrangements



First Reduction
Single Power Path



Double Power Paths

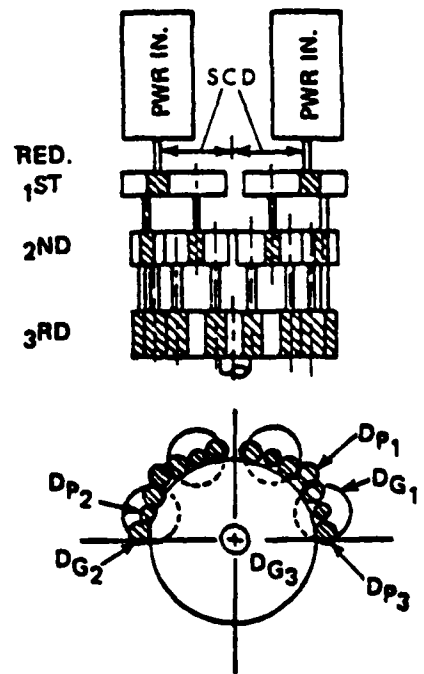
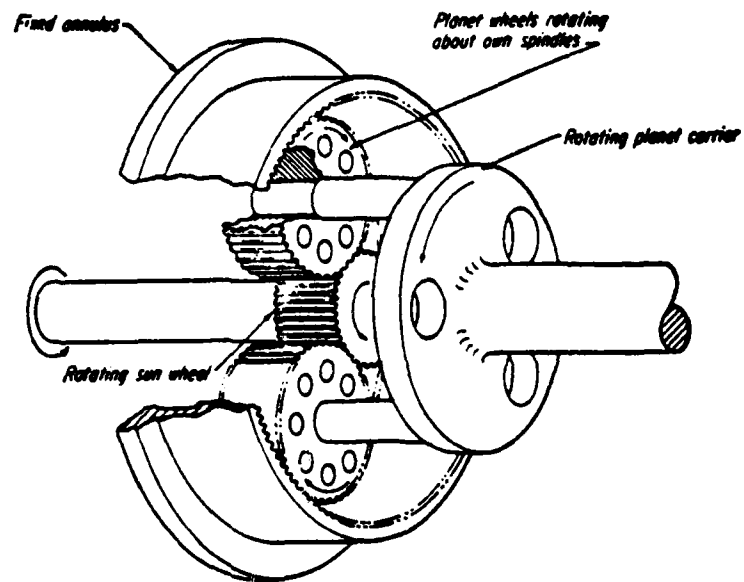
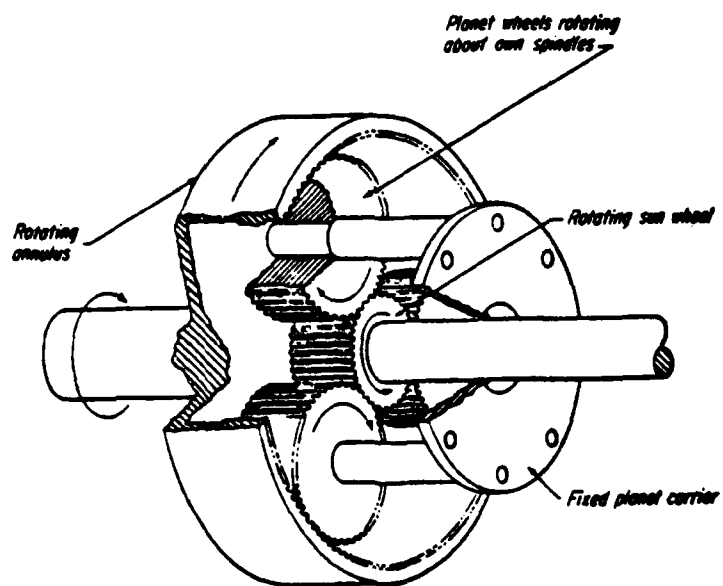


Figure 3: Triple Reduction Parallel Axis Arrangements



Planetary



Star

Figure 4: Single Reduction Epicyclic Arrangements
(from Ref. 7)

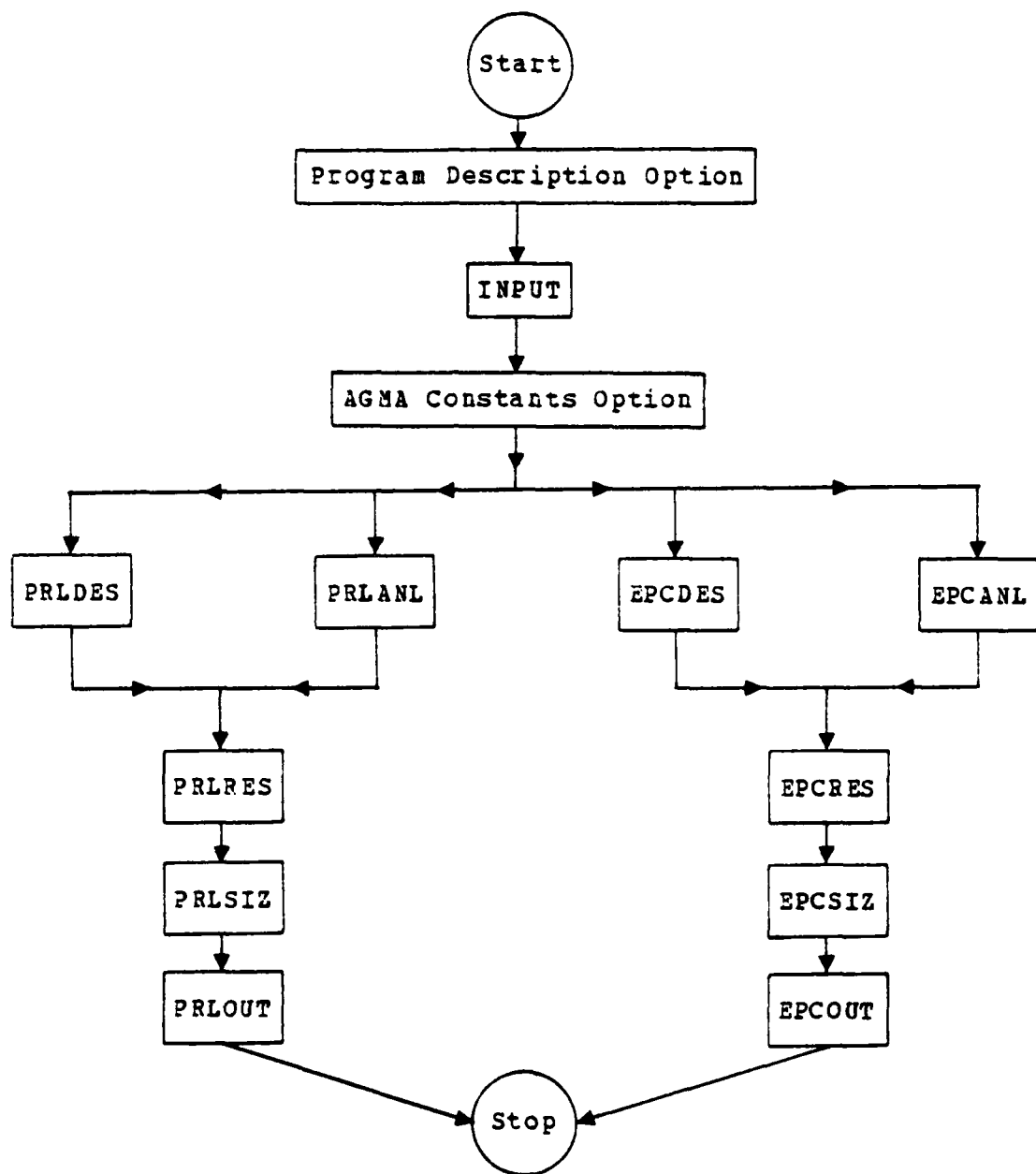


Figure 5: Flow Chart of the REGAD Package

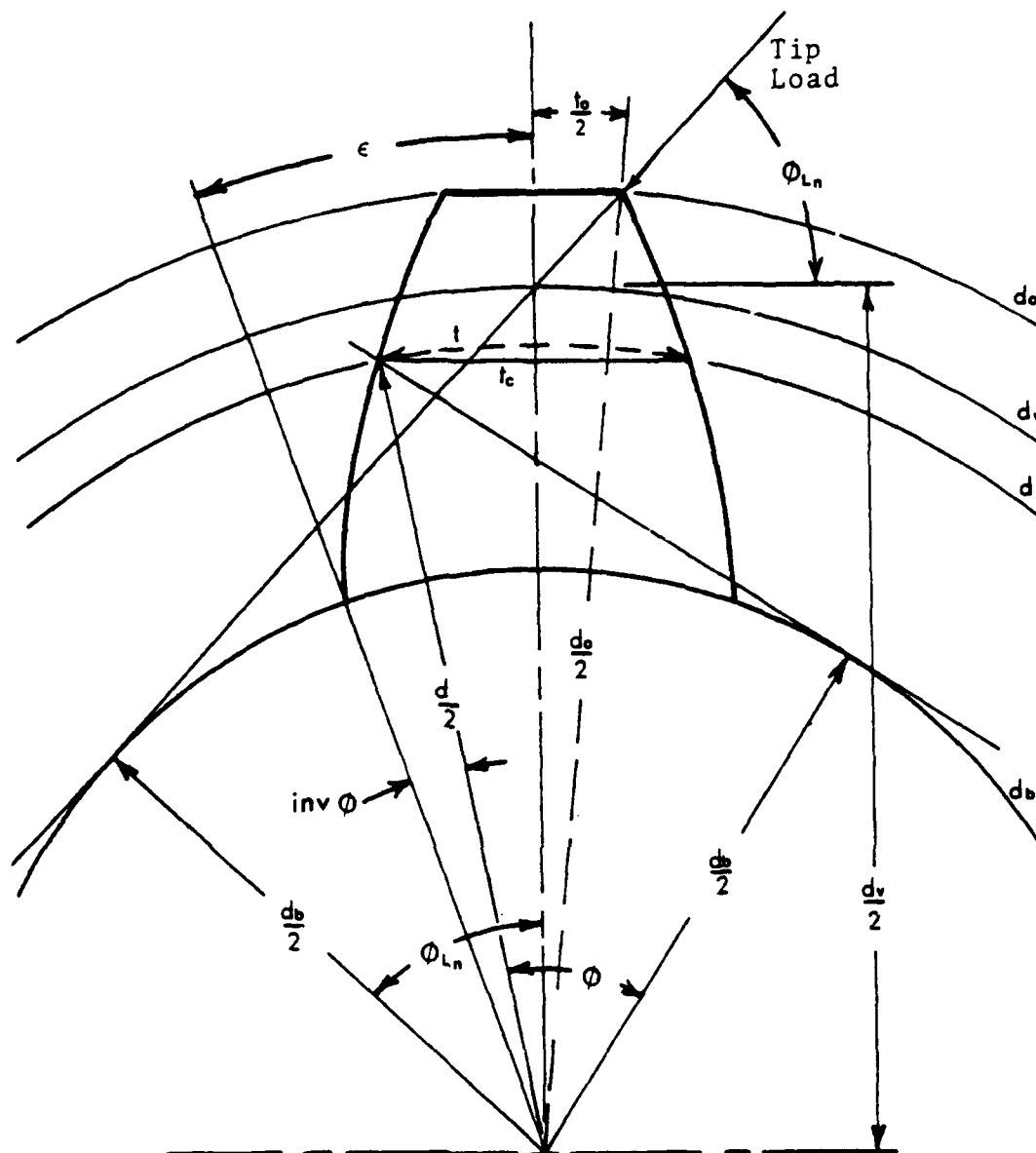


Figure 6: External Tooth Dimensions

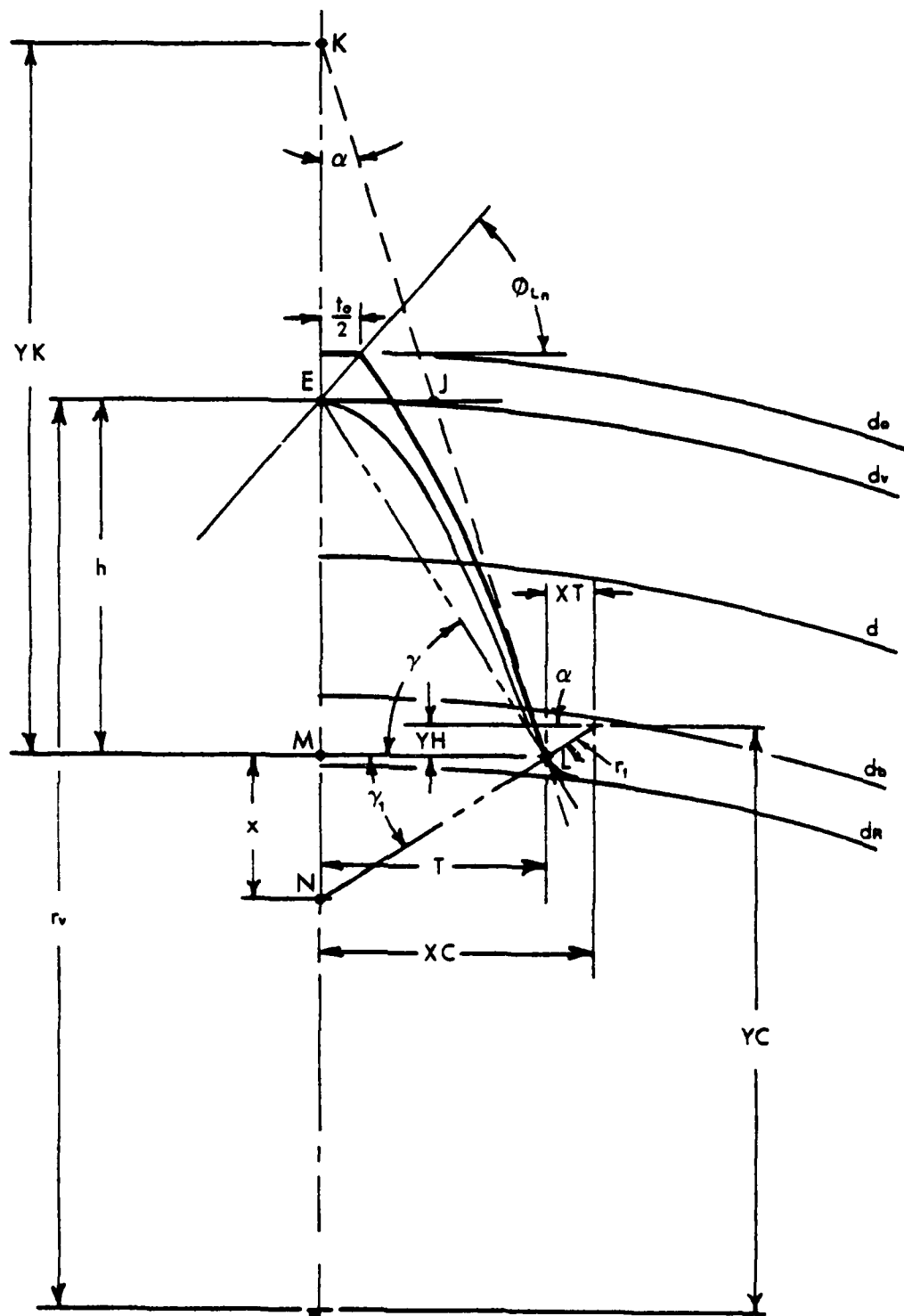


Figure 7: External Tooth Form Layout

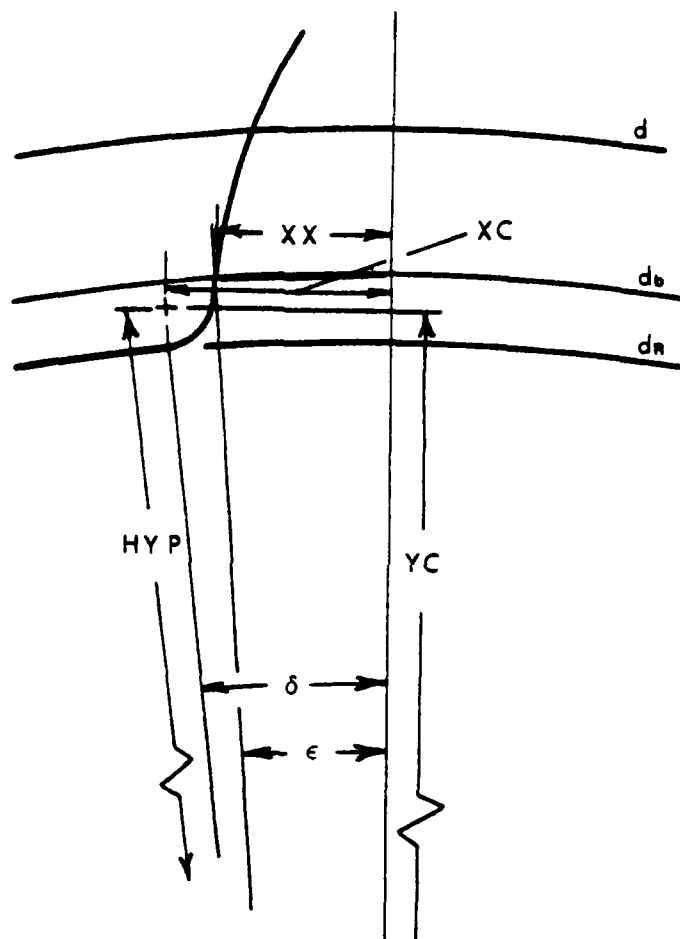


Figure 8: Fillet Center Location - Inside Base Circle

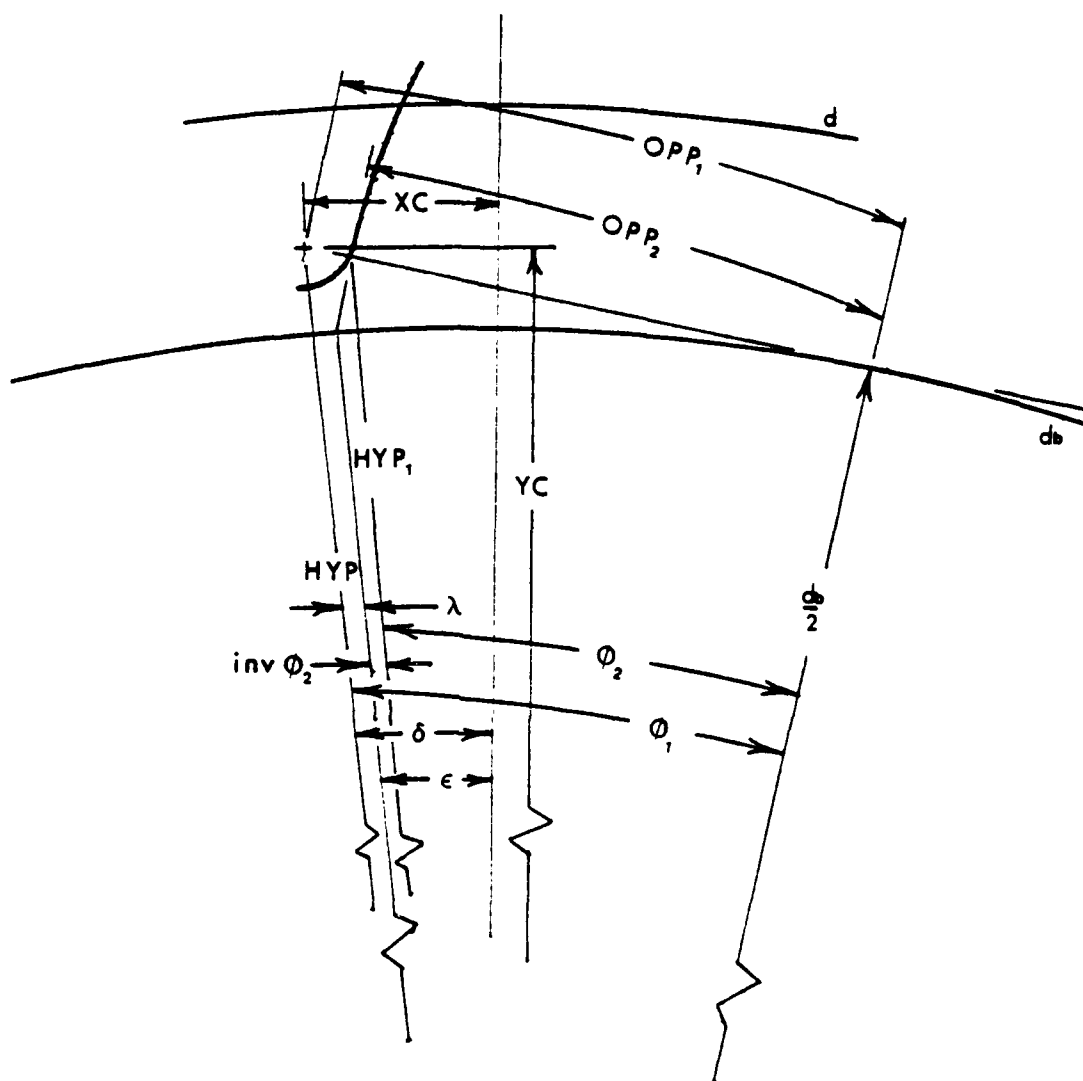


Figure 9: Fillet Center Location - Outside Base Circle

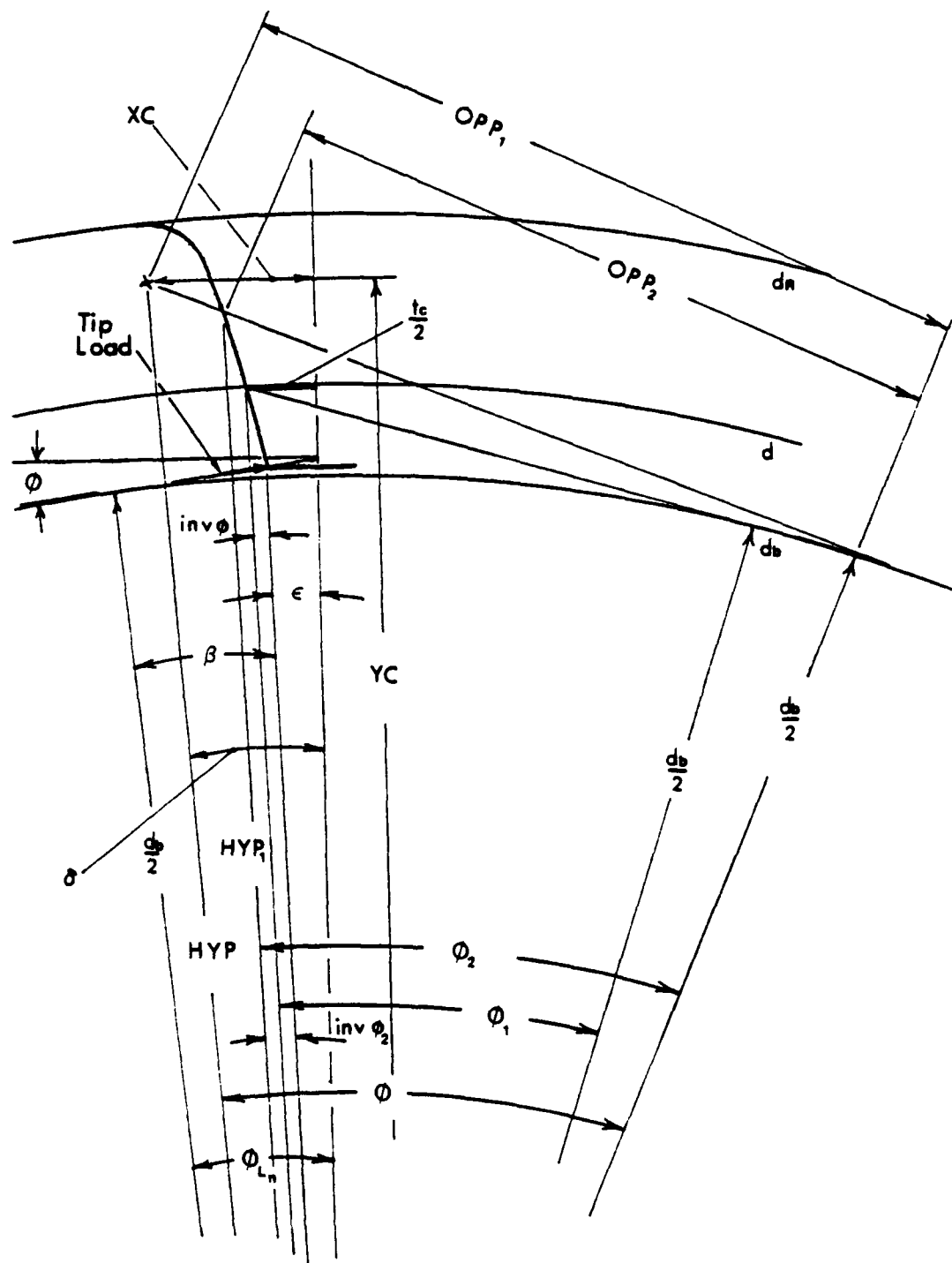


Figure 10: Internal Tooth Dimensions

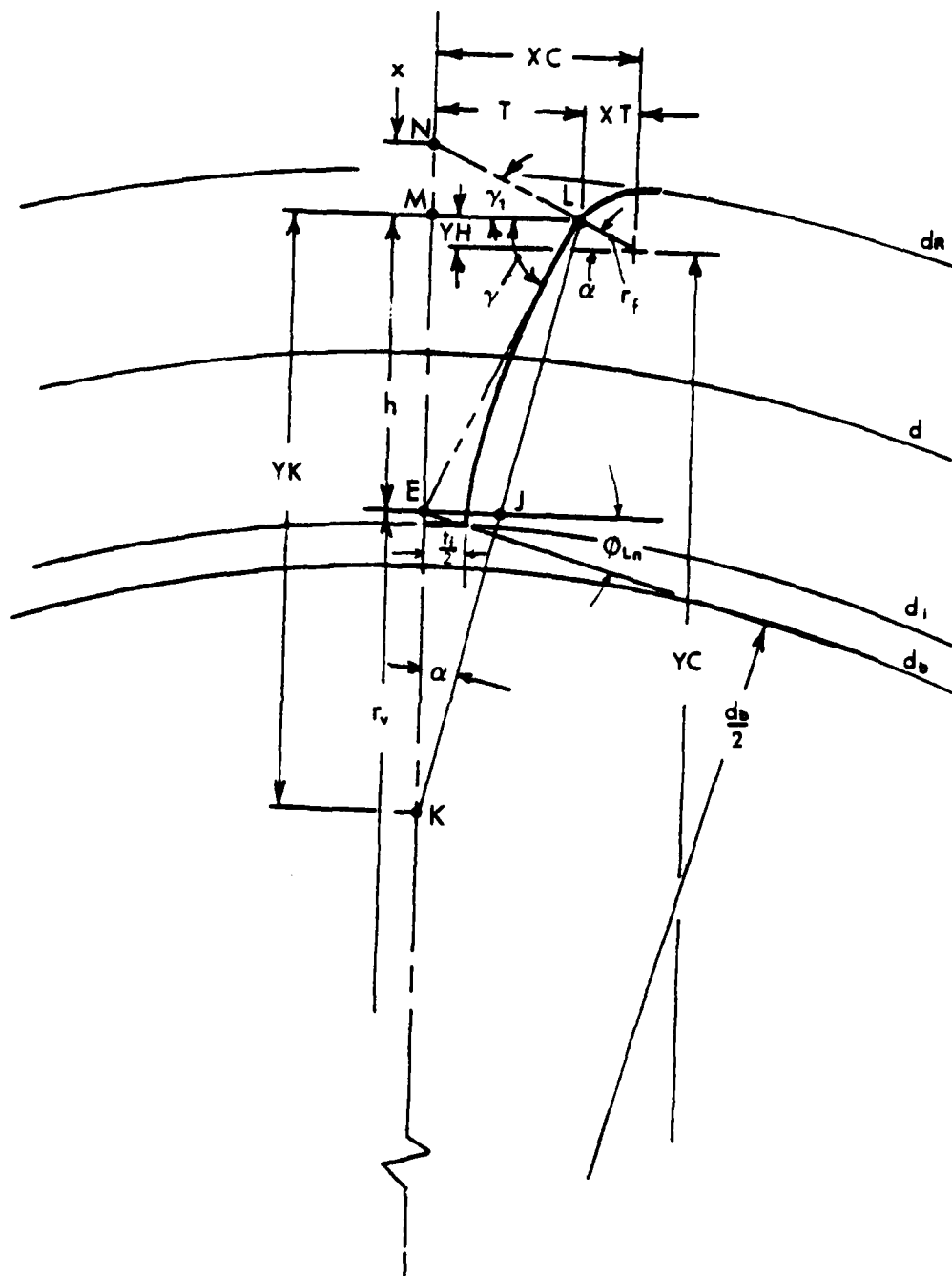


Figure 11: Internal Tooth Form Layout

LIST OF REFERENCES

1. American Gear Manufacturers Association, AGMA Information Sheet - Geometry Factors for Determining the Strength of Spur, Helical, Herringbone and Bevel Gear Teeth (AGMA 226.01), August, 1970.
2. American Gear Manufacturers Association, AGMA Standard for Surface Durability (Pitting) of Helical and Herringbone Gear Teeth (AGMA 211.02), September, 1966.
3. American Gear Manufacturers Association, AGMA Standard for Rating the Strength of Helical and Herringbone Gear Teeth (AGMA 221.02), July, 1965.
4. Thoma, F.A., "An Up-to-Date Look at Marine Gear Tooth Loading," Marine Technology, v. 7, p. 133-148, April 1970.
5. Thoma, F.A., Let's Up-Date Marine Gear Tooth Bending Stress Calculations, paper presented at the Mechanisms Conference & International Symposium on Gearing and Transmissions, San Francisco, California, 8-12 October 1972.
6. Forsythe, G.E., Malcolm, M.A., and Moler, C.B., Computer Methods for Mathematical Computations, p. 161-166, Prentice-Hall, Inc., 1977.
7. Dudley, D.W., ed., Gear Handbook, McGraw-Hill Book Company, Inc., 1962.
8. Benford, R.L., "Numerical Rules for Designing Planetary Gears," Machine Design, v. 30, p. 129-135, 21 August 1958.
9. Allison Division Report EDR 5503, Advancement of Helical Gear Design Technology, by W.L. McIntire and T.A. Lyon, p. 155-192, July 1968.
10. Dudley, D.W., Practical Gear Design, McGraw-Hill Book Company, Inc., 1954.

11. Gugliuzza, T.A. and Hargett, W.H., "Gear Design and Laboratory Experience - Marine Gas Turbine Propulsion," Journal of Engineering for Power, v. 91, October 1969.
12. Harrington, R.L., ed., Marine Engineering, The Society of Naval Architects and Marine Engineers, 1971.
13. Shigley, J.E., Mechanical Engineering Design, 3rd ed., McGraw-Hill Book Company, Inc., 1977.

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